

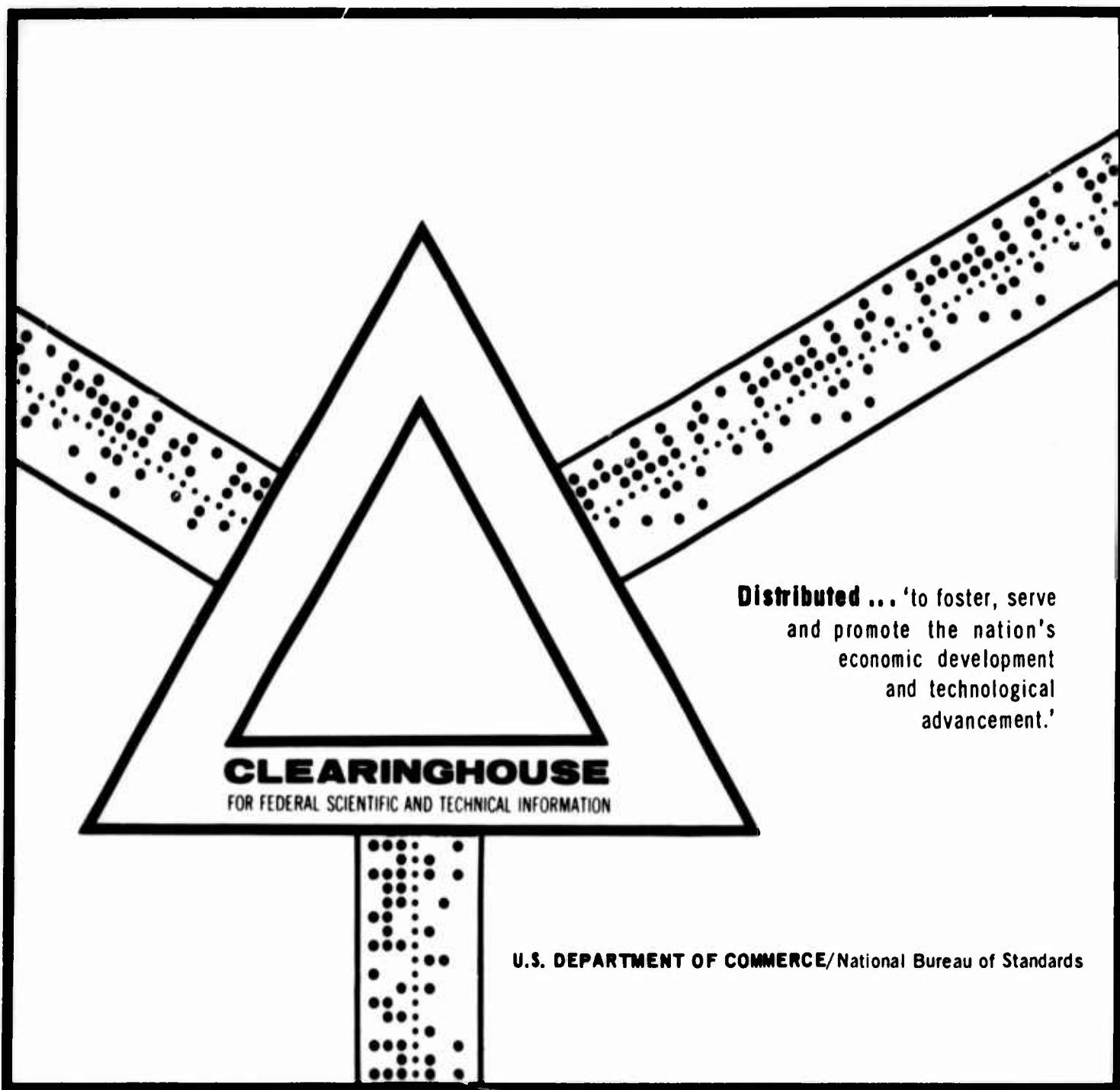
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ANALYSIS OF TRANSVERSELY ISOTROPIC LAMINATED CYLINDERS UNDER AXISYMMETRIC MECHANICAL AND THERMAL LOADS

Jonas A. Zukas

**Ballistic Research Laboratories
Aberdeen Proving Ground, Maryland**

November 1969



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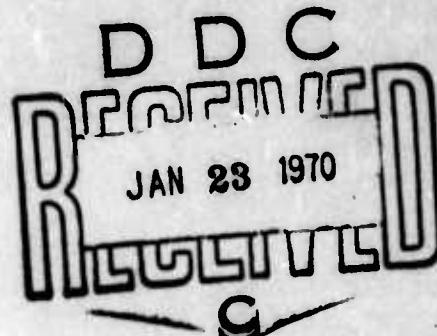
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ABERDEEN PROVING GROUND, MARYLAND**

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BALLISTIC RESEARCH LABORATORIES

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ABERDEEN PROVING GROUND, MARYLAND

BALLISTIC RESEARCH LABORATORIES

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ABSTRACT

A theory for the analysis of stresses in laminated circular cylindrical shells subjected to arbitrary axisymmetric mechanical and thermal loadings has been developed. This theory, specifically for use with pyrolytic graphite type materials, differs from the classical thin shell theory in that it includes the effects of transverse shear deformation and transverse isotropy, as well as thermal expansion through the shell thickness.

Solutions in several forms are developed for the governing equations. The form taken by the solution function is governed by geometric considerations. A range in which the various solution forms occur was determined numerically.

As a sample problem, the slow cooling of pyrolytic graphite deposited onto a commercial graphite mandrel was considered. Investigation of normal and shear stress behavior at the pyrolytic graphite - mandrel interface showed that these stresses decrease in magnitude with increasing E/G_c ratio and increasing deposit to mandrel thickness (h_a/h_b) ratio. This implies that a thin mandrel and a material weak in shear are desirable to minimize the possibilities of flaking and delamination of the pyrolytic graphite.

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NOTATION

- a, b subscripts indicating upper or lower lamina
 a_{ij}, b_{ij} constants defined by equations (C.2) and (C.3)
 A, B constants defined by (C.8)
 c preferred direction in a transversely isotropic material
 c_i roots of equation (22) defined by (C.11) and (C.13)
 C_i $\frac{E_i h_i}{1 - v_i^2}$ ($i = a, b$)
 D_i $\frac{E_i h_i^3}{12(1 - v_i^2)}$
 D d/dx
 e natural base
 E, E_c Young's Modulus in the plane of isotropy and "c" direction respectively
 G_c shear modulus relating stress and strain across the plane of isotropy ($= G_{xz} = G_{\theta z} = G_{zx} = G_{z\theta}$)
 g_i constants defined by equation (C.4)
 h_a, h_b individual lamina thicknesses
 h $h_a + h_b$
 i, j subscripts
 k_i constants defined by equation (C.6)
 m, n constants defined by equation (C.7)
 \bar{m} constant defined by (11a)
 M_x, M_θ stress couples

$M_{Tx}, M_{T\theta}$ thermal couples

\bar{M} defined by equation (41)

N_x, N_θ stress resultants

$N_{Tx}, N_{T\theta}$ thermal resultants defined by equation (41)

$p_{1i} \quad \sigma_z (h_i/2) \quad (i = a, b)$

$p_{2i} \quad \sigma_z (-h_i/2) \quad (i = a, b)$

$p(x) \quad p_{1a} - p_{2b}$

p_j joint normal stress ($= \sigma_z (-\frac{a}{2}) = \sigma_z (+\frac{b}{2})$)

Q_i shear resultant

\bar{Q} defined by equation (41)

R Radius to shell reference surface

T temperature measured from the stress-free temperature
of the material

u_x, u_z deflections in the "x" and "z" directions respectively

u_{0i} axial deflection of lamina middle surface

u_i^* virtual displacement of shell middle surface

w_i radial displacement of lamina middle surface

$\bar{w}_i = \int_0^z (\alpha_c T)_i dz \quad (i = a, b)$

w_i^* virtual radial displacement of lamina middle surface.

x axial coordinate for cylindrical shell

z cylindrical coordinate

α, α_c thermal expansion coefficients in the "x" and "z" directions
respectively

β_i	rotation of the normal to the undeformed lamina middle surface due to deformation
Γ	defined by equation (C.10)
$\lambda_1, \lambda_2, \lambda_3$	defined by equation (C.9)
δ_i^*	virtual rotation
Δ_i	defined by equation (21)
ϵ_{ij}	strain component
ν	Poisson's ratio in the $x - \theta$ plane ($\nu_{x\theta} = \nu_{\theta x}$)
ν_c	Poisson's ratio ($\nu_{xz} = \nu_{\theta z}$)
ν_{ij}	Poisson's ratio defined as the negative of the ratio of the strain in the j -direction to the strain in the i -direction due to a stress in the i -direction
σ_{ij}	stress component
τ_j	joint shear stress ($= \sigma_{zx} (-h_a/2) = \sigma_{zx} (+h_b/2)$)
θ	cylindrical coordinate in the circumferential direction
θ_1, θ_2	defined by equation (C.10)
$()'$	$\frac{d}{dx} ()$

I. INTRODUCTION

The ever-expanding missile and space technology continually demands materials capable of maintaining structural integrity at very high temperatures. Of late, attention has been focused on refractory materials, their anisotropy in physical and mechanical properties making them ideally suited for a wide range of insulation and/or structural applications.

Of the many refractory materials possible, pyrolytic graphite (PG) has probably received the most attention of late although it was known to Edison (1)* in 1883 who described methods for its manufacture, the technique involving formation of carbon deposits onto substrates heated in carbon-containing gases. For structural use, pyrolytic graphite is generally deposited at temperature from 3500°F to 4000°F in a stream of hydrocarbon gas, such as methane, onto a substrate of commercial graphite maintained at temperatures of 1500°F to 5000°F. The rate at which the material is produced depends on a number of factors which include the temperature, the reaction pressure, the hydrocarbon flow rate and

*Numbers in parenthesis indicate corresponding references in the bibliography

the surface to volume ratio of the substrate surface (2), (3), (4), (5). X-ray analysis of the resultant deposit shows a well-crystallized structure having much in common with the single graphite crystal (6). Growth is always normal to the substrate surface and after a thickness of 0.1" - 0.5" is reached, the deposition process is stopped and the deposit allowed to cool for several weeks.

The result of such a formation process is a material highly anisotropic in physical properties. The PG has one plane of isotropy parallel to the mandrel surface (x, θ direction -- see figure II-1 for geometry) and a single preferred direction (z), a state commonly referred to as transverse isotropy. With thermal conductivity between 100 and 1000 times greater in the (x, θ) direction than in the (z) direction, the material acts as an excellent conductor along its surface but also as a good insulator in the thickness (z) direction. The coefficient of thermal expansion in the (z) direction is from 10 to 30 times greater than that in the plane of isotropy (x, θ) so that thermal expansion through-the-thickness must be considered in many analyses of the material's thermal behavior.

Other curious effects due to the anisotropy are manifest in the Poisson's ratio which is negative in the plane of isotropy ($\nu_{x\theta} = \nu_{\theta x} = -0.21$) but large and positive in the preferred direction

($v_{xz} = v_{\theta z} = +0.9$). Furthermore, the ratio of the elastic modulus in the isotropic plant to the shear modulus in the transverse plane (E_x/G_{xz} or $E_\theta/G_{\theta z}$) may range from 20 to 50, compared to an E/G ratio of 2.5 for an isotropic material with $v = 0.25$. Therefore, in an analysis of a structure composed of such material, transverse shear deformation even for thin cross-sections must be considered.

Thermal and mechanical properties can be found readily (2), (5), (7), (8), (9), (10), (11), (12). Typical properties, given in Table I-1, are taken from (10), which are reasonably close to those given in other references, discrepancies most probably being due to variations in the deposition process.

Among the earliest analyses of structures of pyrolytic materials were those of Garber (13) and Levy (14) who treated thermal stresses in cylindrical and spherical shells and also the residual stresses caused by the general anisotropy of pyrolytic graphite, but neglected transverse shear deformation and did not account for the high thermal expansion coefficient in the (z) direction. McDonough (15) has considered thermal stresses in shells of revolution of pyrolytic graphite type materials subjected to axially symmetric loads, including transverse shear deformation and thermal expansion through the thickness. He was able to show that neglect of transverse shear deformation would lead to an over-estimate of the stiffness coefficient

TABLE I-1 PHYSICAL PROPERTIES OF PYROLYTIC GRAPHITE

MECHANICAL PROPERTIES

1. Young's Modulus (PSI)

<u>TEMP</u>	<u>(x,θ) Direction</u>	<u>(z) Direction</u>
70°F.	5.4×10^6	1.5×10^6
1000°F	4.3×10^6	1.29×10^6
2000°F	3.5×10^6	1.05×10^6
3000°F	2.7×10^6	0.81×10^6

2. Poisson's Ratio

$$70°F \quad v_{x\theta} = 0.21 \quad v_{xz} = 0.90$$

THERMAL PROPERTIES

1. Thermal Expansion in/in - °F

70°F	0.0	13.1×10^{-6}
1000°F	0.6×10^{-6}	13.1×10^{-6}
2000°F	1.2×10^{-6}	13.1×10^{-6}
3000°F	1.7×10^{-6}	13.1×10^{-6}

2. Conductivity, BTU/hr-ft- °F

70°F	290.0	1.25
1000°F	165.0	0.82
2000°F	100.0	0.60
3000°F	60.0	0.60

for properties representative of PG while excluding thermal expansion in the (z) direction leads not only to erroneous stress predictions but that even the sign of the stress (tension or compression) may be wrong. Klinger (16), (17) has extended McDonough's work for the case of conical shells in that he derives equations for non-axially symmetric mechanical and thermal loadings. Raju (18) studied the case of shallow shells of pyrolytic graphite type materials subjected to a variety of axially symmetric and non-axially symmetric loads. Daugherty (19), (20) treated the case of non-circular cylindrical shells of pyrolytic materials.

The preceding deal with single-layer shells. Anisotropic laminated shells of revolution with elastic properties symmetric about the middle surface of the composite shell are extensively treated by Ambartsumian (21), whereas Dong (22), (23), et al (24) treat layered shells wherein the structure is assumed to be composed of an arbitrary numbers of bonded layers each of different constant thickness, different orientation of elastic axes and different anisotropic elastic properties. Since Dong does not assume elastic symmetry about the middle surface, flexural and extensional deformations are coupled and solution techniques for homogeneous shells do not carry over directly for anisotropic elastic shells. Hence, alternate methods of solution are developed.

Radkowsky et al (25) considered laminated isotropic shells of revolution with variable thickness using E. Reissner's for-

formulation (26). Radkowsky extended this work to include variable laminated orthotropic material properties (27). Both formulations were restricted to axisymmetric loads. In Radkowsky's works and that of Sepetoski (28), the governing equations were cast in finite difference form and solved with the aid of a digital computer. The introduction to Dong's paper (23) makes interesting reading regarding the hazards of this perfectly valid technique.

Other treatments of laminated cylinders have been by Jones and Whittier (29), Tsai and Azzi (30), Paul (31), Au (32), Keeffe and Windholz (33). These, and most other references cited herein are characterized by neglect of transverse shear deformation. A recent work of great theoretical elegance, even though it neglects transverse shear deformation, is that of Zudans (34) which presents a theory for arbitrarily loaded (mechanically & thermally) shells of revolution with internal masses and ring stiffeners, derived under the Kirchoff Hypothesis and consistent with balance of energy as well as linear and angular momentum and invariance under transformation of middle surface coordinate systems and rigid body displacements. The elegance, unfortunately, does not carry over to the computational techniques (35).

Laminated isotropic plates have been considered by Vinson (36) who treated thermal stresses in circular plates, neglecting transverse shear deformation. Summers (37) treats thick and thin isotropic and orthotropic laminated plates including transverse shear deformation.

Mehta (34) considers orthotropic and isotropic laminated as well as single-layered rectangular plates of pyrolytic graphite type materials under static mechanical and thermal loads. Wu (39), (40), and (41) treats the lateral vibrations of both small and large amplitude for rectangular plates of pyrolytic and graphite materials.

Except for those references dealing with pyrolytic graphite type materials, all the works reviewed which analyze multi-layered structures are characterized by their neglect of either transverse shear deformation or thermal expansion through-the-thickness or both.

Prompted by the absence of a definitive treatment of laminated shells applicable to pyrolytic graphite type materials, this thesis was undertaken. It is an extension of McDonough's work in that layered cylindrical shells, including the effects of transverse shear deformation and thermal expansion through-the-thickness, subjected to arbitrary axi-symmetric loading are considered.

II. DERIVATION OF GOVERNING EQUATIONS

The coordinate system used is shown in Figure (II-1). The three-dimensional equations of thermoelasticity (uncoupled) for the case of axial symmetry are given by:

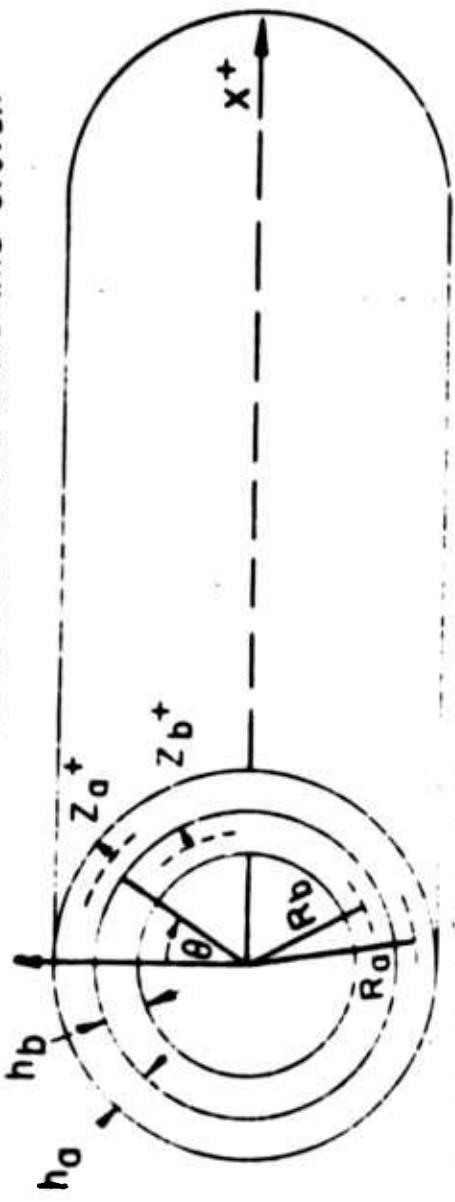
Stress-Strain Relations (transversely isotropic material)

$$\begin{aligned}
 \epsilon_x &= \frac{1}{E} (\sigma_x - v\sigma_\theta - \frac{v}{c}\sigma_z) + \alpha T \\
 \epsilon_\theta &= \frac{1}{E} (\sigma_\theta - v\sigma_x - v\frac{c}{c}\sigma_z) + \alpha T \\
 \epsilon_z &= \frac{\sigma_z}{E_c} - \frac{v}{E} (\sigma_x + \sigma_\theta) + \alpha_c T \\
 \epsilon_{xz} &= \frac{\sigma_{xz}}{2G_c}
 \end{aligned} \tag{1}$$

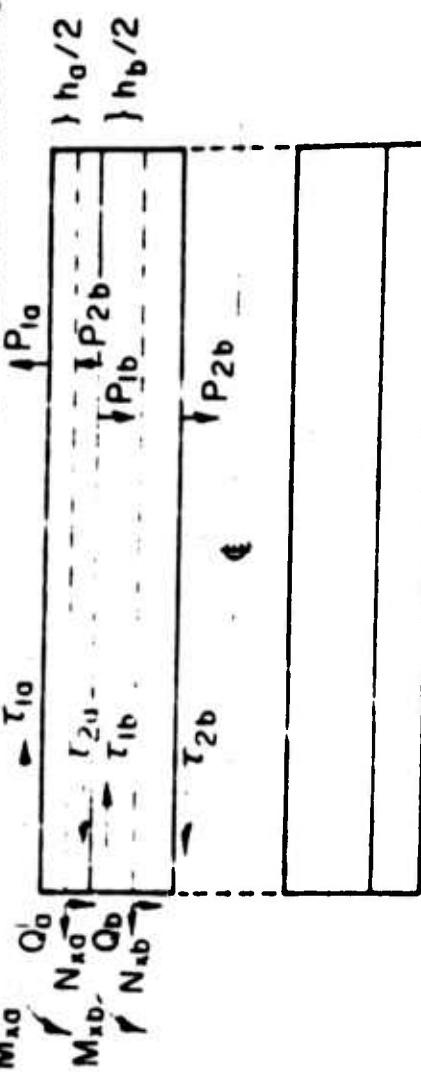
where $\epsilon_{x\theta} = \epsilon_{\theta z} = \sigma_{x\theta} = \sigma_{\theta z} = 0$ by symmetry

ϵ_{ij} and σ_{ij} are the physical components of the strain and stress tensors respectively, and for brevity $\sigma_{ij} = \sigma_i$ when $i = j$. E , E_c , v , v_c , G_c are five independent elastic constants where again for brevity $v = v_{x\theta} = v_{\theta x}$ and $v_c = v_{xz} = v_{\theta z}$. α , α_c are the coefficients of thermal expansion in the x and z directions of the materials. T is the temperature measured from the stress-free temperature of the material in units consistent with the α 's.

FIGURE II-1: LAYERED CYLINDRICAL SHELL COORDINATE SYSTEM



1A. CYLINDER GEOMETRY INDICATING POSITIVE DIRECTIONS



$$P(x) = P_{1a} - P_{2b}; \quad T_j = T_{2a} = T_{1b}; \quad P_j = P_{2a} = P_{1b}$$

1B. FORCES ON LAYERED CYLINDER

The preferred direction for the material is everywhere coincident with the z coordinate. This restriction is carried throughout this work.

Equilibrium Equations

The equilibrium equations when applied to the present problem become for each lamina:

$$R(1 + \frac{z}{R}) \frac{\partial \sigma_x}{\partial x} + R(1 + \frac{z}{R}) \frac{\partial \sigma_{xz}}{\partial z} + \sigma_{xz} = 0 \quad (2)$$

$$R(1 + \frac{z}{R}) \frac{\partial \sigma_z}{\partial z} + R(1 + \frac{z}{R}) \frac{\partial \sigma_{xz}}{\partial x} + \sigma_z - \sigma_\theta = 0$$

The third equation is identically zero from symmetry considerations.

Strain-Deformation Equations

The strain-deformation relations can be written correspondingly as:

$$\begin{aligned} \epsilon_x &= \frac{\partial u_x}{\partial x} \\ \epsilon_\theta &= \frac{1}{R(1+\frac{z}{R})} u_z \\ \epsilon_z &= \frac{\partial u_z}{\partial z} \\ \epsilon_{xz} &= \frac{1}{2} \left\{ \frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right\} \end{aligned} \quad (3)$$

$u_\theta \equiv 0$ by symmetry

The displacements are positive in the direction of the positive corresponding coordinate (See Figure II-1).

Assumptions:

1. The thickness of the shell is small compared with other dimensions, hence Love's First Approximation is applicable:

$$\text{i.e. } \frac{h}{R_{\min}} \ll 1 \quad (4)$$

2. The displacements are small compared to the thickness of the shell and the angles of rotation are small compared to unity.

3. The transverse normal stress is small compared with other normal stress components and is neglected in the stress-strain equations.

4. A linear element normal to the undeformed middle surface undergoes translation and rotation and remains straight, implying deformations of the form

$$u_x = u_0(x) + z\beta(x) \quad (5)$$

5. Transverse normal strain due to thermal expansion will be included; that due to mechanical (or isothermal) loads will be neglected.

This implies a deformation of the form

$$u_z(x, z) = w(x) + \bar{w}(x, z) \quad (6)$$

where

$$\bar{w}(x, z) = \int_0^z \alpha_c T dz$$

6. Material properties are constant for each lamina.

With these assumptions, the equation (1) becomes:

$$\begin{aligned}\epsilon_x &= \frac{1}{E} (\sigma_x - v\sigma_\theta) + \alpha T \\ \epsilon_\theta &= \frac{1}{E} (\sigma_\theta - v\sigma_x) + \alpha T, \\ \epsilon_z &= \alpha_c T \\ \epsilon_{xz} &= \frac{\sigma_{xz}}{2G_c}\end{aligned}\tag{7}$$

Making use of equations (5) and (6) and neglecting z/R in comparison to unity, the equations (3) become:

$$\begin{aligned}\epsilon_x &= u_0'' + z\beta'' \\ \epsilon_\theta &= \frac{1}{R} (w + \bar{w}) \\ \epsilon_z &= \frac{\partial \bar{w}}{\partial z} \\ \epsilon_{xz} &= 1/2(\beta'' + w'')\end{aligned}\tag{8}$$

where $()'' = D \cdot () = \frac{d}{dx} ()$

Integrated Equations

The stress resultants and couples are defined as follows:

$$N_{x_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} \sigma_x dz$$

$$N_{Tx_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} E_i \alpha_i T dz$$

$$N_{\theta_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} \sigma_\theta dz$$

$$N_{T\theta_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} E_i \alpha_i T dz$$

$$Q_i = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} \sigma_{xz} dz$$

(9)

$$M_{x_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} z \sigma_x dz$$

$$M_{Tx_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} z E_i \alpha_i T dz$$

$$M_{\theta_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} z \sigma_\theta dz$$

$$M_{T\theta_i} = \int_{\frac{-h_i}{2}}^{\frac{h_i}{2}} z E_i \alpha_i T dz$$

$i = a, b$

The equilibrium equations in terms of stress resultants and couples are readily obtained. Integrating (2) directly yields

$$N'_{x_i} + \tau_{1i} - \tau_{2i} = 0$$

$$M'_{x_i} - Q_i + \frac{h_i}{2} \tau_{1i} + \frac{h_i}{2} \tau_{2i} = 0$$
(10)

$$Q'_i - \frac{N_{\theta i}}{R_i} + P_{1i} - P_{2i} = 0$$

$$i = a, b$$

Solving equations (7) for normal stresses, first integrating them from $-h_i/2$ to $+h_i/2$; then multiplying through the equations by z and integrating once again between the same limits, making use of the definitions of the stress resultants and couples (9) and simplifying, the following stress-strain relations result:

$$\begin{aligned}
 N_{x_i} &= \frac{E_i h_i}{(1-v_i^2)} \left(u'_{0i} + \frac{v_i w_i}{R_i} \right) - \frac{N_{Tx_i}}{1-v_i} + \frac{E_i v_i}{R_i (1-v_i^2)} \int_{-\frac{h_i}{2}}^{\frac{h_i}{2}} \bar{w}_i dz \\
 N_{\theta_i} &= \frac{E_i h_i}{(1-v_i^2)} \left(v_i u'_{0i} + \frac{w_i}{R_i} \right) - \frac{N_{T\theta_i}}{1-v_i} + \frac{E_i}{R_i (1-v_i^2)} \int_{-\frac{h_i}{2}}^{\frac{h_i}{2}} \bar{w}_i dz \\
 M_{x_i} &= \frac{E_i h_i^3}{12(1-v_i^2)} \beta'_i - \frac{M_{Tx_i}}{1-v_i} + \frac{E_i v_i}{R_i (1-v_i^2)} \int_{-\frac{h_i}{2}}^{\frac{h_i}{2}} z \bar{w}_i dz \\
 M_{\theta_i} &= \frac{E_i h_i^3}{12(1-v_i^2)} \beta'_i - \frac{M_{T\theta_i}}{1-v_i} + \frac{E_i}{R_i (1-v_i^2)} \int_{-\frac{h_i}{2}}^{\frac{h_i}{2}} z \bar{w}_i dz
 \end{aligned} \tag{11}$$

$i = a, b$

An integrated shear stress strain equation is required in addition to (11). The necessary expression is obtained using weighted integration, the procedure being analogous to that in reference 15, and for convenience given in Appendix D.

$$Q_i = \frac{\bar{m}_i}{6} + \frac{5}{6} h_i G_c^i (\beta_i + w'_i) + \Delta_i$$

$$\text{where } \bar{m}_i = \frac{h_i}{2} (\tau_{1i} + \tau_{2i}) \quad (11A)$$

Solving for Q_a and Q_b from the second of the integrated equilibrium equations (10) and substituting into the third, making use of M_{x_i} and N_{θ_i} from (11) and the definitions

$$C_i = \frac{E_i h_i}{(1-v_i^2)} ; \quad D_i = \frac{E_i h_i^3}{12(1-v_i^2)} \quad (i = a, b)$$

$$\tau_j - \sigma_{zx} \left(-\frac{h_a}{2} \right) = \sigma_{zx} \left(+\frac{h_b}{2} \right)$$

$$p_j = \sigma_z \left(-\frac{h_a}{2} \right) = \sigma_z \left(+\frac{h_b}{2} \right)$$

two useful relations are obtained:

$$D_a D^3 \beta_a - \frac{C_a v_a h_a}{2R_a} D \beta_a - \frac{C_a v_a h_b}{2R_a} D \beta_b + \frac{h_a}{2} D \tau_j - p_j - \frac{C_a v_a}{R_a} D u_{\theta_b} - \frac{C_a}{2R_a} w_b \\ = a_{17} + \pi_1 \quad (12)$$

$$D_b D^3 \beta_b + \frac{h_b}{2} D\tau_j + p_j - \frac{C_b v_b}{R_b} Du_{0b} - \frac{C_b}{R_b^2} w_b = a_{27} + \pi_2 \quad (13)$$

Note that in the above, use has been made of the conditions that the laminae are bonded together and that no slippage occurs in the joints between laminae. The former is given by

$$\begin{aligned} u_z\left(\frac{-h_a}{2}\right) &= u_z\left(\frac{h_b}{2}\right) \text{ which implies that } \bar{w}_a(x, \frac{-h_a}{2}) + w_a = \bar{w}_b(x, \frac{h_b}{2}) + w_b \\ \text{or } w_a &= w_b + \bar{w}_b(x, \frac{h_b}{2}) - \bar{w}_a(x, \frac{-h_a}{2}) \\ &= w_b + \hat{E} \end{aligned} \quad (14)$$

where $\hat{E} = \bar{w}_b(x, \frac{h_b}{2}) - \bar{w}_a(x, \frac{-h_a}{2})$.

The latter condition is expressed by

$$\begin{aligned} u_x\left(\frac{-h_a}{2}\right) &= u_x\left(\frac{h_b}{2}\right) \\ u_{0a} - \frac{h_a}{2} \beta_a &= u_{0b} + \frac{h_b}{2} \beta_b \\ u_{0a} &= u_{0b} + \left(\frac{h_a}{2} \beta_a + \frac{h_b}{2} \beta_b\right) \end{aligned} \quad (15)$$

Making use of (11), (14) and (15) in the first integrated equilibrium equation, two further relations are obtained:

$$\frac{C_a h_a}{2} D^2 \beta_a + \frac{C_a h_b}{2} D^2 \beta_b - \tau_j + C_a D^2 u_{0b} + \frac{C_a v_a}{R_a} D w_b = \alpha_{37} + \pi_3 \quad (16)$$

$$\tau_j + C_b D^2 u_{0b} + \frac{C_b v_b}{R_b} D w_b = \alpha_{47} + \pi_4 \quad (17)$$

Using (11A) and the second of the integrated equilibrium equations together with (14) and (15), two final relations are obtained:

$$D_a D^2 \beta_a - \frac{5}{6} G_c^a h_a \beta_a + \frac{5}{12} h_a \tau_j - \frac{5}{6} G_c^a h_a D w_b = \alpha_{57} + \pi_5 \quad (18)$$

$$D_b D^2 \beta_b - \frac{5}{6} G_c^b h_b \beta_b + \frac{5}{12} h_b \tau_j - \frac{5}{6} G_c^b h_b D w_b = \alpha_{67} + \pi_6 \quad (19)$$

where in the above

$$\alpha_{17} = \frac{N_T \theta_a}{R_a (1-v_a)} - \frac{C_a \hat{E}}{R_a^2} - \frac{E_a}{R_a^2 (1-v_a^2)} \int_{-\frac{h_a}{2}}^{\frac{h_a}{2}} \bar{w}_a dz$$

$$+ D^2 \left\{ \frac{M_{Tx}}{1-v_a} - \frac{E_a v_a}{R_a (1-v_a^2)} \right\} \int_{-\frac{h_a}{2}}^{\frac{h_a}{2}} z \bar{w}_a dz$$

$$a_{27} = \frac{N_{T\theta_b}}{R_b(1-v_b)} - \frac{E_b}{R_b^2(1-v_b^2)} \int_{-\frac{h_b}{2}}^{\frac{h_b}{2}} \bar{w}_b dz$$

$$+ D^2 \left\{ \frac{M_{Tx_b}}{1-v_b} - \frac{E_b v_b}{R_b(1-v_b^2)} \int_{-\frac{h_b}{2}}^{\frac{h_b}{2}} z \bar{w}_b dz \right\}$$

$$a_{37} = D \left\{ \frac{N_{Tx_a}}{1-v_a} - \frac{C_a v_a}{R_a} \hat{E} - \frac{E_a v_a}{R_a(1-v_a^2)} \int_{-\frac{h_a}{2}}^{\frac{h_a}{2}} \bar{w}_a dz \right\}$$

$$a_{47} = D \left\{ \frac{N_{Tx_b}}{1-v_b} - \frac{E_b v_b}{R_b(1-v_b^2)} \int_{-\frac{h_b}{2}}^{\frac{h_b}{2}} \bar{w}_b dz \right\}$$

$$a_{57} = \Delta_a + D \left\{ \hat{E} + \frac{M_{Tx_a}}{1-v_a} - \frac{E_a v_a}{R_a(1-v_a^2)} \int_{-\frac{h_a}{2}}^{\frac{h_a}{2}} z \bar{w}_a dz \right\}$$

$$a_{67} = \Delta_b + D \left\{ \frac{M_{Tx_b}}{1-v_b} - \frac{E_b v_b}{R_b(1-v_b^2)} \int_{-\frac{h_b}{2}}^{\frac{h_b}{2}} z \bar{w}_b dz \right\}$$

(20)

with

$$\pi_1 = -p_{1a} - \frac{h_a}{2} D\tau_{1a}$$

$$\pi_2 = +p_{2b} - \frac{h_b}{2} D\tau_{2b}$$

$$\pi_3 = -\tau_{1a}$$

$$\pi_4 = +\tau_{2b}$$

$$\pi_5 = -\frac{5}{12} h_a \tau_{1a}$$

$$\pi_6 = -\frac{5}{12} h_b \tau_{2b}$$

(21)

$$\Delta_i = \frac{5}{4} \int_{-\frac{h_i}{2}}^{\frac{h_i}{2}} \left\{ G_c^i \bar{w}'_i - \left(\frac{2z}{h_i} \right)^2 \left[G_c^i \bar{w}'_i - \frac{1}{R} \int_{-\frac{h_i}{2}}^z \Omega dy \right] \right\} dz \quad (i = a, b)$$

The governing equations may be transformed to a more useful form for solution. Since they are cumbersome for manipulation in their present form, a matrix notation was used to define the coefficients of the unknowns. These are tabulated in Appendix C. After some manipulation of the governing equations, the following are obtained:

$$(g_1 D^7 + g_2 D^5 + g_3 D^3 + g_4 D) w_b = L_I(x) \quad (22)$$

$$(b_{11} D^4 + b_{12} D^2 + b_{13}) \beta_b = L_{II}(x) - (b_{14} D^3 + b_{15} D) w_b \quad (23)$$

$$a_{23} D \beta_a^2 + L_{III}(x) - (a_{21} D^2 + a_{22}) \beta_b - a_{24} D w_b \quad (24)$$

$$D^2 u_{ob} = L_{IV}(x) - k_1 D^2 \beta_a - k_2 D \beta_b - k_3 D w_b \quad (25)$$

$$\tau_j = L_V(x) - k_4 D^2 u_{ob} - k_5 D w_b \quad (26)$$

$$P_j = L_{VI}(x) - k_6 D^3 \beta_b - k_7 D \tau_j + k_8 D u_{ob} + k_9 w_b \quad (27)$$

III. SOLUTION OF GOVERNING EQUATIONS

Homogeneous Solution:

Consider first W_b . Assuming a solution of the form $W_b = e^{sx}$ and letting $y = s^2$, (22) then takes the form

$$y^3 + \frac{g_2}{g_1} y^2 + \frac{g_3}{g_1} y + \frac{g_4}{g_1} = 0 \quad (28)$$

for the homogeneous solution.

The solution of equations in the form (28) is developed in reference (42) and leads to three possibilities for the roots:

Case 1: there are two conjugate imaginary roots and one real root

Case 2: there are three real and unequal roots

Case 3: there are three real roots of which at least two are equal.

Case 1 leads to a solution of (22) which is of the form

$$W_{bH} = V_1 e^{c_1 x} + V_2 e^{-c_1 x} + e^{c_2 x} (V_3 \cos c_3 x + V_4 \sin c_3 x) + e^{-c_2 x} (V_5 \cos c_3 x + V_6 \sin c_3 x) \quad (29)$$

where $V_1 - V_6$ are constants to be evaluated through boundary conditions.

The constants $c_1 - c_3$ are defined in Appendix C.

The Case 2 solution can take on several forms depending on whether the roots of (28) are positive or negative. The final forms

for the case of one, two and three positive real roots respectively, are:

$$w_{bh} = v_1 e^{c_4 x} + v_2 e^{-c_4 x} + v_3 \cos c_5 x + v_4 \sin c_5 x \\ + v_5 \cos c_6 x + v_6 \sin c_6 x \quad (30)$$

$$w_{bh} = v_1 e^{c_4 x} + v_2 e^{-c_4 x} + v_3 e^{c_5 x} + v_4 e^{-c_5 x} \\ + v_5 \cos c_6 x + v_6 \sin c_6 x \quad (31)$$

$$w_{bh} = v_1 e^{c_4 x} + v_2 e^{-c_4 x} + v_3 e^{c_5 x} + v_4 e^{-c_5 x} \\ + v_5 e^{c_6 x} + v_6 e^{-c_6 x} \quad (32)$$

where, again, the $v_1 - v_6$ are boundary value constants and $c_4 - c_6$ are defined in Appendix C.

Case 3 represents the degenerate forms (30) - (32) where two of the roots are equal. The equations then take the form:

$$w_{bh} = v_1 e^{c_4 x} + v_2 e^{-c_4 x} + (v_3 + v_5 x) \cos c_5 x + (v_4 + v_6 x) \sin c_5 x \quad (33)$$

$$w_{bh} = (v_1 + v_3 x) e^{c_4 x} + (v_2 + v_4 x) e^{-c_4 x} + v_5 \cos c_6 x \\ + v_6 \sin c_6 x \quad (34)$$

$$w_{bh} = (v_1 + v_3 x) e^{c_4 x} + (v_2 + v_4 x) e^{-c_4 x} + v_5 e^{c_6 x} \\ + v_6 e^{-c_6 x}. \quad (35)$$

In treating single-layered cylinders, it is possible to write the $c_1 - c_6$ explicitly in terms of the physical quantities

involved and thus have a feel for the physical behavior of the shell. In the present derivation, however, these expressions are so lengthy and involved that, unless one has a specific example in mind, their explicit form for the general case is of dubious utility. A point is reached where one must decide whether to obscure the physical situation with mathematics or obscure the mathematics with the physical quantities involved. The former course was chosen to show where the mathematical formulation is analogous to that for the single-layered cylinder (Case 1) and where it diverges (Cases 2 and 3). To import greater physical significance to the equations it becomes necessary to either consider a specific case and determine the form and constants listed in Appendix C will take or resort to numerical evaluation of the constants.

Consider next equation (25). This may be integrated directly to give

$$U_{ob} = \iint L_{IV}(x) dx dx - k_1 \beta_a - k_2 \beta_b - k_3 \int w_b dx + V_7 x + V_8 \quad (36)$$

where V_7 and V_8 are also boundary value constants. Since there are also eight boundary conditions, four at each edge, it follows that the homogeneous solutions for the remaining unknowns β_a and β_b are not required. Their particular solutions will suffice to satisfy the governing equations. This is equivalent to setting the boundary value constants of the β_a and β_b homogeneous solutions to zero.

Total Solution:

The total solution will consist of the homogeneous solutions given in the previous section together with whatever particular solutions are called for due to the mechanical and thermal loading for any given problem. Total solutions for displacements will take the form:

$$w_b(x) = w_{bH} + w_{b \text{ part}} \quad (37)$$

$$\beta_b(x) = \beta_{b \text{ part}} \quad (38)$$

$$\beta_a(x) = \beta_{a \text{ part}} \quad (39)$$

where $w_{b \text{ part}}$ is the particular solution of (22) due to $L_I(x)$, and

$\beta_{b \text{ part}}$ is the particular solution of (23) due to $L_{II}(x) - (b_{14}D^3 + b_{15}D)w_b(x)$, and

$\beta_{a \text{ part}}$ is the particular solution of (24) due to $L_{III}(x) - (a_{21}D^2 + a_{22})\beta_b(x) - a_{24}DW_b(x)$.

Once (37) - (39) are known, u_{ob} may be found from (36) and the joint shear and normal stresses from (26) and (27) respectively.

IV. BOUNDARY CONDITIONS

Boundary conditions for plates and shells are listed in many sources (21), (34), (37), (36), (37). For the axi-symmetric case, they are usually stated as:

- At the edges $x = 0$ and $x = L$
- either u or N prescribed
 - either w or Q prescribed
 - either β or M prescribed
- (40)

For multilayered problems, the same boundary conditions usually apply if N , M and Q are interpreted as resultants \bar{N} , \bar{M} , \bar{Q} . Considering a two-layered cylinder for example, laminas a and b, the resultants would be:

$$\begin{aligned}\bar{N} &= N_a + N_b \\ \bar{Q} &= Q_a + Q_b \\ M &= M_a + M_b + \left(\frac{h_a}{2} + \frac{h_b}{2}\right)N_a\end{aligned}\tag{41}$$

However, (40), (41) do not make use of the no slip - no delamination conditions. These provide two constraints not only on displacements but on boundary conditions as well.

Appropriate boundary conditions can be derived using the principle of virtual displacements of the layer middle surfaces.

$$U_i^*$$

$$B_i^* \quad (i = a, b)$$

(42)

$$W_i^*$$

where the asterisk denotes a virtual quantity. Multiplying equations (10) by the virtual displacements (42) in the order listed, adding the products, integrating over shell length and summing over the layers, we get:

$$\sum_{i=a}^b \int_C \{ [N_{x_i} + (\tau_{1i} - \tau_{2i})] U_i^* +$$

$$[M_{x_i} - Q_i + 1/2 h_i (\tau_{1i} + \tau_{2i})] B_i^* + \quad (43)$$

$$[Q_i - (1/R) N_{\theta i} + (P_{1i} - P_{2i})] W_i^* \} dC = 0$$

since the virtual work done by a shell in equilibrium through a virtual displacement is equal to zero.

After integration by parts, the virtual work principle for the multilayered cylinder takes the form:

$$\begin{aligned} & \sum_{i=a}^b \{ [N_{x_i} U_i^* + M_{x_i} B_i^* + Q_i W_i^*]_0 \\ & + \int_0^L [(\tau_{1i} - \tau_{2i}) U_i^* + 1/2 h_i (\tau_{1i} + \tau_{2i}) B_i^* \end{aligned}$$

$$+ (p_{1i} - p_{2i}) w_i^*] dx \} =$$

$$\sum_{i=a}^b \int_0^L [N_{x_i} u_i^* + M_{x_i} \beta_i^* + Q_i (\beta_i^* + w_i^*)] dx \quad (44)$$

$$+ \frac{N_{\theta i}}{R} w_i \Big] dx$$

The first quantity in brackets on the left hand side of (44) involves terms which are specified at the boundaries, namely:

$$\sum_{i=a}^b [N_{x_i} u_i^* + M_{x_i} \beta_i^* + Q_i w_i^*]_0^L$$

Summing over layers gives

$$[N_{x_a} u_a^* + M_{x_a} \beta_a^* + Q_a w_a^* + N_{x_b} u_b^* + M_{x_b} \beta_b^* + Q_b w_b^*]_0^L$$

Since this derivation does not allow for slip or delamination, the virtual displacements must be constrained to:

$$u_a^* = u_b^* + (1/2)h_a \beta_a^* + (1/2)h_b \beta_b^* \quad (45)$$

$$w_a^* = w_b^* + (\bar{w}_b - \bar{w}_a)^*$$

Substituting into (49) and collecting terms, the quantities to be specified at $x = 0$ and $x = L$ are found to be:

either $N_{x_a} + N_{x_b}$ or u_b specified

either $1/2 h_a N_{x_a} + M_{x_a}$ or β_a specified

(46)

either $1/2 h_b N_{x_b} + M_{x_b}$ or β_b specified

either $Q_a + Q_b$ or w_b specified

$Q_a (\bar{w}_b - \bar{w}_a)$ specified

Note that the first four conditions are not a unique set.

Other possibilities are (u_a, u_b, β_a, w_b) or (u_a, u_b, β_b, w_b) specified, but the set (46) seems to be a good choice.

The last condition of (46) is the result of retaining thermal expansion through-the-thickness while dropping terms of order h/R throughout the remainder of the derivation. For cases where \bar{w} must be retained, the following procedure can be employed. For simplicity, free-free boundary conditions are considered, though the treatment is analogous for any set of conditions specified.

Since the last of (46) is a temperature dependent, $\bar{w}_b - \bar{w}_a$

is a priori specified. Hence, for free boundaries, either $Q_a = 0$ or $(\bar{w}_b - \bar{w}_a) = 0$. $Q_a = 0$ is acceptable for special cases but is not generally true. $(\bar{w}_b - \bar{w}_a)$ can be rigorously satisfied by redefining the reference surface location in layer a, i.e.,

$$\bar{w}_a(x, z_0) = \int_0^{z_0} \alpha_c a T_a(x, z) dz \quad (47)$$

$$w_b(x, \frac{h_b}{2}) = \int_0^{\frac{h_b}{2}} \alpha_c b T_b(x, z) dz$$

Under these conditions, the no-slip requirement becomes

$$u_{o_a} = u_{o_b} + (-z_0 \beta_a + 1/2 h_b \beta_b) \quad (48)$$

and all other results remain the same if $1/2 h_a$ is replaced by $-z_0$.

Note that when this approach is used, the second strain definition of equation (8) must be used in the form:

$$\epsilon_\theta = \frac{w_i + \bar{w}_i}{R_i + z} \quad (i = a, b)$$

in order to have a zero stress state for the case of an isotropic material with the same α_c in both layers and $T = \text{constant}$.

V. SAMPLE PROBLEM

The problem selected was the slow cooling of pyrolytic graphite. Because of the difference in thermal expansion coefficients in both "a" and "c" directions of the pyrolytic graphite and mandrel material and also because of curvature effects, normal and shear stresses at the deposit-mandrel interface will be formed during the cooling process. These may be of sufficient magnitude to cause flaking or delamination. An investigation of their behavior with changing material and geometric properties is therefore of interest.

The test case considered a laminated cylinder with free-free edges and a constant temperature $T_0 = -1000^{\circ}\text{F}$. The material properties used were averaged in the range $3000^{\circ}\text{F} - 2000^{\circ}\text{F}$. Layer a, the top-most layer, was taken to be pyrolytic graphite, layer b commercial (ATJ) graphite. The properties used are given in Table (B.1). The calculations were performed with the aid of a CDC 6600 computer. The program tabulation is given in Appendix A. Results showing the behavior of τ_j and p_j due to variation in lamina thickness and/or E/G_c ratio are in Appendix B. From these, the following conclusions may be drawn:

1. Figures (B.1) and (B.2) indicate a decrease in normal and shear stresses at the mandrel - PG interface as the E/G_c ratio is increases, implying that a material weak in shear is desirable to minimize stresses and therefore the possibility of debonding.

2. Figures (B.3) and (B.4) show results when $h = 0.50$, $E/G_c = 20$ and $5 \geq h_a/h_b \geq 1$. Figures (B.5) and (B.6) show the case where $h_b = 0.25$, $E/G_c = 20$, $4 \geq h_a/h_b \geq 1$. All the curves indicate that a high h_a/h_b ratio is the desirable, for given material properties, for minimization of normal and shear stresses at the joint. This implies that a thin graphite mandrel is preferable to a thick mandrel. Note that when $h_a/h_b = 1$, the ultimate stress ($\sigma_{ult} = 18,000$ psi) in tension for pyrolytic graphite is exceeded.

3. To determine the behavior of the roots of equation (22) and establish a range wherein the various forms of solution (29)-(35) will occur, computations were also made for $1000 \geq h_a/h_b \geq 0.001$ with E/G_c at 50, 20 and 2.6. Results show that the Case 1 solution in the form (29) occurs whenever $h_a/h_b \geq 1$ independent of the E/G_c ratio. The constants $c_1 - c_3$ are affected by E/G_c , the constant c_1 being much more sensitive to a change in E/G_c than c_2 or c_3 .

4. The first two terms of (29) have their principal effects at the edge only. It was also observed that for $h_a/h_b \gg 1$, the value c_1 becomes so very large that the boundary value constants v_1 and v_2 tend to become very small, and as a first approximation the terms containing them can be dropped from the solution functions. In this case, (29) takes on a form similar to that for the solution for radial displacement of a single-layered cylinder, with suitably defined constants.

5. It should be kept in mind that all the above remarks are based on the numerical results and are valid in the range.
 $1000 \geq h_a/h_b \geq 0.001$; $400 \geq L/h \geq 26$; $0.05 \geq h/R \geq 0.0033$

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APPENDIX A
Computer Program

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FORTRAN IV PROGRAM PGE(INPUT,OUTPUT,TAPES=INPUT,TAPE9=OUTPUT)      ***** 1
DIMENSION ICO(6), IIM(6),IRE(6)                                     PGE 2
DIMENSION ORQ(10), HP(10), HQ(10), COEF(5), G3ACO(5), ROOTS(2,6)   PGE 3
COMMON /SHEAR/ DA,QB,FQ                                         PGE 4
COMMON /KLP/ IC1,IC2,IC3,IC4,IC5,IC6,IM1,IM2,IM3,IM4,IM5,IM6,IR1,IPGE 5
IR2,IR3,IR4,IR5,IR6                                           PGE 6
COMMON DLTEMP                                                 PGE 7
COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUR,NUBC,HA,HB,R,G3A,G3B,CA,CB,PGE 8
1DA,DB,H,CACB,CAPCB,IUNU,GAMA,GAMB,K(40),A(55),G(7),RT(30),F4TOA,FNPGE 9
2TOB,BDE(600),LOAD1,AM(50),LOAD2,EN(6,7),FNXA,FNXB,FMXA,FMXB,FYX,FMPGE 10
3X,FYOA,FYOB,AC(3),KLIP,ELL,DELPH(20),V(6),AL(18),WW,DDWW,WB,DWPGE 11
4B,DDWB,DDDWB,WA,DWA,HU,DDHU,DDDWU,TAU,DTAU,WPJ,WHA,WWA,WUA,DWPGE 12
SUA                                                       PGE 13
EQUIVALENCE (IC1,ICO(1)), (IM1,IIM(1)), (IR1,IRE(1))          PGE 14E
REAL NUA,NUAC,NUB,NUBC,NUNU,K,LUAD1,LUAD2                      PGE 15
READ (5,12) (ORQ(I),I=1,4)                                     PGE 16R
READ (5,12) (HP(I),I=1,6)                                     PGE 17R
READ (5,12) (HQ(M),M=1,4)                                     PGE 18R
READ (5,11) TEMP,DLTEMP                                       PGE 19R
READ (5,11) EA,EAC,NUA,NUAC                                     PGE 20R
READ (5,11) EB,EBC,NUB,NUBC                                     PGE 21R
READ (5,9) BT(1),RT(2),BT(3),BT(4)                           PGE 22R
READ (5,10) (DELPH(J),J=1,11)                               PGE 23R
WRITE (9,13) (HQ(M),M=1,4)                               PGE 24W
WRITE (9,14) TEMP                                         PGE 25W
WRITE (9,15) EA,EAC,NUA,NUAC                                     PGE 26W
WRITE (9,16) EB,EBC,NUB,NUBC                                     PGE 27W
COEF(1)=50.                                              PGE 28
COEF(2)=20.                                              PGE 29
COEF(3)=2.6                                              PGE 30
DO 8 I14=1,6                                         PGE 31
DO 8 M=1,4                                         PGE 32
DO 8 L=1,3                                         PGE 33
HA=HP(14)                                             PGE 34
HB=HQ(M)                                              PGE 35
R=30.                                                 PGE 36
G3ACO(L)=EA/CCEF(L)                                     PGE 37
G3A=G3ACO(L)                                         PGE 38
G3B=EB/(2.*(1.+NUBC))                                PGE 39
CALL PRELIM (L,M,I14)                                 PGE 40
CALL POLYR (6,G,ROOTS,D)                            PGE 41
CALL RICF (ROOTS,AC,KLIP)                           PGE 42
CALL THERM                                         PGE 43
CALL MISC                                         PGE 44
DO 7 IJP=1,4                                         PGE 45
ELL=ORQ(IJP)                                         PGE 46
IF (ELL.EQ.0.0) GO TO 7                                PGE 47
AM(1)=AC(2)**2-AC(3)**2                                PGE 48
AM(2)=2.*(AC(2)*AC(3))                                PGE 49
AM(3)=AC(2)*AM(1)-AC(3)*AM(2)                         PGE 50
AM(4)=AC(3)*AM(1)+AC(2)*AM(2)                         PGE 51
AM(17)=AM(1)*AC(3)-AM(2)*AM(2)                       PGE 52
AM(18)=AM(1)*AC(2)+AM(2)*AC(3)                         PGE 53
AM(25)=AM(1)*AC(2)+AM(2)*AC(3)                         PGE 54
AM(26)=AM(1)*AC(3)-AM(2)*AC(2)                         PGE 55
AM(5)=AC(2)*AM(3)-AC(3)*AM(4)                         PGE 56
AM(6)=AC(3)*AM(3)+AC(2)*AM(4)                         PGE 57
IF (KLIP-5) 1,2,3                                     PGE 58
1 CONTINUE                                         PGE 59
IF (KLIP-2) 4,5,6                                     PGE 60

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2 CALL EKLIPS          PGE  61
GO TO 6               PGE  62
3 CALL EKLIP6          PGE  63
GO TO 6               PGE  64
4 CALL EKLIP1          PGE  65
GO TO 6               PGE  66
5 CALL EKLIP2          PGE  67
6 CONTINUE             PGE  68
CALL MISC              PGE  69
7 CONTINUE             PGE  70
8 CONTINUE             PGE  71
PGE 72
PGE 73
PGE 74
PGE 75
PGE 76
PGE 77
PGE 78
PGE 79
PGE 80
PGE 81
PGE 82
PGE 83-
***** 1
2
3
4
5
6
7
8
9
10
11
12
13
14
15
16
END
SUBROUTINE UC0FF (FNTHW,CONST,X)
COMMON DLTEMP
COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,NUBC,HA,HR,R,G3A,G3B,CA,CB,
1DA,DB,H,CACB,CAPCB,NUNU,GAMA,GAMB,K(40),A(55),G(7),BT(30),FNT0A,FN
2TOB,BDE(600),LOAD1,AM(50),LOAD2,EN(6,7),FNXA,FNXB,FMXA,FMXB,FNX,FM
3X,FN0A,FN0B,AC(3),KLIP,ELL,DELPH(20),V(6),AL(18),WH,DWW,DDWW,WR,DW
4B,DDWB,DDDWB,WA,DWA,WU,DWU,DDWU,TAU,DTAU,WPJ,WWA,DWHA,WUA,DW
SUA
REAL NUA,NUAC,NUB,NUBC,NUNU,K,LOAD1,LOAD2
PP4=(ELL/2.)*AC(1)           9
PP5=(ELL/2.)*AC(2)           10
PP6=(ELL/2.)*AC(3)           11
CONST=BUE(19)*WA+HDE(20)*WB+BDE(21)*FNT0H-BT(20)*X/CAPCB      12
RETURN
13
14
15-
***** 1
2
3
4
5
6
7
8
PRELI 1
PRELI 2
PRELI 3
PRELI 4
PRELI 5
PRELI 6
PRELI 7
PRELI 8
PRELI 9
PRELI 10
PRELI 11
PRELI 12
PRELI 13
PRELI 14
PRELI 15
PRELI 16
PRELI 17
PRELI 18
PRELI 19
PRELI 20
PRELI 21
PRELI 22
SUBROUTINE PRELIM (L,M,I14)
COMMON DLTEMP
COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,NUBC,HA,HR,R,G3A,G3B,CA,CB,PRELI
1DA,DB,H,CACB,CAPCB,NUNU,GAMA,GAMB,K(40),A(55),G(7),BT(30),FNT0A,FN
2TOB,RDE(600),LOAD1,AM(50),LOAD2,EN(6,7),FNXA,FNXB,FMXA,FMXB,FNX,FM
3X,FN0A,FN0B,AC(3),KLIP,ELL,DELPH(20),V(6),AL(18),WH,DWW,DDWW,WR,DW
4B,DDWB,DDDWB,WA,DWA,WU,DWU,DDWU,TAU,DTAU,WPJ,WWA,DWHA,WUA,DWPRELI
SUA
REAL NUA,NUAC,NUB,NUBC,NUNU,K,LOAD1,LOAD2
CA=EA*HA/(1.-NUA**2)          PRELI 8
CB=EB*HR/(1.-NUB**2)          PRELI 9
DA=EA*(1A**3)/(12.*(1.-NUA**2))  PRELI 10
DB=EB*(1B**3)/(12.*(1.-NUB**2))  PRELI 11
H=HA+HB
CACB=CA*CB
CAPCB=CA*CB
NUNU=NUA-NUB
GAMA=(5./6.)*(G3A*HA)
GAMB=(5./6.)*(G3B*HB)
K(15)=1.*CB/CA
K(16)=1.*CA/CB
K(17)=HA/HB

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A(1)=(1./12.)*(HA**2)+H*HA/(4.*K(16))          PREL123
A(2)=-HA*NNUU/(12.*R)*K(16)                     PREL124
A(3)=(HB+HB)*(K(15)/K(16))/12.+H*HB/(4.*K(16)) PREL125
A(4)=HB*A(12)/HA                                 PREL126
A(15)=-(H/HA)*A(12)                            PREL127
FDUM=(CA*NUA+CR*NUB)**2                         PREL128
CDUM=CAPCR**2                                    PREL129
BDUM=(CA*CAPCB)*(R**2)                          PREL130
A(16)=(FDUM-CDUM)/RDUM                         PREL131
A(21)=(HA**2)/12.+5./24.*((HB**2)/K(15))       PREL132
A(22)=-(5./6.)*((G3A/ER)*(1.-NUB**2))          PREL133
A(23)=(5./24.)*((HA*HB)/K(15))                  PREL134
A(24)=-(5./6.)*HB/HA*A(12)-A(22)                PREL135
A(31)=(1./12.)*(HA**2)*(1.+5./(2.*K(16)))      PREL136
A(32)=-(5./6.)*(G3A/ER)*(1.-NUA**2)              PREL137
A(33)=(K(15)/K(16))*A(23)                        PREL138
A(34)=(5./12.)*(HA/(R*K(16)))-GAMA/CA          PREL139
    A(41) = B(11)                                A(51) = B(21)
    A(42) = B(12)                                A(52) = B(22)
    A(43) = B(13)                                A(53) = B(23)
    A(44) = B(14)                                A(54) = B(24)
    A(45) = B(15)                                A(55) = B(25)          PREL145

A(41)=A(13)*A(23)-A(11)*A(21)                  PREL146
A(42)=A(14)*A(23)-A(11)*A(22)-A(21)*A(12)    PREL147
A(43)=-A(12)*A(22)                            PREL148
A(44)=A(15)*A(23)-A(11)*A(24)                  PREL149
A(45)=A(16)*A(23)-A(12)*A(24)                  PREL150
A(51)=A(23)**2-A(31)*A(21)                      PREL151
A(52)=-A(31)*A(22)-A(32)*A(21)                PREL152
A(53)=-A(32)*A(22)                            PREL153
A(54)=A(23)*A(34)-A(31)*A(24)                  PREL154
A(55)=-A(32)*A(24)                            PREL155
    G(1) = D(1)                                PREL156
    G(3) = D(2)                                PREL157
    G(5) = D(3)                                PREL158
    G(7) = D(4)                                PREL159
G(1)=A(51)*A(44)-A(41)*A(54)                  PREL160
G(3)=A(51)*A(45)+A(52)*A(44)-A(42)*A(54)-A(41)*A(55) PREL161
G(5)=A(52)*A(45)+A(53)*A(44)-A(42)*A(55)-A(43)*A(54) PREL162
G(7)=A(53)*A(45)-A(43)*A(55)                  PREL163
G(2)=0.0                                     PREL164
G(4)=G(2)                                    PREL165
G(6)=G(4)                                    PREL166
WRITE(9,3) L,G3A,G3B                         PREL167W
WRITE(9,4) HA,HB,R,TEMP                         PREL168W
WRITE(9,5) IOP,G1(IOP),IOP=1,7)                 PREL169W
WRITE(9,1) CA,CB,DA,DB                         PREL170W
WRITE(9,6) CACR,CAPCE,GAMA,GAMB             PREL171W
WRITE(9,8) HA,HB,ELL,L,K(17)                   PREL172W
WRITE(9,7) K(1)=1.0                           PREL173W
K(2)=G(3)/G(1)                                PREL174
K(3)=G(5)/G(1)                                PREL175
K(4)=G(7)/G(1)                                PREL176
FRUMP=(3.*K(3)-K(2)**2)/3.                     PREL177
HRMP=(2.*K(2)**3)-9.*K(2)*K(3)+27.*K(4))/27.   PREL178
    FRUMP = SMALL A
    HRMP = SMALL B
RATIO=HA/HB                                     PREL181
                                                PREL182

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DISC=(HRMP**2)/4.+ (FRUMP**3)/27.          PREL183
WRITE(9,2) HA,HB,RATIO,DISC                PREL184W
      DISCRIMINANT.LT. 0    IMPLIES KLIP = 2,3,4, OR 5 DEPENDING ON RPREL185
      DISCRIMINANT.EQ.0    IMPLIES DEGENERATE CASE           PREL186
      DISCRIMINANT.GT.0    KLIP = 1    MODIFIED CLASSICAL SOLUTION PREL187
      RETURN               PREL188
                                PREL189
                                PREL190
1 FORMAT (1X,3HCA=E12.5,1X,3HCB=E12.5,1X,3HDA=E12.5,1X,3HDB=E12.5/) PREL191
2 FORMAT (1/1X,3HHA=F10.3,1X,3HHB=F10.3,5X,6HHA/HB=F15.7,5X,13HUISCRPREL192
      IMIIVANT=E15.7//)                         PREL193
3 FORMAT (2X,4HG3A(12,2H)=E15.7,4HG3B:E15.7/1)                   PREL194
4 FORMAT (2X,4H HA=F10.2,2X,4H HB=F10.2,2X,2HR=F10.2,2X,6H TEMP=F10.PREL195
      12/1
5 FORMAT (4(2X,2HG(12,2H)=E12.5)/3(2X,2HG(12,2H)=E12.5))        PREL196
6 FORMAT (2X,5HCACB=E12.5,1X,6HCAPCB=E12.5,1X,5HGAMA=E12.5,1X,5HGAMBPKEL197
      1=E12.5/)                         PREL198
7 FORMAT (20H A(1) ARE      )                  PREL199
8 FORMAT (F12.5,F12.5,F12.5,I3,1X,7H HA/HB=F12.5)                 PREL1
      END
      SUBROUTINE MISC
      COMMON DLTEMP
      COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,NUBC,HA,HB,R,C3A,G3B,CA,CB,
      1DA,DB,H,CACB,CAPCB,NUNU,GAMA,GAMB,K(49),A(55),G(7),HT(30),FTD9,A,FN
      2TOB,RDE(600),LOAD1,AM(50),LOAD2,EN(6,7),FNXA,FNX9,FMXA,FMXB,FNX,FM
      3X,FYOA,FNOH,AC(3),KLIP,ELL,DELPH(20),V(6),AL(18),WH,DWH,DUWH,HB,DH
      4B,DOWB,DDDWB,WA,DWA,WU,DHU,DUWU,TAU,DTAU,WPJ,WHA,DHWA,WUA,DW
      SUA                                         5
      REAL NUA,NUAC,NUB,NUBC,NUNU,K,LOAD1,LOAD2
      BDE(19)=CA*HA/(2.*CAPCB)                      6
      BDE(20)=(HR/HA)*BDE(19)                        7
      BDE(21)=(1./R)*(CA*NUA+CB*NUB)/CAPCB          8
      BDE(22)=CB                                     9
      BDE(23)=BDE(22)*NUB/R                         10
      BDE(24)=(HB**2)*BDE(22)/12.                    11
      BDE(25)=HR/2.                                  12
      BDE(26)=BDE(22)/(R**2)                         13
      K(1)=BDE(19)                                    14
      K(2)=BDE(20)                                    15
      K(3)=BDE(21)                                    16
      K(4)=BDE(22)                                    17
      K(5)=K(8)=BDE(23)                            18
      K(6)=BDE(24)                                    19
      K(7)=BDE(25)                                    20
      K(9)=BDE(26)                                    21
      AM(24)=AC(2)**2+AC(3)**2                      22
      BDE(37)=GAMA*GAMB                           23
      RDE(38)=RDE(37)-(H/12.)*BDE(23)              24
      LOAD2=BT(11)                                    25
      LOAD1=0.0                                      26
      BDE(61)=BDE(19)*RDE(27)                      27
      BDE(62)=BDE(20)*BDE(22)                      28
      BDE(63)=BDE(21)*BDE(22)-BDE(23)             29
      BDE(64)=BDE(24)+BDE(25)*(BDE(22)*BDE(20))  30
      BDE(65)=RDE(25)*(BDE(22)*BDE(19))            31
                                              32
                                              33
                                              34
                                              35
                                              36
                                              37
                                              38
                                              39
                                              40

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BDE(66)=BDE(25)+BDE(63).	41
BDE(67)=BDE(23)+BDE(19)	42
BDE(68)=BDE(23)+BDE(20)	43
BDE(69)=BDE(26)-BDE(23)+BDE(21)	44
AM(32)=(CA*NUA+CB*NUB)/R	45
AM(33)=CA*HA/2.	46
AM(34)=CA*HR/2.	47
AM(35)=DA+(H*HA)*CA/4.	48
AM(36)=DB+(H*HB)*CA/4.	49
AM(37)=AM(33)+AM(34)	50
AM(38)=AM(37)*NUA/R	51
AM(39)=GAMA+GAMB	52
AM(42)=DA+(CA/4.)*(HA**2)	53
AM(43)=(HA*HB)*(CA/4.)	54
BDE(81)=AM(33)*NUA/R	55
BDE(82)=AM(34)*NUA/R	56
AM(44)=AM(42)-AM(33)*BDE(19)	57
AM(45)=AM(43)-AM(33)*BDE(20)	58
AM(46)=BDE(81)-AM(33)*BDE(21)	59
AM(47)=DB+(CA/4.)*(HB**2)	60
AM(48)=AM(43)-AM(34)*BDE(19)	61
AM(49)=AM(47)-AM(34)*BDE(20)	62
AM(50)=BDE(82)-AM(34)*BDE(21)	63
BDE(63)=EH/12.)*((BDE(22)*BDE(19))	64
BDE(64)=EH/12.)*((BDE(22)*BDE(20))	65
BDE(65)=AM(34)+(H/12.)*(BDE(22)*BDE(21)-BDE(23))	66
BDE(71)=AM(33)-CA*BDE(19)	67
BDE(72)=AM(34)-CA*BDE(20)	68
BDE(73)=CA*(NUA/R-BDE(21))	69
BDE(74)=CA*(NUA+BT(4)/R+BT(20)/CAPCB)-BT(5)/(L.-NUA)	70
BDE(75)=-CA*BDE(19)	71
BDE(76)=-CB*BDE(20)	72
BDE(77)=C9*(NUB/R-BDE(21))	73
BDE(91)=AM(44)+AM(48)	74
BDE(92)=AM(45)+AM(49)	75
BDE(93)=AM(46)+AM(50)	76
	77
RETURN	78
END	79

SUBROUTINE THERM
 COMMON DLTEMP
 COMMON TEMP,EA,EAC,NUA,NUAC,NUB,NUBC,NUNU,K,LOAD1,LOAD2
 IDA,DB,H,CACB,CAPCB,NUNU,GAMA,GAMB,K(40),A(55),G(7),RT(30),FNTOA,FNTHERM
 2TOB,BDE(1930),LNADL,AM(50),LNAD2,E(6,7),FNXA,FNXR,FMXA,FMXB,FNX,FMTHERM
 3X,FYOA,FZOB,AC(3),KLIP,ELL,DELPH(20),V(6),AL(18),WW,DDWW,HB,DWTHERM
 4B,DDWB,DDWW,MA,DWA,NU,DU,DDNU,DDWW,TAU,DTAU,WPJ,WHA,UNWA,WUA,CWTHERM
 SUA
 REAL NUA,NUAC,NUB,NUBC,NUNU,K,LOAD1,LOAD2
 BT(1) = ALPHA(A)
 BT(2) = ALPHA(AC)
 BT(3) = ALPHA(B)
 BT(4) = ALPHA(BC)
 BT(5) = NTXA = NTOA
 BT(6) = NTXB = NTOB
 BT(7) = H(BAR-A) AT Z = -HA/2.
 BT(8) = H(BAR-A) AT Z = +HB/2.
 BT(9) = E(BARI) = H(RAR-B) - H(BAR-A)
 BT(10) = ALPHA(1,7)
 BT(11) = ALPHA(2,7)
 BT(12) = NTXA

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BT(13) = NTXB                                     THERM22
BT(14) = EXPANSION IN THICKNESS DIRECTION - A   THERM23
BT(15) = EXPANSION IN THICKNESS DIRECTION - B   THERM24
BT(16) = MODIFIED AXIAL THERMAL RESULTANT A      THERM25
BT(17) = AXIAL THERMAL RESULTANT (MODIFIED) - B   THERM26
BT(18) = BT(16)                                     THERM27
BT(19) = BT(15)                                     THERM28
BT(5)=(EA*HA)*(TEMP*BT(1))                         THERM29
BT(6)=(ER*HB)*(TEMP*BT(2))                         THERM30
BT(7)=-(BT(2)*(TEMP*HA/2.1)+DLTEMP*(BT(2)*HA/8.)) THERM31
BT(8)=BT(4)*(TEMP*HB/2.1)+DLTEMP*(BT(4)*HB/8.1)   THERM32
BT(9)=BT(8)-BT(7)                                   THERM33
BT(10)=BT(5)/(R*(1.-NUA))-CA*BT(9)/(R**2)-(DA/(2.*HA))*(DLTEMP/(R*TH
1*2))+BT(2)                                         THERM34
BT(11)=BT(6)/(R*(1.-NUB))-(DB/(2.*HB))*(DLTEMP/(R**2))*BT(4)           THERM35
BT(12)=(TEMP*BT(1))*(EA*HA**3)/(12.*R)+(EA*HA**3)*(DLTEMP*BT(1))/(THERM37
112.*HA)                                           THERM38
BT(13)=(TEMP*BT(3))*(EB*HB**3)/(12.*R)+(EB*HB**3)*(DLTEMP*BT(3))/(THERM39
112.*HB)                                           THERM40
BT(14)=(DA*NUA)*(TEMP*BT(2))/R                   THERM41
BT(15)=(DB*NUB)*(TEMP*BT(4))/R                   THERM42
BT(16)=BT(5)/(1.-NUA)-CA*(NUA*BT(9))/R-(DA+DLTEMP/(2.*HA))*(NUA*BT(2)/R) THERM43
BT(17)=BT(6)/(1.-NUB)-(DB+DLTEMP/(2.*HB))*(NUB*BT(4)/R)                   THERM44
BT(18)=(DA*NUA/R)*(BT(2)*TEMP)                  THERM45
BT(19)=(DB*NUB/R)*(BT(4)*TEMP)                  THERM46
BT(20)=BT(16)+BT(17)                           THERM47
BT(21)=(DA*(DLTEMP*BT(2))/(2.*HA))*(NUA/R)       THERM48
BT(22)=(DB*(DLTEMP*BT(4))/(2.*HB))*(NUB/R)       THERM49
PP3=A(23)*A(53)                                 THERM50
PP10=A(32)*A(43)-A(12)*A(53)                  THERM51
ADUM=(PP3/CA)*(BT(19)+BT(11))                  THERM52
BDUM=(CA*NUA*CB*NUB)/(CA*(R*CAPCB))           THERM53
CDUM=PP3*(BT(16)+BT(17))                        THERM54
DOUM=RDUM*CDUM                                  THERM55
EDUM=ADUM+DOUM                                  THERM56
FDUM=CA*HH/(CH*CAPCB)                          THERM57
GDUM=IFDUM*(5./12.1)*(PP10*BT(5)/(1.-NUA))    THERM58
HDUM=(5./12.1)*(HR*PP10/CAPCB)*BT(16)          THERM59
PDUM=EDUM-GDUM+HDUM                            THERM60
QDUM=(PP10/LC)*(BT(13)-BT(18))                 THERM61
ROUM=PDUM+QDUM                                  THERM62
BDE(49)=ROUM/G(7)                               THERM63
WRITE (9,1)                                     THERM64
WRITE (9,4) ADUM,BDUM,CDUM,DOUM,EDUM,FDUM,GDUM,HDUM,PDUM,ROUM,BDE(THE
149)                                              THERM65W
WRITE (9,2)                                     THERM67W
WRITE (9,3) (BT(I),I=1,30)                      THERM68W
RETURN                                            THERM69W
1 FORMAT (1X,35) COMPONENTS OF BDE(49) ARE        /
2 FORMAT (1X,1CH BT ARE   )                      THERM70
3 FORMAT (1X,6E12.5)                           THERM71
4 FORMAT (1X,4E15.7)                           THERM72
END
SUBROUTINE EKLIP1
DIMENSION JIP(11)
COMMON /SHEAR/ QA,QR,FQ
COMMON DLTEMP
COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,NUBC,HA,HR,R,G3A,G3B,CA,CB,EKLIP

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1DA,DR,M,CACB,CAPCB,NUNU,GAMA,GAMH,K(40),A(55),G(7),RT(30),FNTOA,FNEKLIP 6
 2T08,BDE(600),LOAD1,AM(50),LOAD2,EN(6,7),FNXA,FNXB,FMXA,FMXB,FNX,FNEKLIP 7
 3X,FYOA,FYOB,AC(3),KLIP,ELL,BELPH(20),V(6),AL(18),WH,DWW,DDWU,WB,DWEKLIP 8
 4R,DDWB,DDDWB,WA,DWA,WU,DWU,DDWU,TAU,OTAU,WPJ,WHA,DWWA,WUA,DWEKLIP 9
 SUA
 REAL NUA,YUAC,NUB,MIBC,NUNU,K,LOAD1,LOAD2 EKLIP10
 REAL JIP EKLIP11
 P4=AC(1)*ELL EKLIP12
 P5=AC(2)*ELL EKLIP13
 P6=AC(3)*ELL EKLIP14
 BDE(1)=A(44)*(AC(1)**3)+A(45)*AC(1) EKLIP15
 BDE(2)=A(41)*(AC(1)**4)+A(42)*(AC(1)**2)+A(43) EKLIP16
 AM(30)=BDE(1)/BDE(2) EKLIP17
 BDE(3)=AM(5)*A(41)+AM(1)*A(42)+A(43) EKLIP18
 BDE(4)=AM(6)*A(41)+AM(2)*A(42) EKLIP19
 BDE(5)=AM(3)*A(44)+AC(2)*A(45) EKLIP20
 BDE(6)=AM(4)*A(44)+AC(3)*A(45) EKLIP21
 BDE(7)=BDE(5)*BDF(3)+BDE(6)*BDE(4) EKLIP22
 BDE(8)=BDE(6)*BDE(3)-PDE(5)*PDE(4) EKLIP23
 BDE(9)=BDE(3)**2+BDE(4)**2 EKLIP24
 BDE(10)=BDE(7)/BDE(4) EKLIP25
 BDE(11)=BDE(8)/BDE(3) EKLIP26
 AM(7)=-A(24)*AC(1)+AM(30)*(A(21)*(AC(1)**2)+A(22)) EKLIP27
 AM(8)=A(23)*(AC(1)**2) EKLIP28
 BDE(12)=AM(7)/AM(8) EKLIP29
 AM(19)=A(21)*AM(1)+A(22) EKLIP30
 AM(20)=A(21)*AM(2) EKLIP31
 AM(21)=A(24)*AC(2) EKLIP32
 AM(22)=A(24)*AC(3) EKLIP33
 BDE(13)=AM(19)*BDE(10)-AM(20)*BDE(11)-AM(21) EKLIP34
 BDE(14)=AM(19)*BDE(11)+AM(20)*BDE(13)-AM(22) EKLIP35
 BDE(15)=BDE(13)/A(23) EKLIP36
 BDE(16)=BDE(14)/A(23) EKLIP37
 AM(9)=AM(1)**2+AM(2)**2 EKLIP38
 AM(10)=AM(1)*BDE(15)+AM(2)*BDE(16) EKLIP39
 AM(11)=AM(1)*BDE(16)-AM(2)*BDE(15) EKLIP40
 BDE(17)=AM(13)/AM(4) EKLIP41
 BDE(18)=AM(11)/AM(9) EKLIP42
 AM(13)=-BDE(14)*BDE(12)+BDE(20)*AM(30)-BDE(21)/AC(1) EKLIP43
 BDE(27)=BDE(19)*(AM(2)*BDE(18)-AM(1)*BDE(17))+BDE(20)*(AM(1)*BDE(11) EKLIP44
 10)-AM(2)*BDE(11))-BDE(21)*AC(2) EKLIP45
 BDE(28)=-BDE(19)*(AM(2)*BDE(17)+AM(1)*BDE(18))+BDE(20)*(AM(1)*BDE(11) EKLIP46
 11)+AM(2)*BDE(10))-BDE(21)*AC(3) EKLIP47
 BDE(29)=AM(1)*BDE(27)+AM(2)*BDE(28) EKLIP48
 BDE(30)=AM(1)*BDE(28)-AM(2)*BDE(27) EKLIP49
 AM(14)=BDE(29)/AM(9) EKLIP50
 AM(15)=BDE(30)/AM(9) EKLIP51
 BDE(31)=EXP(P5)*COS(P6) EKLIP52
 BDE(32)=EXP(P5)*SIN(P6) EKLIP53
 BDE(33)=EXP(-P5)*COS(P6) EKLIP54
 BDE(34)=EXP(-P5)*SIN(P6) EKLIP55
 BDE(51)=AC(1)*(HA*BDE(12)-HB*AM(30)) EKLIP56
 BDE(52)=-BDE(16)*HA+BDE(11)*HB EKLIP57
 BDE(53)=BDE(17)*HA-BDE(10)*HB EKLIP58
 BDE(54)=AC(3)*BDE(52)+AC(2)*BDE(53) EKLIP59
 BDE(55)=AC(2)*BDE(52)-AC(3)*BDE(53) EKLIP60
 AM(12)=BT(20)/CAPCB-BDE(21)*BT(19) EKLIP61
 BDE(70)=LOAD2*BDE(23)*RT(20)/CAPCB EKLIP62
 EKLIP63
 EKLIP64
 EKLIP65

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BDE(78)=CB*BT(20)/CAPCB-BT(6)/(1.-NUB) EKLIP66
DO I I=1,6 EKLIP67
V(I)=0.0 EKLIP68
DO I J=1,7 EKLIP69
EN(I,J)=J.0 EKLIP70
1 CONTINUE EKLIP71
EN(1,1)=AC(1)*AM(44)*BDE(12)-AM(45)*AM(30)+AM(46) EKLIP72
EN(1,2)=EN(1,1) EKLIP73
EN(1,3)=AM(44)*AC(2)*BDE(17)-AC(3)*BDE(18)+AM(45)*(AC(3)*BDE(11))EKLIP74
1-AC(2)*BDE(10)+AM(46) EKLIP75
EN(1,4)=AM(44)*AC(3)*BDE(17)+AC(2)*BDE(18))-AM(45)*(AC(3)*BDE(10))EKLIP76
1+AC(2)*BDE(11) EKLIP77
EN(1,5)=EN(1,3) EKLIP78
EN(1,6)=-EN(1,4) EKLIP79
EN(2,1)=AC(1)*(BDE(12)*AM(48)-AM(30)*AM(49))+AM(50) EKLIP80
EN(2,2)=EN(2,1) EKLIP81
EN(2,3)=AM(44)*(-AC(3)*BDE(18)+AC(2)*BDE(17))+AM(49)*(AC(3)*BDE(11))EKLIP82
1)-AC(2)*BDE(10))+AM(50) EKLIP83
EN(2,4)=AM(44)*(AC(3)*BDE(17)+AC(2)*BDE(18))-AM(49)*(AC(3)*BDE(10))EKLIP84
1+AC(2)*BDE(11) EKLIP85
EN(2,5)=EN(2,3) EKLIP86
EN(2,6)=-EN(2,4) EKLIP87
EN(3,1)=PDE(85)*AC(1)-AM(30)*(BDE(84)*(AC(1)**2)+GAMB)*BDE(12)*(BDE)EKLIP88
1*(AC(1)**2)+GAMA) EKLIP89
EN(3,2)=-EN(3,1) EKLIP90
EN(3,3)=BDE(85)*AC(2)-BDE(10)*(GAMB+BDE(34)*AM(1))+BDE(11)*(BDE(84)EKLIP91
1)*AM(2))+BDE(17)*(GAMA+AM(1)*BDE(83))-BDE(18)*(PDE(83)*AM(2)) EKLIP92
EN(3,4)=AC(3)*BDE(85)-BDE(11)*(GAMB+BDE(84)*AM(1))-BDE(10)*(BDE(84)EKLIP93
1)*AM(2))+BDE(13)*(GAMA+AM(1)*BDE(83))+BDE(17)*(BDE(83)*AM(2)) EKLIP94
EN(3,5)=-EN(3,3) EKLIP95
EN(3,6)=EN(3,4) EKLIP96
EN(4,1)=EXP(P4)*EN(1,1) EKLIP97
EN(4,2)=EXP(-P4)*EN(1,2) EKLIP98
EN(4,3)=EN(1,3)*PDE(31)-EN(1,4)*BDE(32) EKLIP99
EN(4,4)=EN(1,3)*PDE(32)+EN(1,4)*BDE(31) EKLIP
EN(4,5)=EN(1,5)*PDE(33)-EN(1,6)*BDE(34) EKLIP
EN(4,6)=EN(1,5)*PDE(34)+EN(1,6)*BDE(33) EKLIP
EN(5,1)=EXP(P4)*EN(2,1) EKLIP
EN(5,2)=EXP(-P4)*EN(2,2) EKLIP
EN(5,3)=EN(2,3)*BDE(31)-EN(2,4)*BDE(32) EKLIP
EN(5,4)=EN(2,3)*BDE(32)+EN(2,4)*BDE(31) EKLIP
EN(5,5)=EN(2,5)*PDE(33)-EN(2,6)*PDE(34) EKLIP
EN(5,6)=EN(2,5)*PDE(34)+EN(2,6)*PDE(33) EKLIP
EN(6,1)=EXP(P4)*EN(3,1) EKLIP
EN(6,2)=EXP(-P4)*EN(3,2) EKLIP
EN(6,3)=EN(3,3)*PDE(31)-EN(3,4)*PDE(32) EKLIP
EN(6,4)=EN(3,3)*PDE(32)+EN(3,4)*PDE(31) EKLIP
EN(6,5)=EN(3,5)*PDE(33)-EN(3,6)*PDE(34) EKLIP
EN(6,6)=EN(3,5)*PDE(34)+EN(3,6)*PDE(33) EKLIP
EN(6,7)=AM(33)*BT(20)/CAPCB+BDE(81)*BT(9)+BT(18)-(HA/2.)*(BT(5)/(1.EKLIP
1.-NUA))-BT(12)/(1.-NUA)+(HA/2.)*BT(21)+AM(45)*BDE(49) EKLIP
EN(2,7)=AM(34)*BT(20)/CAPCB+BDE(92)*BT(9)+BT(19)-(HB/2.)*(BT(5)/(1.EKLIP
1.-NUA))-BT(13)/(1.-NUA)+(HB/2.)*BT(22)+AM(50)*BDE(49) EKLIP
EN(1,7)=-EN(1,7) EKLIP
EN(2,7)=-EN(2,7) EKLIP
EN(3,7)=LGAD1 EKLIP
EN(4,7)=EN(1,7) EKLIP
EN(5,7)=EN(2,7) EKLIP
EN(6,7)=EN(3,7) EKLIP

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      WRITE (9,3) (BDE(I),I=1,100)          EKLP
      WRITE (9,5)          EKLP
      WRITE (9,4) (AM(J),J=1,50)          EKLP
      WRITE (9,10)          EKLP
      WRITE (9,6) ((EN(I,J),J=1,7),I=1,6)    EKLP
      CALL F1YGLE (EN,VI)          EKLP
      WRITE (9,7) (I,VI(I),I=1,6)          EKLP

      AM(12)=BT(20)/CAPCA-BDE(21)*BDE(49)
      P1=AC(1)*ELL/2.          EKLP
      P2=AC(2)*ELL/2.          EKLP
      P3=AC(3)*ELL/2.          EKLP
      CALL CHECK1 (V,AC,P1,P2,P3)          EKLP
      WRITE (9,7) (I,V(I),I=1,6)          EKLP

      AL(1)=AM(30)*V(1)          EKLP
      AL(2)=AM(30)*V(2)          EKLP
      AL(3)=-BDE(10)*V(3)-BDE(11)*V(4)    EKLP
      AL(4)=BDE(11)*V(3)-ADE(10)*V(4)    EKLP
      AL(5)=BDE(10)*V(5)-HUE(11)*V(6)    EKLP
      AL(6)=BDE(11)*V(5)+ADE(10)*V(6)    EKLP
      AL(7)=BDE(12)*V(1)          EKLP
      AL(8)=-BDE(12)*V(2)          EKLP
      AL(9)=BDE(11)*V(3)+BDE(18)*V(4)    EKLP
      AL(10)=-BDE(18)*V(3)+BDE(17)*V(4)    EKLP
      AL(11)=-BDE(17)*V(5)+BDE(18)*V(6)    EKLP
      AL(12)=-BDE(18)*V(5)-ADE(17)*V(6)    EKLP

      AL(13)=AM(13)*V(1)          EKLP
      AL(14)=-AM(13)*V(2)          EKLP
      AL(15)=AM(14)*V(3)+AM(15)*V(4)    EKLP
      AL(16)=-AM(15)*V(3)+AM(14)*V(4)    EKLP
      AL(17)=-AM(14)*V(5)+AM(15)*V(6)    EKLP
      AL(18)=-AM(15)*V(5)-AM(14)*V(6)    EKLP
      WRITE (9,8)          EKLP
      WRITE (9,7) (AL(I),I=1,18)          EKLP

      DO 2 KF=1,11          EKLP
      X=DELPH(KF)          EKLP
      IF (X.GT.ELL) GO TO 2          EKLP
      P1=AC(1)*X          EKLP
      P2=AC(2)*X          EKLP
      P3=AC(3)*X          EKLP

      CALL DIFF (P1,P2,P3)          EKLP
      EKLP
      EKLP

      WW=V(1)*EXP(P1)+V(2)*EXP(-P1)+EXP(P2)*(V(3)*COS(P3)+V(4)*SIN(P3))+EKLIP
      IEXP(-P2)*(V(5)*COS(P3)+V(6)*SIN(P3))+BDE(49)          EKLP
      DWW=V(1)*BDE(10)+V(2)*BDE(11)+V(3)*BDE(12)+V(4)*BDE(13)+V(5)*BDE(14)          EKLP
      IDE(14)+V(6)*BDE(15)          EKLP
      ODMW=V(1)*RDE(10)+V(2)*RDE(11)+V(3)*RDE(12)+V(4)*RDE(13)+V(5)*RDE(14)          EKLP
      IBDE(14)+V(6)*BDE(15)          EKLP
      WB=AL(1)*EXP(P1)+AL(2)*EXP(-P1)+EXP(P2)*(AL(3)*COS(P3)+AL(4)*SIN(P3))          EKLP
      AL(3)*EXP(-P2)*(AL(5)*COS(P3)+AL(6)*SIN(P3))          EKLP
      DMB=AL(1)*BDE(10)+AL(2)*BDE(11)+AL(3)*BDE(12)+AL(4)*BDE(13)+ALEKLP

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1(S1*BDE(141)*AL(6)*BDE(151) EKLI P
  DDBWB=AL(1)*BDE(102)+AL(2)*BDE(112)+AL(3)*BDE(122)+AL(4)*BDE(132)+AEKLI P
1L(5)*BDE(142)+AL(6)*BDE(152) EKLI P
  DDDWB=AL(1)*BDE(103)+AL(2)*BDE(113)+AL(3)*BDE(123)+AL(4)*BDE(133)+EKLI P
1AL(5)*BDE(143)+AL(6)*BDE(153) EKLI P
  WA=AL(7)*EXP(P1)+AL(8)*EXP(-P1)+EXP(P2)*(AL(9)*COS(P3)+AL(10)*SIN(EKLI P
1P3))+EXP(-P2)*(AL(11)*COS(P3)+AL(12)*SIN(P3)) EKLI P
  DWA=AL(7)*BDE(101)+AL(8)*BDE(111)+AL(9)*BDE(121)+AL(10)*BDE(131)+AEKLI P
1L(11)*BDE(141)+AL(12)*BDE(151) EKLI P
  DDWA=AL(7)*BDE(102)+AL(8)*BDE(112)+AL(9)*BDE(122)+AL(10)*BDE(132)+EKLI P
1AL(11)*BDE(142)+AL(12)*BDE(152) EKLI P
  DDDWA=AL(7)*BDE(103)+AL(8)*BDE(113)+AL(9)*BDE(123)+AL(10)*BDE(133)EKLI P
1+AL(11)*BDE(143)+AL(12)*BDE(153) EKLI P
EKLI P

FNTWH=(1./AC(1))*(V(1)*EXP(P1)-V(2)*EXP(-P1))+(EXP(P2)/AM(24))*(V(EKLI P
13)*(AC(2)*COS(P3)+AC(3)*SIN(P3))+V(4)*(AC(2)*SIN(P3)-AC(3)*COS(P3)EKLI P
2))+EXP(-P2)/AM(24))*(V(5)*(-AC(2)*COS(P3)+AC(3)*SIN(P3))-V(6)*(ACEKLI P
3(2)*SIN(P3)+AC(3)*COS(P3)))+BDE(47)*X EKLI P
IF (KF,EV,1) CALL UC OFF (FNTWH,CNST,0) EKLI P
WU=-BDE(19)*WA-BDE(20)*WR-BDE(21)*FNTWH+BTE(20)/CAPCB)*X+CUYST EKLI P
DWU=-BDE(19)*DWA-BDE(20)*DWR-BDE(21)*WU+BT(20)/CAPCB EKLI P
DDWU=-BDE(19)*DDWA-BDE(20)*DDWNB-BDE(21)*DDWW EKLI P
DDDWU=-BUE(19)*DDDWHA-BDE(20)*DDDWB-BDE(21)*DDWW EKLI P
TAU=LOAD1-BDE(22)*DDWU-BDE(23)*DDWW EKLI P
DTAU=-BDE(22)*DDWU-BDE(23)*DDWW EKLI P
WPJ=LOAD2-BDE(24)*DDDWB-BDE(25)*UTAU+BDE(23)*DWU+BDE(26)*WW EKLI P
WUA=WU+HA*WA/2.+HB*W/2. EKLI P
DWUA=DWU+HA*DWA/2.+HB*DWB/2. EKLI P
DDWUA=DDWU+HA*DDWA/2.+HB*DDWB/2. EKLI P
CALL RSLT (X) EKLI P
CALL PRINT (X) EKLI P

2 CONTINUE EKLI P
RETURN EKLI P

3 FORMAT (1X,9H BDE ARE /10E12.5) EKLI P
4 FORMAT (10E12.5) EKLI P
5 FORMAT (20H AM(J) ARE ) EKLI P
6 FORMAT (1X,7E11.4) EKLI P
7 FORMAT (4(1X,2HV(1,2H)=E12.5)/2(1X,2HV(1,2H)=E12.5)) EKLI P
8 FORMAT (1X,2CH AL(I) ARE ) EKLI P
9 FORMAT (6E12.5) EKLI P
10 FORMAT (21H EN(I,J)BY ROWS ARE ) EKLI P
END EKLI P
SURROUNGEKLI P2
DIMENSION JIP(11) EKLI P 2
COMMON /SHEAR/ QA,OR,FQ EKLI P 3
COMMON DLTEMP EKLI P 4
COMMON TEMP,EA,EAC,NUA,YUAC,EB,ERC,NUB,YUBC,MA,HA,R,G3A,G3B,CA,CB,EKLI P 5
1DA,DR,H,CACB,CAPCB,YUNU,GAMA,GAVB,K(40),A(55),G(7),H(30),FNT0A,FNEKLI P 6
2T0B,BDE(600),LOAD1,AM(5),LOAD2,E416,7),FNXA,FNXB,FYXA,FHKB,FYX,FMFKLIP 7
3X,FYCA,FYCB,AC(3),KLIP,ELL,DELPHI(20),V(6),AL(18),WW,DWW,UDWW,WR,DWEKLI P 8
4B,DDWB,DDWNB,WA,DWA,WU,DDWU,DDDWU,TAU,DTAU,WPJ,WWA,WWA,WUA,DWEKLI P 9
5UA EKLI P 10
REAL NUA,NUAC,NUB,YUBC,YUNU,K,LOAD1,LOAD2 EKLI P 11
REAL JIP EKLI P 12
P4=AC(1)*ELL EKLI P 13
P5=AC(2)*ELL EKLI P 14
P6=AC(3)*ELL EKLI P 15
BDE(1)=A(44)*(AC(1)**3)-A(45)*AC(1) EKLI P 16

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BDE(2)=A(41)*(AC(1)**4)-A(42)*(AC(1)**2)+A(43)	EKLIP17
AM(30)=BDE(11)/BDE(2)	EKLIP18
BDE(3)=AM(5)*A(41)+AM(1)*A(42)+A(43)	EKLIP19
BDE(4)=AM(6)*A(41)+AM(2)*A(42)	EKLIP20
BDE(5)=AM(3)*A(46)+AC(2)*A(45)	EKLIP21
BDE(6)=AM(4)*A(46)+AC(3)*A(45)	EKLIP22
BDE(7)=BDE(5)*BDE(3)+BDE(6)*BDE(4)	EKLIP23
BDE(8)=BDE(6)*BDE(3)-BDE(5)*BDE(4)	EKLIP24
BDE(9)=BDE(3)**2+BDE(4)**2	EKLIP25
BDE(10)=BDE(7)/BDE(9)	EKLIP26
BDE(11)=BDE(8)/BDE(9)	EKLIP27
AM(7)=-A(24)*AC(1)+AM(10)*(A(21)*(AC(1)**2)-A(22))	EKLIP28
AM(9)=A(23)*(AC(1)**2)	EKLIP29
BDE(12)=AM(7)/AM(8)	EKLIP30
AM(19)=A(21)*AM(1)+A(22)	EKLIP31
AM(20)=A(21)*AM(2)	EKLIP32
AM(21)=A(24)*AC(2)	EKLIP33
AM(22)=A(24)*AC(3)	EKLIP34
BDE(13)=AM(19)*BDE(10)-AM(20)*BCE(11)-AM(21)	EKLIP35
BDE(14)=AM(19)*BDE(11)+AM(20)*BDE(10)-AM(22)	EKLIP36
BDE(15)=BDE(13)/A(23)	EKLIP37
BDE(16)=BDE(14)/A(23)	EKLIP38
AM(9)=AM(1)**2+AM(2)**2	EKLIP39
AM(10)=AM(1)*BDE(15)+AM(2)*BDE(16)	EKLIP40
AM(11)=AM(1)*BDE(16)-AM(2)*BDE(15)	EKLIP41
BDE(17)=AM(10)/AM(9)	EKLIP42
BDE(18)=AM(11)/AM(9)	EKLIP43
AM(13)=-BDE(19)*BDE(12)+BDE(20)*AM(30)-BDE(21)*AC(1)	EKLIP44
BDE(27)=BDE(19)*(AM(2)*BDE(18)-AM(1)*BDE(17))+BDE(20)*(AM(1)*BDE(1)	EKLIP45
101-AM(2)*BDE(11))-BDE(21)*AC(2)	EKLIP46
BDE(28)=-BDE(14)*(AM(2)*BDE(17)+AM(1)*BDE(18))+BDE(20)*(AM(1)*BDE(1)	EKLIP47
111)+AM(2)*BDE(10))-BDE(21)*AC(3)	EKLIP48
BDE(29)=AM(1)*BDE(27)+AM(2)*BDE(28)	EKLIP49
BDE(30)=AM(1)*BDE(28)-AM(2)*BDE(27)	EKLIP50
AM(14)=BDE(29)/AM(9)	EKLIP51
AM(15)=BDE(30)/AM(9)	EKLIP52
BDE(31)=EXP(P5)*COS(P6)	EKLIP53
BDE(32)=EXP(P5)*SIN(P6)	EKLIP54
BDE(33)=EXP(-P5)*COS(P6)	EKLIP55
BDE(34)=EXP(-P5)*SIN(P6)	EKLIP56
BDE(51)=AC(1)*(HA*BDE(12)-HB*AM(30))	EKLIP57
BDE(52)=-BDE(18)*HA+BDE(11)*HB	EKLIP58
BDE(53)=BDE(17)*HA-BDE(10)*HB	EKLIP59
BDE(54)=AC(3)*BDE(15)+AC(2)*BDE(13)	EKLIP60
BDE(55)=AC(2)*BDE(15)-AC(3)*BDE(13)	EKLIP61
AM(12)=BT(20)/CAPCB-BDE(21)*BT(9)	EKLIP62
	EKLIP63
AM(32)=(CA*NUA+CR*NUB)/R	EKLIP64
AM(33)=CA*HA/2	EKLIP65
AM(34)=CA*HB/2	EKLIP66
AM(35)=DA+(HB*HA)*CA/4	EKLIP67
AM(36)=DR+(HB*HB)*CA/4	EKLIP68
AM(37)=AM(35)+AM(34)	EKLIP69
AM(38)=AM(37)*NUA/R	EKLIP70
AM(39)=GAMA+GAMB	EKLIP71
AM(42)=DA+(CA/4.)*((HA**2)	EKLIP72
AM(43)=(HA*HB)*((CA/4.))	EKLIP73
BDE(81)=AM(33)*NUA/R	EKLIP74
BDE(82)=AM(34)*NUA/R	EKLIP75
	EKLIP76

AM(44)=AM(42)-AM(33)*BDE(19)	EKLIP77
AM(45)=AM(43)-AM(33)*BDE(20)	EKLIP78
AM(46)=BDE(81)-AM(33)*BDE(21)	EKLIP79
AM(47)=DR+(CA/4,1)*(HB**2)	EKLIP80
AM(48)=AM(43)-AM(34)*BDE(19)	EKLIP81
AM(49)=AM(47)-AM(34)*BDE(20)	EKLIP82
AM(50)=BDE(82)-AM(34)*BDE(21)	EKLIP83
BDE(83)=(H/12,1)*(BDE(22)*BDE(19))	EKLIP84
BDE(84)=(H/12,1)*(BDE(22)*BDE(20))	EKLIP85
BDE(85)=AM(39)+(H/12,1)*(BDE(22)*BDE(21))-BDE(23))	EKLIP86
DO 1 I=1,6	EKLIP87
V(I)=0.0	EKLIP88
DO 1 J=1,7	EKLIP89
EN(I,J)=0.0	EKLIP90
I CONTINUE	EKLIP91
EN(1,1)=AC(1)*(AM(44)*BDE(12)-AM(45)*AM(30))+AM(46)	EKLIP92
EN(1,2)=C.0	EKLIP93
EN(1,3)=AM(44)*(AC(2)*BDE(17)-AC(3)*BDE(18))+AM(45)*(AC(3)*BDE(11))	EKLIP94
1-AC(2)*BDE(10))+AM(46)	EKLIP95
EN(1,4)=AM(44)*(AC(3)*BDE(17)+AC(2)*BDE(18))-AM(45)*(AC(3)*BDE(10))	EKLIP96
1+AC(2)*BDE(11))	EKLIP97
EN(1,5)=EN(1,3)	EKLIP98
EN(1,6)=-EN(1,4)	EKLIP99
EN(2,1)=AC(1)*(BDE(12)*AM(48)-AM(30)*AM(49))+AM(50)	EKLIP
EN(2,2)=0.0	EKLIP
EN(2,3)=EN(2,1)	EKLIP
EN(2,4)=AM(46)*(-AC(3)*BDE(18)+AC(2)*BDE(17))+AM(49)*(AC(3)*BDE(11))	EKLIP
1-AC(2)*PDE(10))+AM(50)	EKLIP
EN(2,5)=AM(48)*(AC(3)*BDE(17)+AC(2)*BDE(18))-AM(49)*(AC(3)*BDE(10))	EKLIP
1+AC(2)*BDE(11))	EKLIP
EN(2,6)=EN(2,3)	EKLIP
EN(2,7)=-EN(2,4)	EKLIP
EN(3,1)=0.0	EKLIP
EN(3,2)=BDE(85)*AC(1)*BDE(12)*(BDE(83)*AC(1)**2-GAMA)-AM(30)*(BDE(84)*AC(1)**2-GAMA)	EKLIP
184)*AC(1)**2-GAMA)	EKLIP
EN(3,3)=BDE(85)*AC(2)-BDE(10)*(GAM8+BDE(84)*AM(1))+BDE(11)*(BDE(84)*AM(2))	EKLIP
1+AM(21)*BDE(17)*(GAMA+AM(1)*BDE(83))-BDE(18)*(BDE(83)*AM(2))	EKLIP
EN(3,4)=AC(3)*BDE(85)-BDE(11)*(GAM8+BDE(84)*AM(1))-BDE(10)*(BDE(84)*AM(2))	EKLIP
1+AM(21)*BDE(18)*(GAMA+AM(1)*BDE(83))+BDE(17)*(BDE(83)*AM(2))	EKLIP
EN(3,5)=-EN(3,3)	EKLIP
EN(3,6)=EN(3,4)	EKLIP
EN(4,1)=FN(1,1)*COS(P6)	EKLIP
EN(4,2)=EN(1,1)*SIN(P6)	EKLIP
EN(4,3)=EN(1,3)*BDE(31)-EN(1,4)*BDE(32)	EKLIP
EN(4,4)=EN(1,3)*BDE(32)+EN(1,4)*BDE(31)	EKLIP
EN(4,5)=EN(1,5)*BDE(33)-EN(1,6)*BDE(34)	EKLIP
EN(4,6)=EN(1,5)*BDE(34)+EN(1,6)*BDE(33)	EKLIP
EN(5,1)=EN(2,1)*COS(P6)	EKLIP
EN(5,2)=EN(2,1)*SIN(P6)	EKLIP
EN(5,3)=EN(2,3)*BDE(31)-EN(2,4)*BDE(32)	EKLIP
EN(5,4)=EN(2,3)*BDE(32)+EN(2,4)*BDE(31)	EKLIP
EN(5,5)=EN(2,5)*BDE(33)-EN(2,6)*BDE(34)	EKLIP
EN(5,6)=EN(2,5)*BDE(34)+EN(2,6)*BDE(33)	EKLIP
EN(6,1)=-EN(3,1)*SIN(P6)	EKLIP
EN(6,2)=EN(3,1)*COS(P6)	EKLIP
EN(6,3)=EN(3,3)*BDE(31)-EN(3,4)*BDE(32)	EKLIP
EN(6,4)=EN(3,3)*BDE(32)+EN(3,4)*BDE(31)	EKLIP
EN(6,5)=EN(3,5)*BDE(33)-EN(3,6)*BDE(34)	EKLIP
EN(6,6)=EN(3,5)*BDE(34)+EN(3,6)*BDE(33)	EKLIP

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EN(1,7)=AM(33)*BT(20)/CAPCB+BDE(81)*BT(9)+BT(18)-(HA/2.)*(BT(5)/(1)EKLP
1.-NUA))-BT(12)/(1.-NUA)+(HA/2.)*BT(21)+AM(46)*BDE(49) EKLP
EN(2,7)=AM(34)*BT(20)/CAPCR+BDE(82)*BT(9)+BT(19)-(HA/2.)*(BT(5)/(1)EKLP
1.-NUA))-PT(13)/(1.-NUB)+(HB/2.)*BT(22)+AM(50)*BDE(49) EKLP
EN(1,7)=-EN(1,7) EKLP
EN(2,7)=-EN(2,7) EKLP
EN(3,7)=LOAD1 EKLP
EN(4,7)=EN(1,7) EKLP
EN(5,7)=EN(2,7) EKLP
EN(6,7)=EN(3,7) EKLP

WRITE (9,3) (BDE(I),I=1,100) EKLP W
WRITE (9,5) EKLP W
WRITE (9,4) (AM(J),J=1,50) EKLP W
WRITE (9,12) EKLP W
WRITE (9,6) ((EN(I,J),J=1,7),I=1,6) EKLP W
CALL FINGLE (EN,V) EKLP
WRITE (9,7) (I,V(I)),I=1,6) EKLP W

AM(12)=BT(20)/CAPCB-BDE(21)*BDE(49)
PP1=AC(1)*ELL/2. EKLP
PP2=AC(2)*ELL/2. EKLP
PP3=AC(3)*ELL/2. EKLP
CALL CHECK2 (V,AC,PP1,PP2,PP3) EKLP
WRITE (9,7) (I,V(I)),I=1,6) EKLP W

AL(1)=AM(30)*V(1) EKLP
AL(2)=-AM(31)*V(2) EKLP
AL(3)=-BDE(10)*V(3)-BDE(11)*V(4) EKLP
AL(4)=BDE(11)*V(3)-BDE(10)*V(4) EKLP
AL(5)=BDE(11)*V(5)-BDE(11)*V(6) EKLP
AL(6)=BDE(11)*V(5)+BDE(10)*V(6) EKLP
AL(7)=-BDE(12)*V(1) EKLP
AL(8)=BDE(12)*V(2) EKLP
AL(9)=BDE(12)*V(1) EKLP
AL(10)=-BDE(12)*V(2) EKLP
AL(11)=-BDE(17)*V(5)+BDE(18)*V(6) EKLP
AL(12)=-BDE(18)*V(5)-BDE(17)*V(6) EKLP

WRITE (9,8) EKLP W
WRITE (9,9) (AL(I),I=1,12) EKLP W

DO 2 KF=1,11 EKLP
X=DELPH(KF) EKLP
IF (X.GT.ELL) GO TO 2 EKLP
P1=AC(1)*X EKLP
P2=AC(2)*X EKLP
P3=AC(3)*X EKLP

CALL DIFF (P1,P2,P3) EKLP
ECLIP
ECLIP
ECLIP

WW=V(1)*COS(P1)+V(2)*SIN(P1)*EXP(P2)*(V(3)*COS(P3)+V(4)*SIN(P3))+E EKLP
XP(-P2)*(V(5)*COS(P3)+V(6)*SIN(P3))+BDE(49) EKLP
DWW=BDE(18)*V(2)*HDE(19)*V(3)*BDE(12)*V(4)*BDE(13)*V(5)*B EKLP

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10E(141)*V(6)*BDE(151) EKLI P
  DOHW=V(1)*BCE(182)*V(2)*RDE(192)*V(3)*BDE(122)*V(4)*BDE(132)*V(5)*EKLIP
  1BDE(142)*V(6)*BDE(152) EKLI P
    WB=AL(1)*COS(P1)*AL(2)*SIN(P1)*EXP(P2)*(AL(3)*COS(P3)+AL(4)*SIN(P3))EKLI P
  11)*EXP(-P2)*(AL(5)*COS(P3)+AL(6)*SIN(P3)) EKLI P
  DWB=AL(1)*BUE(181)*AL(2)*BDE(191)*AL(3)*RDE(121)*AL(4)*BDE(131)*ALEKLI P
  1(5)*BDE(141)*AL(6)*RDE(151) EKLI P
  DDWB=AL(1)*BDE(182)*AL(2)*BUE(192)*AL(3)*BDE(122)*AL(4)*BDE(132)*AEKLI P
  1L(5)*RDE(142)*AL(6)*BDE(152) EKLI P
  DDDWB=AL(1)*BUE(143)*AL(2)*BDE(193)*AL(3)*BDE(123)*AL(4)*BDE(133)*EKLIP
  1AL(5)*BDE(143)*AL(6)*RDE(153) EKLI P
    WA=AL(7)*COS(P1)*AL(8)*SIN(P1)*EXP(P2)*(AL(9)*COS(P3)+AL(10)*SIN(P3))EKLI P
  13)*EXP(-P2)*(AL(11)*COS(P3)+AL(12)*SIN(P3)) EKLI P
  DWA=AL(7)*BDE(141)*AL(8)*BJE(191)*AL(9)*RDE(121)*AL(10)*BDE(131)*AEKLI P
  1L(11)*RDE(141)*AL(12)*RDE(151) EKLI P
  DDWA=AL(7)*BUE(1H2)*AL(8)*BDE(192)*AL(9)*BDE(122)*AL(10)*BUE(132)*EKLIP
  1AL(11)*BDE(142)*AL(12)*RDE(152) EKLI P
  DDDWA=AL(7)*BDE(1H3)*AL(8)*BUE(193)*AL(9)*RDE(123)*AL(10)*BDE(133)*EKLIP
  1*AL(11)*BDE(143)*AL(12)*BDE(153) EKLI P
  EKLI P
  FNTWH=(1./AC(1)*((V(1)*SIN(P1)-V(2)*COS(P1))+((EXP(P2)/AM(24))+((V(3)*EKLIP
  1)*((AC(2)*COS(P3)+AC(3)*SIN(P3))+V(4)*((AC(2)*SIN(P3)-AC(3)*COS(P3))))EKLI P
  21+((EXP(-P2)/AM(24))+(V(5)*((-AC(2)*COS(P3)+AC(3)*SIN(P3))-V(6)*((AC(EKLIP
  32)*SIN(P3)+AC(3)*COS(P3))))+HUE(49)*X EKLI P
  IF (KF,EQ,1) CALL UC OFF (FNTWH,C04ST,0) EKLI P
  MU=-BDE(19)*WA-RDE(20)*WB-BDE(21)*F'ITWH+(RT(20)/CAPCB)*X*CONST EKLI P
  DWU=-BDE(19)*PUA-RDE(20)*DWA-BDE(21)*W4+BT(2G)/CAPCB EKLI P
  DDMU=-BDE(19)*DDWA-RDE(20)*DDWB-BDE(21)*DWU EKLI P
  DDDMU=-RDE(14)*DDWA-RDE(20)*DDWB-RDE(21)*DDWU EKLI P
  TAU=LOAD1-RDE(22)*DDWU-PDE(23)*DWU EKLI P
  DTAU=-BDE(22)*DDWU-HDE(23)*DWU EKLI P
  WPJ=LOAD2-BDE(24)*DDWU-RDE(25)*DTAU+RDE(23)*DWU+RDE(26)*WW EKLI P
  WRITE (9,10) WB,DA8,DDWB,DDDWB,WA,DWA,DUWA,DDWUA EKLI P
  WUA=HU*HA*W4/2.+MB*WB/2. EKLI P
  DWUA=DUU*HA*DWA/2.+MB*DWU/2. EKLI P
  DDWUA=DDWU*HA*DDWA/2.+MB*DDWB/2. EKLI P
  WRITE (9,11) WU,DWU,DDWU,WUA,DWUA,DDWUA EKLI P
  CALL RSLT (X) EKLI P
  CALL PRINT (X) EKLI P
  2 CONTINUE EKLI P
  RETURN EKLI P
  EKLI P
  3 FORMAT (1X,9H BDE ARE /10E12.5) EKLI P
  4 FORMAT (10E12.5) EKLI P
  5 FORMAT (20H AM(J) ARE ) EKLI P
  6 FORMAT (1X,7E11.4) EKLI P
  7 FORMAT (4(1X,2HV(11,2H)=E12.5)/2(1X,2HV(11,2H)=E12.5)) EKLI P
  8 FORMAT (1X,20H AL(I) ARE ) EKLI P
  9 FORMAT (6E12.5) EKLI P
  10 FORMAT (/1X,25H BETA(I) DERIVATIVES /E12.5,1X,E12.5,1X,E12.5,1EKLIP
  1X,E12.5/1X,E12.5,20H BETA(I) DERIVATIVES/1X,E12.5,1X,E12.5,1X,E12.5,1X,EKLI P
  21E.5//) EKLI P
  11 FORMAT (1X,20H U(I) DERIVATIVES /1X,E12.5,1X,E12.5,1X,E12.5/1X,2EKLIP
  1OMU(I) DERIVATIVES /1X,E12.5,1X,E12.5,1X,E12.5//) EKLI P
  12 FORMAT (21H ENII,J1BY ROWS ARE ) EKLI P
  END EKLI P
  SUBROUTINE EKLI P
  DIMENSION JIP(111) EKLI P
  COMMON DLTEMP EKLI P
  1 EKLI P
  2 EKLI P
  3 EKLI P

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COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,VURC,HA,HB,R,G3A,G3B,CA,CB,EKLIP 4
1DA,0B,H,CACH,CAPCB,NUNU,GAMA,GAMB,K(40),A(55),G(7),B(30),F4FOA,FNEKLI P 5
2TOB,RDE(600),LOAD1,AM(53),LOAD2,EN(6,7),FNXA,FNXH,FMXA,FMXB,FNX,FMEKLI P 6
3X,FYOA,FYOB,AC(13),KLIP,ELL,DELPH(20),V(6),AL(18),HH,DHH,DWH,W,A,DWEKLI P 7
4B,DDWB,DDDWB,W,A,DWA,WU,DHU,DWU,DUWU,TAU,DTAU,WPJ,WHA,DWWA,WUA,DWEKLI P 8
SUA
REAL NUA,NUAC,NUB,NIBC,NUNU,K,LOAD1,LOAD2
BDE(1)=A(44)*AC(1)**3)+A(45)*AC(1)
BDE(2)=A(41)*AC(1)**4)+A(42)*AC(1)**2)+A(43)
BDE(3)=BDE(1)/BDE(2)
BDE(4)=A(44)*AC(2)**3)-A(45)*AC(2)
BDE(5)=A(41)*AC(2)**4)-A(42)*AC(2)**2)+A(43)
BDE(6)=BDE(4)/BDE(5)
BDE(7)=A(44)*AC(3)**3)-A(45)*AC(3)
BDE(8)=A(41)*AC(3)**4)-A(42)*AC(3)**2)+A(43)
BDE(9)=BDE(7)/BDE(8)
BDE(10)=BDE(3)*(A(21)*(AC(1)**2)+A(22))-A(24)*AC(1)
BDE(11)=A(23)*(AC(1)**2)
BDE(12)=BDE(10)/BDE(11)
BDE(13)=BDE(6)*(A(21)*(AC(2)**2)-A(22))-A(24)*AC(2)
BDE(14)=A(23)*(AC(2)**2)
BDE(15)=BDE(13)/BDE(14)
BDE(16)=BDE(9)*(A(21)*(AC(3)**2)-A(22))-A(24)*AC(3)
BDE(17)=A(23)*(AC(3)**2)
BDE(18)=BDE(16)/BDE(17)
P4=AC(1)*ELL
P5=AC(2)*ELL
P6=AC(3)*ELL
EN(1,1)=(AM(44)*BDE(12)-AM(45)*BDE(3))*AC(1)+AM(46)
EN(1,2)=EN(1,1)
EN(1,3)=(AM(44)*BDE(15)-AM(45)*BDE(6))*AC(2)+AM(46)
EN(1,4)=0.0
EN(1,5)=(AM(44)*BDE(18)-AM(45)*BDE(9))*AC(3)+AM(46)
EN(1,6)=0.0
EN(4,1)=EN(1,1)*EXP(P4)
EN(4,2)=EN(1,2)*EXP(-P4)
EN(4,3)=EN(1,3)*COS(P5)
EN(4,4)=EN(1,3)*SIN(P5)
EN(4,5)=EN(1,5)*COS(P6)
EN(4,6)=EN(1,5)*SIN(P6)
EN(2,1)=(AM(48)*BDE(12)-AM(49)*BDE(3))*AC(1)+AM(50)
EN(2,2)=EN(2,1)
EN(2,3)=(AM(48)*BDE(15)-AM(49)*BDE(6))*AC(2)+AM(50)
EN(2,4)=0.0
EN(2,5)=(AM(48)*BDE(18)-AM(49)*BDE(9))*AC(3)+AM(50)
EN(2,6)=0.0
EN(5,1)=EN(2,1)*EXP(P4)
EN(5,2)=EN(2,2)*EXP(-P4)
EN(5,3)=EN(2,3)*COS(P5)
EN(5,4)=EN(2,3)*SIN(P5)
EN(5,5)=EN(2,5)*COS(P6)
EN(5,6)=EN(2,5)*SIN(P6)

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EN(3,1)=BDE(85)*AC(1)-BDE(3)*(BUE(84)*(AC(1)**2)+GAMB)+BDE(12)*(BUEKLI P64
1E(83)*(AC(1)**2)+GAMA) EKLIP65
EN(3,2)=EN(3,1) EKLIP66
EN(3,3)=0.0 EKLIP67
EN(3,4)=BUE(1H5)*AC(2)-BUE(6)*BDE(84)*(AC(2)**2)+GAMB)+BDE(15)*(BUEKLI P68
1E(83)*(AC(2)**2)+GAMA) EKLIP69
EN(3,5)=0.0 EKLIP70
EN(3,6)=BDE(85)*AC(3)+BDE(18)*(BDE(83)*(AC(3)**2)-GAM4)-BDE(9)*(BUEKLI P71
1E(84)*(AC(3)**2)-GAMB) EKLIP72
EN(6,1)=EN(3,1)*EXP(P4) EKLIP73
EN(6,2)=EN(3,2)*EXP(-P4) EKLIP74
EN(6,3)=-EN(3,4)*SIN(P5) EKLIP75
EN(6,4)=EN(3,4)*COS(P5) EKLIP76
EN(6,5)=-EN(3,6)*SIN(P6) EKLIP77
EN(6,6)=EN(3,6)*COS(P6) EKLIP78
EKLIP79
EKLIP80
WRITE (9,2) (BDE(I),I=1,20) EKLIP81W
WRITE (9,3) EKLIP82W
WRITE (9,4) ((EN(I,J),J=1,7),I=1,6) EKLIP83W
CALL BINGO (EN,V) EKLIP84
WRITE (9,5) ((V,I),I=1,6) EKLIP85W
AM(12)=BT(2G)/CAPCB-BDE(21)*BDE(49) EKLIP86
PP1=AC(1)*ELL/2. EKLIP87
PP2=AC(2)*ELL/2. EKLIP88
PP3=AC(3)*ELL/2. EKLIP89
CALL CHECK4 (V,AC,PP1,PP2,PP3) EKLIP90
WRITE (9,5) ((V,I),I=1,6) EKLIP91W
EKLIP92
AL(1)=-BDE(1)*V(1) EKLIP93
AL(2)=BUE(3)*V(2) EKLIP94
AL(3)=BUE(6)*V(3) EKLIP95
AL(4)=-BDE(6)*V(4) EKLIP96
AL(5)=BUE(9)*V(5) EKLIP97
AL(6)=-BDE(9)*V(5) EKLIP98
EKLIP99
AL(7)=BDE(12)*V(1) EKLIP
AL(8)=-BDE(12)*V(2) EKLIP
AL(9)=-BDE(15)*V(3) EKLIP
AL(10)=BDE(15)*V(4) EKLIP
AL(11)=-BDE(16)*V(6) EKLIP
AL(12)=BUE(18)*V(5) EKLIP
WRITE (9,6) EKLIP W
WRITE (9,7) (AL(I),I=1,12) EKLIP W
EKLIP
DO I KF=1,11 EKLIP
X=DELPH(KF) EKLIP
P1=AC(1)*X EKLIP
P2=AC(2)*X EKLIP
P3=AC(3)*X EKLIP
WW=V(1)*EXP(P1)+V(2)*EXP(-P1)+V(3)*COS(P2)+V(4)*SIN(P2)+V(5)*COS(P6KLI P
13)+V(6)*SIN(P3)+BDE(49) EKLIP
DWW=V(1)*BDE(101)+V(2)*BDE(111)+V(3)*BDE(201)+V(4)*BDE(211)+V(5)*BDE(221)
+BDE(231) EKLIP
DDWW=V(1)*BDE(102)+V(2)*BDE(112)+V(3)*BDE(202)+V(4)*BDE(212)+V(5)*BDE(222)
+BDE(232) EKLIP
WB=AL(1)*EXP(P1)+AL(2)*EXP(-P1)+AL(3)*COS(P2)+AL(4)*SIN(P2)+AL(5)*EKLIP
+ICOS(P3)+AL(6)*SIN(P3) EKLIP
DWB=AL(1)*BDE(101)+AL(2)*BDE(111)+AL(3)*BDE(201)+AL(4)*BDE(211)+ALEKLI P

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1(S)*BDE(221)+AL(6)*BDF(231) EKLIP
DDWB=AL(1)*BDE(102)+AL(2)*BDE(112)+AL(3)*BDE(202)+AL(4)*BDE(212)+AEKLIP
1(L5)*BDE(222)+AL(6)*BDE(232) EKLIP
DDDWB=AL(1)*BDE(103)+AL(2)*BDE(113)+AL(3)*BDE(203)+AL(4)*BDE(213)+EKLIP
1AL(5)*BDE(223)+AL(6)*BDE(233) EKLIP
EKLIP
WA=AL(7)*EXP(P1)+AL(8)*EXP(-P1)+AL(9)*COS(P2)+AL(10)*SIN(P2)+AL(11)EKLIP
1)*COS(P3)+AL(12)*SIN(P3) EKLIP
DWA=AL(7)*BDE(101)+AL(8)*BDE(111)+AL(9)*BDE(201)+AL(10)*BDE(211)+AEKLIP
1L(11)*BDE(221)+AL(12)*BDE(231) EKLIP
DDWA=AL(7)*BDE(102)+AL(8)*BDE(112)+AL(9)*BDE(202)+AL(10)*BDE(212)+EKLIP
1AL(11)*BDE(222)+AL(12)*BDE(232) EKLIP
DDDWA=AL(7)*BDE(103)+AL(8)*BDE(113)+AL(9)*BDE(203)+AL(10)*BDE(213)EKLIP
1+AL(11)*BDE(223)+AL(12)*BDE(233) EKLIP
EKLIP
FNTWW=(V(1)*EXP(P1)-V(2)*EXPI-P1))/AC(1)+(V(3)*COS(P2)-V(4)*SIN(P2)EKLIP
1)/AC(2)+(V(5)*SIN(P3)-V(6)*COS(P3))/AC(3)+BDE(49)*X EKLIP
IF (KF.EQ.1) CALL UCOFF (FNTWW,CONST,0) EKLIP
WU=-BDE(19)*WA-BDE(20)*WB-BDE(21)*FNTWW+BT(20)/CAPCR1*X+CONST EKLIP
DWU=-BDE(19)*WU-BDE(20)*DWE-BDE(21)*WW+BT(20)/CAPCH EKLIP
DDWU=-BDE(19)*DDWA-HDE(20)*DJWB-BDF(21)*UWW EKLIP
DDDWU=-BDE(19)*DDDWA-BDE(20)*DDWB-BDE(21)*DDWW EKLIP
TAU=LOAD1-BDE(22)*DDWU-BDE(23)*LWW EKLIP
DTAU=-BDE(22)*DDWU-BDE(23)*UWW EKLIP
WPJ=LCAD2-BDE(24)*DDWB-BDE(25)*DTAU+BDE(23)*DWU+BDE(26)*WW EKLIP
CALL RSLT (X) EKLIP
CALL PRINT (X) EKLIP
IF (X.EQ.ABS(ELL/2)) CALL VINSON (X,EA,EB,HA,HB,G3A,G3B,FNXB) EKLIP
1 CONTINUE EKLIP
RETURN EKLIP
EKLIP
2 FORMAT (1X,9HBDE ARE / (10E12.5)) EKLIP
3 FORMAT (1X,20H EN(I,J) ARE ) EKLIP
4 FORMAT (1X,7E11.4) EKLIP
5 FORMAT (4(1X,2HV(11,2H)=E12.5)/2(1X,2HV(11,2H)=E12.5)) EKLIP
6 FORMAT (1X,20H AL(I) ARE ) EKLIP
7 FORMAT (6E12.5) EKLIP
END EKLIP -
SUBROUTINE EKLIPS EKLI P 1
DIMENSION JIP(11) EKLIP 2
COMMON DLTEMP EKLIP 3
COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,NUBC,HA,MR,R,G3A,G3B,CA,CR,EKLIP 4
IDA,DB,M,CACB,CAPCH,NUHU,GAMA,GAMR,K140),A(55),G(7),BT(30),FNTGA,FNEKLIP 5
2TOB,BDE(1600),LOAD1,AM(50),LJAD2,EN(6,7),FNKA,FNKB,FXA,FMXB,FYA,FMEXB 6
3X,FNDA,FNCR,AC(3),KLIP,ELL,DELPH(20),V(6),AL(1H),WW,DWW,CWW,HB,UNEKLI P 7
4B,DDWB,DDWU,WA,DWA,HU,UWW,DDWU,TAU,DTAU,WPJ,WWA,UWWA,WUA,UNERKLIP 8
5UA EKLIP 9
REAC NUA,NUAC,NUB,NUBC,NUHU,K,LJAD1,LOAD2 EKLIP 10
BDE(1)=A(44)*IAC(1)*03+A(45)*AC(1) EKLIP 11
BDE(2)=A(41)*IAC(1)*04+A(42)*IAC(2)*02+A(43) EKLIP 12
BDE(3)=BDE(1)/BDE(2) EKLIP 13
EKLIP 14
BDE(4)=A(44)*IAC(2)*03+A(45)*AC(2) EKLIP 15
BDE(5)=A(41)*IAC(2)*04+A(42)*IAC(2)*02+A(43) EKLIP 16
BDE(6)=BDE(4)/BDE(5) EKLIP 17
EKLIP 18
BDE(7)=A(44)*IAC(3)*03-A(45)*AC(3) EKLIP 19
BDE(8)=A(41)*IAC(3)*04-A(42)*IAC(3)*02+A(43) EKLIP 20
BDE(9)=BDE(7)/BDE(8) EKLIP 21
EKLIP 22

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BDE(10)=BDE(3)*(A(21)*(AC(1)**2)+A(22))-A(24)*AC(1) EKLP23
BDE(11)=A(23)*(AC(1)**2) EKLP24
BDE(12)=BDE(10)/BDE(11) EKLP25
BDE(13)=BUE(6)*(A(21)*(AC(2)**2)+A(22))-A(24)*AC(2) EKLP26
BDE(14)=A(23)*(AC(2)**2) EKLP27
BDE(15)=BDE(13)/BDE(14) EKLP28
BDE(16)=BDE(4)*(A(21)*(AC(3)**2)+A(22))-A(24)*AC(3) EKLP29
BDE(17)=A(23)*(AC(3)**2) EKLP30
BDE(18)=BDE(16)/BDE(17) EKLP31
P4=AC(1)*ELL EKLP32
P5=AC(2)*ELL EKLP33
P6=AC(3)*ELL EKLP34
EN(1,1)=(AM(44)*BDE(12)-AM(45)*BDE(3))*AC(1)+AM(46) EKLP35
EN(1,2)=EN(1,1) EKLP36
EN(1,3)=(AM(44)*BDE(15)-AM(45)*BDE(6))*AC(2)+AM(46) EKLP37
EN(1,4)=EN(1,3) EKLP38
EN(1,5)=(AM(44)*BDE(18)-AM(45)*BDE(9))*AC(3)+AM(46) EKLP39
EN(1,6)=0.0 EKLP40
EN(4,1)=EN(1,1)*EXP(P6) EKLP41
EN(4,2)=EN(1,2)*EXP(-P6) EKLP42
EN(4,3)=EN(1,3)*EXP(P5) EKLP43
EN(4,4)=EN(1,4)*EXP(-P5) EKLP44
EN(4,5)=EN(1,5)*COS(P6) EKLP45
EN(4,6)=EN(1,6)*SIN(P6) EKLP46
EN(2,1)=(AM(48)*BDE(12)-AM(49)*BDE(3))*AC(1)+AM(50) EKLP47
EN(2,2)=EN(2,1) EKLP48
EN(2,3)=(AM(48)*BDE(15)-AM(49)*BDE(6))*AC(2)+AM(50) EKLP49
EN(2,4)=EN(2,3) EKLP50
EN(2,5)=(AM(48)*BDE(18)-AM(49)*BDE(9))*AC(3)+AM(50) EKLP51
EN(2,6)=0.0 EKLP52
EN(5,1)=EN(2,1)*EXP(P4) EKLP53
EN(5,2)=EN(2,2)*EXP(-P4) EKLP54
EN(5,3)=EN(2,3)*EXP(P5) EKLP55
EN(5,4)=EN(2,4)*EXP(-P5) EKLP56
EN(5,5)=EN(2,5)*COS(P6) EKLP57
EN(5,6)=EN(2,6)*SIN(P6) EKLP58
EN(3,1)=BDE(85)*AC(1)-BDE(3)*(BDE(84)*(AC(1)**2)+GAMB)+BUE(12)*(BDE EKLP59
IE(83)*(AC(1)**2)+GAMA) EKLP60
EN(3,2)=EN(3,1) EKLP61
EN(3,3)=BDE(85)*AC(2)-BDE(6)*(BDE(84)*(AC(2)**2)+GAMB)+BUE(15)*(BDE EKLP62
IE(83)*(AC(2)**2)+GAMA) EKLP63
EN(3,4)=-EN(3,3) EKLP64
EN(3,5)=0.0 EKLP65
EN(3,6)=BDE(85)*AC(3)+ADE(18)*(BDE(83)*(AC(3)**2)-GAMA)-BDE(9)*(PDE EKLP66
IE(84)*(AC(3)**2)-GAMB) EKLP67
EN(6,1)=EN(3,1)*EXP(P6) EKLP68
EN(6,2)=EN(3,2)*EXP(-P6) EKLP69
EN(6,3)=EN(3,3)*EXP(P5) EKLP70
EN(6,4)=EN(3,4)*EXP(-P5) EKLP71
EN(6,5)=-EN(3,5)*SIN(P6) EKLP72
EN(6,6)=EN(3,6)*COS(P6) EKLP73
WRITE(9,2) (BDE(11),I=1,20) EKLP74
WRITE(9,3) EKLP75

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      WRITE (9,4) ((EN(I,J),J=1,7),I=1,6)          EKLIP83W
      CALL BINGO (EN,V)                           EKLIP84
      WRITE (9,5) ((V(I)),I=1,6)                  EKLIP85W
      AN(12)=BT(20)/CAPCB-BDE(21)*BDE(49)        EKLIP86
      PP1=AC(1)*ELL/2.                            EKLIP87
      PP2=AC(2)*ELL/2.                            EKLIP88
      PP3=AC(3)*ELL/2.                            EKLIP89
      CALL CHECKS (V,AC,PP1,PP2,PP3)              EKLIP90
      WRITE (9,5) ((V(I)),I=1,6)                  EKLIP91W
      AL(1)=-BDE(3)*V(1)                         EKLIP92
      AL(2)=BDE(3)*V(2)                          EKLIP93
      AL(3)=-BDE(6)*V(3)                         EKLIP94
      AL(4)=BDE(6)*V(4)                          EKLIP95
      AL(5)=BDE(9)*V(5)                          EKLIP96
      AL(6)=-BDE(9)*V(6)                         EKLIP97
      AL(7)=BDE(12)*V(1)                         EKLIP98
      AL(8)=-BDE(12)*V(2)                        EKLIP99
      AL(9)=BDE(15)*V(3)                         EKLIP
      AL(10)=-BDE(15)*V(4)                        EKLIP
      AL(11)=-BDE(16)*V(6)                        EKLIP
      AL(12)=BDE(19)*V(5)                         EKLIP
      WRITE (9,6)                                 EKLIP W
      WRITE (9,7) (AL(I),I=1,12)                  EKLIP W
      DO 1 KF=1,11
      X=DELP(KF)
      P1=AC(1)*X
      P2=AC(2)*X
      P3=AC(3)*X
      MW=V(1)*EXP(P1)+V(2)*EXP(-P1)+V(3)*EXP(P2)+V(4)*EXP(-P2)+V(5)*COS(EKLIP
      IP3)+V(6)*SIN(P3)+BDE(49)                  EKLIP
      DMW=V(1)*BDE(101)+V(2)*BDE(111)+V(3)*BDE(161)+V(4)*BDE(171)+V(5)*BEKLIP
      BDE(221)+V(6)*BDE(231)                      EKLIP
      DDMW=V(1)*BDE(102)+V(2)*BDE(112)+V(3)*BDE(162)+V(4)*BDE(172)+V(5)*EKLIP
      BBDE(222)+V(6)*BDE(232)                      EKLIP
      MB=AL(1)*EXP(P1)+AL(2)*EXP(-P1)+AL(3)*EXP(P2)+AL(4)*EXP(-P2)+AL(5)*EKLIP
      +COS(P3)+AL(6)*SIN(P3)                       EKLIP
      DMA=AL(1)*BDE(101)+AL(2)*BDE(111)+AL(3)*BDE(161)+AL(4)*BDE(171)+AL(EKLIP
      151)*BDE(221)+AL(6)*BDE(231)                 EKLIP
      DDMB=AL(1)*BDE(102)+AL(2)*BDE(112)+AL(3)*BDE(162)+AL(4)*BDE(172)+AL(5)*EKLIP
      1151)*BDE(222)+AL(6)*BDE(232)                 EKLIP
      DDWB=AL(1)*BDE(103)+AL(2)*BDE(113)+AL(3)*BDE(163)+AL(4)*BDE(173)+AL(5)*HUE(173)+EKLIP
      1AL(6)*BDE(223)+AL(6)*BDE(233)                 EKLIP
      MA=AL(7)*EXP(P1)+AL(8)*EXP(-P1)+AL(9)*EXP(P2)+AL(10)*EXP(-P2)+AL(11)*EKLIP
      121)*COS(P3)+AL(12)*SIN(P3)                  EKLIP
      DMA=AL(7)*BDE(101)+AL(8)*BDE(111)+AL(9)*BDE(161)+AL(10)*BDE(171)+AL(EKLIP
      1111)*BDE(221)+AL(112)*BDE(231)                 EKLIP
      DDMWA=AL(7)*BDE(102)+AL(8)*BDE(112)+AL(9)*BDE(162)+AL(10)*BDE(172)+AL(11)*EKLIP
      1AL(111)*BDE(222)+AL(112)*BDE(232)                 EKLIP
      DDDWA=AL(7)*BDE(103)+AL(8)*BDE(113)+AL(9)*BDE(163)+AL(10)*BDE(173)+AL(11)*EKLIP
      1AL(111)*BDE(223)+AL(112)*BDE(233)                 EKLIP
      FNTHW=(V(1)*EXP(P1)-V(2)*EXP(-P1))/AC(1)+(V(3)*EXP(P2)-V(4)*EXP(-P2))/AC(2)
      +(V(5)*SIN(P3)-V(6)*COS(P3))/AC(3)+BDE(49)*X          EKLIP
      IF (X.EU.-ELL/2.) CALL UCDF (FNTHW,CONST)             EKLIP
      MU=-BDE(19)*WA-BDE(20)*WB-BDE(21)*FNTHW+(BT(20)/CAPCB)*X+CUYST   EKLIP

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DWU=BDE(19)*DWA-BDE(20)*DWA-BDE(21)*WW+BT(20)/CAPCB EKLIP
DDWU-BDE(19)*DDWA-BDE(20)*DDWB-BDE(21)*DWW EKLIP
DDDWU-BDE(19)*DDDWU-BDE(20)*DDWB-BDE(21)*DDWW EKLIP
TAU=LOAD1-BDE(22)*DDWU-BDE(23)*DWW EKLIP
DTAU=BDE(22)*DDDWU-BDE(23)*DDWW EKLIP
WPJ=LOAD2-BDE(24)*DDDWU-BDE(25)*DTAU+BDE(23)*DWU+BDE(26)*WW EKLIP
CALL RSLT (X) EKLIP
CALL PRINT (X) EKLIP
QA=(WA/12.1)*TAU+GAMA*(WA+DWW) EKLIP
QB=(WB/12.1)*TAU+GAMA*(WB+DWW) EKLIP
FO=QA+QB EKLIP
WRITE (9,8) QA,QB,FO EKLIP W
IF (X.EQ.ABS(ELL/2.1)) CALL VENSON (X,EA,EB,WA,WB,G3A,G3B,FNXB)
2 CONTINUE EKLIP
RETURN EKLIP

2 FORMAT (1X,9H8DE ARE /((10E12.5))
3 FORMAT (1X,20H EN(1,1) ARE . ) EKLIP
4 FORMAT (1X,7E11.4) EKLIP
5 FORMAT (4(1X,2HV(1L,2H)=E12.5)/2(1X,2HV(1L,2H)=E12.5)) EKLIP
6 FORMAT (1X,20H AL(1) ARE ) EKLIP
7 FORMAT (6E12.5) EKLIP
8 FORMAT (1X,3HQA=E12.5,2X,3HQB=E12.5,2X,5HQBAR=E12.5/) EKLIP
END EKLIP -
SUBROUTINE EKLIP6 00000 1
DIMENSION JIP(11) EKLIP 2
COMMON DLTEMP EKLIP 3
COMMON TEMP,EA,EAC,MUA,NUAC,NUAC,EBC,NUBC,NUBC,WA,WB,R,G3A,G3B,CA,CB,EKLIP 4
BDA,DR,M,CACB,CAPCB,YNUU,GAMA,GAMC,K(4U),A(55),G(7),RT(3L),FYTC,A,FMEKIP 5
ZT0B,PDE(600),LOAD1,AM(50),LOAD2,EN(6,7),FNXA,FNXC,FMXB,FNX,FMEKIP 6
3X,FYCA,FNQB,AC(3),KLIP,ELL,DELPH(201),V(6),AL(18),WW,DWW,CONW,WB,DWEKIP 7
4B,DDWB,DDDWU,WA,DWA,WU,DWU,DDWU,TAU,DTAU,WPJ,WHA,DHWA,MUA,DWEKIP 8
SUA EKLIP 9
REAL NUA,NUAC,NUR,NUBC,NUBC,K,LOAD1,LOAD2 EKLIP10
CASE OF THREE REAL ROOTS,UNEQUAL,POSITIVE EKLIP11
BDE(1)=A(41)*(AC(1)**3)+A(45)*AC(1) EKLIP12
BDE(2)=A(41)*(AC(1)**4)+A(42)*(AC(1)**2)+A(43) EKLIP13
BDE(3)=BDE(1)/BDE(2) EKLIP14
BDE(4)=A(41)*(AC(2)**3)+A(45)*AC(2) EKLIP15
BDE(5)=A(41)*(AC(2)**4)+A(42)*(AC(2)**2)+A(43) EKLIP16
BDE(6)=BDE(4)/BDE(5) EKLIP17
BDE(7)=A(41)*(AC(3)**3)+A(45)*AC(3) EKLIP18
BDE(8)=A(41)*(AC(3)**4)+A(42)*(AC(3)**2)+A(43) EKLIP19
BDE(9)=BDE(7)/BDE(8) EKLIP20
BDE(10)=BDE(3)*(AC(1)**2)+A(22))-A(24)*AC(1) EKLIP21
BDE(11)=A(23)*(AC(1)**2) EKLIP22
BDE(12)=BDE(10)/BDE(11) EKLIP23
BDE(13)=BDE(6)*(AC(2)**2)+A(22))-A(24)*AC(2) EKLIP24
BDE(14)=A(23)*(AC(2)**2) EKLIP25
BDE(15)=BDE(13)/BDE(14) EKLIP26
BDE(16)=BDE(9)*(AC(2)**2)+A(22))-A(24)*AC(3) EKLIP27
BDE(17)=A(23)*(AC(3)**2) EKLIP28
BDE(18)=BDE(16)/BDE(17) EKLIP29
EKLIP30
EKLIP31
EKLIP32
EKLIP33
EKLIP34
EKLIP35
EKLIP36

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P4=AC(1)*ELL	EKLIP37
P5=AC(2)*ELL	EKLIP38
P6=AC(3)*ELL	EKLIP39
EN(1,1)=(AM(44)*BDE(12)-AM(45)*BDE(3))*AC(1)+AM(46)	EKLIP40
EN(1,2)=EN(1,1)	EKLIP41
EN(1,3)=(AM(44)*BDE(15)-AM(45)*BDE(6))*AC(2)+AM(46)	EKLIP42
EN(1,4)=EN(1,3)	EKLIP43
EN(1,5)=(AM(44)*BDE(18)-AM(45)*BDE(9))*AC(3)+AM(46)	EKLIP44
EN(1,6)=EN(1,5)	EKLIP45
EN(4,1)=EN(1,1)*EXP(P4)	EKLIP46
EN(4,2)=EN(1,2)*EXP(-P4)	EKLIP47
EN(4,3)=EN(1,3)*EXP(P5)	EKLIP48
EN(4,4)=EN(1,4)*EXP(-P5)	EKLIP49
EN(4,5)=EN(1,5)*EXP(P6)	EKLIP50
EN(4,6)=EN(1,6)*EXP(-P6)	EKLIP51
EN(2,1)=(AM(48)*BDE(12)-AM(49)*BDE(3))*AC(1)+AM(50)	EKLIP52
EN(2,2)=EN(2,1)	EKLIP53
EN(2,3)=(AM(48)*BDE(15)-AM(49)*BDE(6))*AC(2)+AM(50)	EKLIP54
EN(2,4)=EN(2,3)	EKLIP55
EN(2,5)=(AM(48)*BDE(18)-AM(49)*BDE(9))*AC(3)+AM(50)	EKLIP56
EN(2,6)=EN(2,5)	EKLIP57
EN(5,1)=EN(2,1)*EXP(P4)	EKLIP58
EN(5,2)=EN(2,2)*EXP(-P4)	EKLIP59
EN(5,3)=EN(2,3)*EXP(P5)	EKLIP60
EN(5,4)=EN(2,4)*EXP(-P5)	EKLIP61
EN(5,5)=EN(2,5)*EXP(P6)	EKLIP62
EN(5,6)=EN(2,6)*EXP(-P6)	EKLIP63
EN(3,1)=BDE(85)*AC(1)-BDE(3)*(BDE(84)*(AC(1)**2)+GAMB)+BDE(12)*(BDEKLP69 +E(83)*(AC(1)**2)+GAMA)	EKLIP64
EN(3,2)=EN(3,1)	EKLIP65
EN(3,3)=BDE(85)*AC(2)-BDE(6)*(BDE(84)*(AC(2)**2)+GAMB)+BDE(15)*(BDEKLP72 +E(83)*(AC(2)**2)+GAMA)	EKLIP66
EN(3,4)=-EN(3,3)	EKLIP67
EN(3,5)=BDE(85)*AC(3)-BDE(9)*(BDE(84)*(AC(3)**2)+GAMB)+BDE(18)*(BDEKLP75 +E(83)*(AC(3)**2)+GAMA)	EKLIP68
EN(3,6)=-EN(3,5)	EKLIP69
EN(6,1)=EN(3,1)*EXP(P4)	EKLIP70
EN(6,2)=EN(3,2)*EXP(-P4)	EKLIP71
EN(6,3)=EN(3,3)*EXP(P5)	EKLIP72
EN(6,4)=EN(3,4)*EXP(-P5)	EKLIP73
EN(6,5)=EN(3,5)*EXP(P6)	EKLIP74
EN(6,6)=EN(3,6)*EXP(-P6)	EKLIP75
AL(1)=-BDE(3)*V(1)	EKLIP76
AL(2)=BDE(3)*V(2)	EKLIP77
AL(3)=-BDE(6)*V(3)	EKLIP78
AL(4)=BDE(6)*V(4)	EKLIP79
AL(5)=-BDE(9)*V(5)	EKLIP80
AL(6)=BDE(9)*V(6)	EKLIP81
	EKLIP82
	EKLIP83
	EKLIP84
	EKLIP85
	EKLIP86
	EKLIP87
	EKLIP88
	EKLIP89
	EKLIP90
	EKLIP91
	EKLIP92
	EKLIP93
	EKLIP94
	EKLIP95
	EKLIP96

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AL(7)=BDE(12)*V(1) EKLI P97
AL(8)=-BDE(12)*V(2) EKLI P98
AL(9)=BDE(15)*V(3) EKLI P99
AL(10)=-BDE(15)*V(4) EKLI P
AL(11)=BDE(18)*V(5) EKLI P
AL(12)=-BDE(18)*V(6) EKLI P
WW=V(1)*EXP(P1)+V(2)*EXP(-P1)+V(3)*EXP(P2)+V(4)*EXP(-P2)+V(5)*EXP(EKLI P
1*P3)+V(6)*EXP(-P3)+BDE(49) EKLI P
DWK=V(1)*BDE(101)+V(2)*BDE(111)+V(3)*BDE(161)+V(4)*BDE(171)+V(5)*BDE(171) EKLI P
1*BDE(124)+V(6)*BDE(251) EKLI P
DDWK=V(1)*BDE(102)+V(2)*BDE(112)+V(3)*BDE(162)+V(4)*BDE(172)+V(5)*BDE(172) EKLI P
1*BDE(124)+V(6)*BDE(252) EKLI P
WB=AL(1)*EXP(P1)+AL(2)*EXP(-P1)+AL(3)*EXP(P2)+AL(4)*EXP(-P2)+AL(5)*EKLIP
1*EXP(P3)+AL(6)*EXP(-P3) EKLI P
DWB=AL(1)*BDE(101)+AL(2)*BDE(111)+AL(3)*BDE(161)+AL(4)*BDE(171)+ALEKLI P
1(5)*BDE(241)+AL(6)*BDE(251) EKLI P
DDWB=AL(1)*BDE(102)+AL(2)*BDE(112)+AL(3)*BDE(162)+AL(4)*BDE(172)+AEKLI P
1L(5)*BDE(124)+AL(6)*BDE(252) EKLI P
DDDWB=AL(1)*BDE(103)+AL(2)*BDE(113)+AL(3)*BDE(163)+AL(4)*BDE(173)+EKLIP
1AL(5)*BDE(124)+AL(6)*BDE(253) EKLI P
EKLI P
WA=AL(7)*EXP(P1)+AL(8)*EXP(-P1)+AL(9)*EXP(P2)+AL(10)*EXP(-P2)+AL(11)EKLI P
11)*EXP(P3)+AL(12)*EXP(-P3) EKLI P
DWA=AL(7)*BDE(101)+AL(8)*BDE(111)+AL(9)*BDE(161)+AL(10)*BDE(171)+AEKLI P
1L(11)*BDE(241)+AL(12)*BDE(251) EKLI P
DDWA=AL(7)*BDE(102)+AL(8)*BDE(112)+AL(9)*BDE(162)+AL(10)*BDE(172)+EKLIP
1AL(11)*BDE(124)+AL(12)*BDE(252) EKLI P
DDDWHA=AL(7)*BUE(103)+AL(8)*BUE(113)+AL(9)*BDE(163)+AL(10)*BDE(173)+EKLIP
1*AL(11)*BDE(124)+AL(12)*BUE(253) EKLI P
FN7WW=(V(1)*EXP(P1)-V(2)*EXP(-P1))/AC(1)+(V(3)*EXP(P2)-V(4)*EXP(-P2))/AC(2) EKLI P
12)/AC(2)+(V(5)*EXP(P3)-V(6)*EXP(-P6))/AC(3)+BDE(49)*X EKLI P
IF (X,EQ.,-ELL/2.) CALL UCJFF (FN7WW,CONST) EKLI P
WU=-BDE(19)*WA-BDE(20)*WB-BDE(21)*FNTWH+BT(20)/CAPCB*X+CONST EKLI P
DWU=RDE(14)*DWA-RDF(20)*DWB-BDE(21)*WW+BT(20)/CAPCB EKLI P
DDWU=-BDE(14)*DDWA-RDF(20)*DDWB-BDE(21)*UWW EKLI P
DDDWU=-BDE(19)*DDWA-BDE(20)*DDWB-BDE(21)*DDWW EKLI P
TAU=LCA01-BDE(22)*DUWU-BDE(23)*DUWW EKLI P
DTAU=-BDE(22)*DDDWU-BDE(23)*DDWW EKLI P
NPJ=LCA02-BUE(24)*DDWB-BDE(25)*DTAU+BDE(23)*DWU+BDE(26)*WW EKLI P
EKLI P
CALL RESULT (X) EKLI P
CALL PRINT (X) EKLI P
RETURN EKLI P
END EKLI P
SUBROUTINE DIFF (P1,P2,P3) EKLI P
COMMON DLTEMP
COMMON TEMP,EA,EAC,NUA,NUAC,EB,FBC,NUB,NUBC,MA,HB,R,G3A,G3B,CA,CB,
1DA,UR,H,CACB,CAPCB,NNUU,GAMA,GAMR,K(40),A(55),G(7),HT(30),FVTOA,FN
2T0B,BDE(100),LOAD1,AM(50),LAU2,EN(6,7),FNXA,FNXB,FMXA,FMXB,FNX,FN
3X,FN0A,FN0B,AC(3),KLIP,ELL,DELPHI(20),V(6),AL(18),WW,DWU,DDWW,-B,DW
4B,DDWA,DDWB,WA,DWA,WU,DWU,DDWU,TAU,DTAU,NPJ,WWA,DWNA,WUA,DW
5UA
REAL NUA,NUAC,NUB,NUBC,NNUU,K,LCA01,LOAD2
BDE(101)=AC(1)*EXP(P1)
BDE(102)=BDE(101)*AC(1)
BDE(103)=BDE(102)*AC(1)

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BDE(104)=BDE(103)*AC(1)	14
	15
	16
BDE(111)=-AC(1)*EXP(-P1)	17
BDE(112)=-BDE(111)*AC(1)	18
BDE(113)=-BDE(112)*AC(1)	19
BDE(114)=-BDE(113)*AC(1)	20
	21
BDE(121)=EXP(P2)*(AC(2)*COS(P3)-AC(3)*SIN(P3))	22
BDE(122)=EXP(P2)*(AM(1)*COS(P3)-AM(2)*SIN(P3))	23
BDE(123)=EXP(P2)*(AM(3)*COS(P3)-AM(4)*SIN(P3))	24
BDE(124)=EXP(P2)*(AM(5)*COS(P3)-AM(6)*SIN(P3))	25
	26
BDE(131)=EXP(P2)*(AC(3)*COS(P3)+AC(2)*SIN(P3))	27
BDE(132)=EXP(P2)*(AM(2)*COS(P3)+AM(1)*SIN(P3))	28
BDE(133)=EXP(P2)*(AM(4)*COS(P3)+AM(3)*SIN(P3))	29
BDE(134)=EXP(P2)*(AM(6)*COS(P3)+AM(5)*SIN(P3))	30
	31
BDE(141)=-EXP(-P2)*(AC(2)*COS(P3)+AC(3)*SIN(P3))	32
BDE(142)=+EXP(-P2)*(AM(1)*COS(P3)+AM(2)*SIN(P3))	33
BDE(143)=-EXP(-P2)*(AM(3)*COS(P3)+AM(4)*SIN(P3))	34
BDE(144)=+EXP(-P2)*(AM(5)*COS(P3)+AM(6)*SIN(P3))	35
	36
BDE(151)=+EXP(-P2)*(AC(3)*COS(P3)-AC(2)*SIN(P3))	37
BDE(152)=-EXP(-P2)*(AM(2)*COS(P3)-AM(1)*SIN(P3))	38
BDE(153)=+EXP(-P2)*(AM(4)*COS(P3)-AM(3)*SIN(P3))	39
BDE(154)=+EXP(-P2)*(AM(6)*COS(P3)-AM(5)*SIN(P3))	40
	41
BDE(161)=AC(2)*EXP(P2)	42
BDE(162)=AC(2)*BDE(161)	43
BDE(163)=AC(2)*BDE(162)	44
BDE(164)=AC(2)*BDE(163)	45
	46
BDE(171)=-AC(2)*EXP(-P2)	47
BDE(172)=-AC(2)*PDE(171)	48
BDE(173)=-AC(2)*RDE(172)	49
BDE(174)=-AC(2)*BDE(173)	50
	51
BDE(181)=-AC(1)*SIN(P1)	52
BDE(182)=-(AC(1)**2)*COS(P1)	53
BDE(183)=(AC(1)**3)*SIN(P1)	54
BDE(184)=(AC(1)**4)*COS(P1)	55
	56
BDE(191)=AC(1)*COS(P1)	57
BDE(192)=-(AC(1)**2)*SIN(P1)	58
BDE(193)=-(AC(1)**3)*COS(P1)	59
BDE(194)=(AC(1)**4)*SIN(P1)	60
	61
BDE(201)=-AC(2)*SIN(P2)	62
BDE(202)=-(AC(2)**2)*COS(P2)	63
BDE(203)=(AC(2)**3)*SIN(P2)	64
BDE(204)=(AC(2)**4)*COS(P2)	65
	66
	67
	68
	69
	70
	71
	72
	73

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BDE(211)=AC(2)*COS(P2)          74
BDE(212)=-(AC(2)**2)*SIN(P2)    75
BDE(213)=-(AC(2)**3)*COS(P2)    76
BDE(214)=(AC(2)**4)*SIN(P2)     77
BDE(221)=-(AC(3)*SIN(P3))      78
BDE(222)=-(AC(3)**2)*COS(P3)    79
BDE(223)=(AC(3)**3)*SIN(P3)     80
BDE(224)=(AC(3)**4)*COS(P3)     81
BDE(231)=AC(3)*COS(P3)          82
BDE(232)=-(AC(3)**2)*SIN(P3)    83
BDE(233)=-(AC(3)**3)*COS(P3)    84
BDE(234)=(AC(3)**4)*SIN(P3)     85
BDE(241)=AC(3)*EXP(P3)          86
BDE(242)=AC(3)*BDE(241)         87
BDE(243)=AC(3)*BDE(242)         88
BDE(244)=AC(3)*BDE(243)         89
BDE(251)=-(AC(3)*EXP(-P3))     90
BDE(252)=-(AC(3)*BDE(251))     91
BDE(253)=-(AC(3)*BDE(252))     92
BDE(254)=-(AC(3)*BDE(253))     93
RETURN                         94
END                            95
SUBROUTINE POLYR (N,COEFF,ROOTS,D) 106-
DIMENSION A(51,3), IA(51,3), ROOTS(2,N), D(1), COEFF(1)
INTEGER DEGREE
DEGREE=4                          1
N1=DEGREE+1                        2
M=10                             3
MMAX=15                           4
DELTA=0.0001                        5
EPSILON=0.000001                   6
DO 1 I=1,N1                        7
A(I,1)=COEFF(1)                   8
IA(I,1)=0                           9
CALL SCALE (A(I,1),IA(I,1))        10
CONTINUE                         11
CALL RSSR (A,IA,ROOTS,DEGREE,M,MMAX,DELTA,EPSILON,D) 12
IF (N1-(DEGREE+1)) 3,3,2          13
RETURN                           14
PRINT 4                           15
RETURN                           16
FORMAT (2)NO SOME ROOTS NOT FOUND 17
END                            18P
SUBROUTINE RSSR (A,IA,ROOTS,DEGREE,M,MMAX,DELTA,FPSILON,D) 22-
DIMENSION A(51,3), IA(51,3), ROOTS(2,50), D(51), ROMGD(50), MNOMGD
I(50), MNONRT(50)                 1
INTEGER DEGREE                     2
M=DEGREE                          3

```

```

1 IF (IN) 1,1,3          6
2 DEGREE=NCUR           7
3 RETURN                 8
4 N1=N+1                 9
5 N2=N1+1               10
6 DO 5 I=1,N             11
7 K=N2-I               12
8 IF (A(K,I)) 6,4,6     13
9 J=N1-1               14
10 ROOTS(1,J)=0.0        15
11 ROOTS(2,J)=0.0        16
12 CONTINUE               17
13 DEGREE=0               18
14 GO TO 2               19
15 N1=K                  20
16 N=K-1                 21
17 NCUR=N                22
18 NL=4                  23
19 CALL ROOTSQ (A,IA,NCUR,M) 24
20 CALL REALRT (A,IA,M,NCUR,DELT,A,EPISLON,ROMOD,MROMOD,NONRT,MNONRT,N
21 1CO,ROOTS)            25
22 IF (NCO) 12,12,8      26
23 N1=NCUR+1             27
24 CALL COMPR (A,IA,ROMOD,ROOTS,M,MNONRT,NONRT,MROMOD,NCO,DELT,A,EPISI
25 1LON,NCUR)            26
26 IF (NCUR) 12,12,9      27
27 IF (NL-NCUR) 11,11,10   28
28 NL=NCUR               29
29 GO TO 7               30
30 M=M+1                 31
31 IF (MMAX-M) 1,7,7      32
32 CALL RECOV (ROOTS,A(1,1),IA(1,1),D,DEGREE) 33
33 GO TO 1               34
34 END                   35
35 SUBROUTINE ROOTSQ (A,IA,NCUR,MM)
36 DIMENSION A(51,3), IA(51,3) 36
37 N1=NCUR+1              37
38 DO 1 J=1,N1             38
39 A(J,2)=A(J,1)           39
40 IA(J,2)=IA(J,1)         40
41 A(J,3)=0.0               41
42 IA(J,3)=0               42
43 CONTINUE               43
44 DO 10 M=1,MM             44
45 DO 7 J=1,N1             45
46 K1=N1-J                 46
47 K2=J-1                 47
48 KM=XMINOF(K1,K2)        48
49 IF (KM) 2,5,2             49
50 DO 5 L=1,KM             50
51 LR=XMODF(L,2)           51
52 JL=J-L                 52
53 JLP=J+L                 53
54 IF (LR) 3,3,4             54
55 X=A(JL,2)*A(JLP,2)       55
56 BX=IA(JL,2)*IA(JLP,2)    56
57 CALL SCALE (X,BX)        57
58 CALL ADD (A(J,3),IA(J,3),X,BX,A(J,3),IA(J,3)) 58
59 GO TO 5                 59
60 X=A(JL,2)*A(JLP,2)       60
61 RTSQ 1
62 RTSQ 2
63 RTSQ 3
64 RTSQ 4
65 RTSQ 5
66 RTSQ 6
67 RTSQ 7
68 RTSQ 8
69 RTSQ 9
70 RTSQ 10
71 RTSQ 11
72 RTSQ 12
73 RTSQ 13
74 RTSQ 14
75 RTSQ 15
76 RTSQ 16
77 RTSQ 17
78 RTSQ 18
79 RTSQ 19
80 RTSQ 20
81 RTSQ 21
82 RTSQ 22
83 RTSQ 23
84 RTSQ 24
85 RTSQ 25
86 RTSQ 26

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```

1 IX=IA(JL,2)+IA(JLP,2) RTSQ 27
2 CALL SCALE (X,[X]) RTSQ 28
3 CALL SBTRT (A(J,3),IA(J,3),X,IX,A(J,3),IA(J,3)) RTSQ 29
4 5 CONTINUE RTSQ 30
5 A(J,3)=2.0*A(J,3) RTSQ 31
6 CALL SCALE (A(J,3),IA(J,3)) RTSQ 32
7 X=A(J,2)*2 RTSQ 33
8 IX=IA(J,2)+IA(J,2) RTSQ 34
9 CALL SCALE (X,[X]) RTSQ 35
10 CALL ADD (A(J,3),IA(J,3),X,IX,A(J,3),IA(J,3)) RTSQ 36
11 JR=XMODF(J,2) RTSQ 37
12 IF (JR) 6,6,7 RTSQ 38
13 6 A(J,3)=-A(J,3) RTSQ 39
14 7 CONTINUE RTSQ 40
15 IF (MM-M) 10,10,8 RTSQ 41
16 8 DO 9 J=1,N1 RTSQ 42
17 A(J,2)=A(J,3) RTSQ 43
18 IA(J,2)=IA(J,3) RTSQ 44
19 A(J,3)=0.0 RTSQ 45
20 IA(J,3)=0 RTSQ 46
21 9 CONTINUE RTSQ 47
22 10 CONTINUE RTSQ 48
23 RETURN RTSQ 49
24 END RTSQ 50-
25 SUBROUTINE REALRT (A,IA,M,NCUR,DELTA,EPISILON,ROMOD,MROMOD,NONRT,MN***** 1
26 00000 2
27 IONRT,NCU,ROUTS)
28 DIMENSION A(51,3), [A(51,3)], ROOTS(2,50), ROMOD(50), MROMOU(50), NRLRT 3
29 IONRT(50), MNDRHT(50), RATIO(51), IPIV(51), ARED(50), IARED(50) RLRT 4
30 RATIO(1)=1.0 RLRT 5
31 DO 6 I=2,NCUR RLRT 6
32 II=XMODF(I,2) RLRT 7
33 IF (A(II,3)) 2,1,2 RLRT 8
34 1 RATIO(I)=0.0 RLRT 9
35 GO TO 6 RLRT 10
36 2 T=A(II,2)*A(I,2) RLRT 11
37 IT=IA(II,2)+IA(I,2) RLRT 12
38 CALL SCALE (T,IT) RLRT 13
39 T=T/A(I,3) RLRT 14
40 IT=IT-IA(I,3) RLRT 15
41 IF (IT-2) 3,3,1 RLRT 16
42 3 IF (IT+2) 1,4,4 RLRT 17
43 4 CALL UNSCALE (T,IT) RLRT 18
44 RATIO(I)=T RLRT 19
45 IF (II) 5,5,6 RLRT 20
46 5 RATIO(I)=-RATIO(I) RLRT 21
47 6 CONTINUE RLRT 22
48 RATIO(NCUR+1)=1.0 RLRT 23
49 IPIV(1)=1 RLRT 24
50 IPIV(NCUR+1)=1 RLRT 25
51 DO 9 I=2,NCUR RLRT 26
52 X=ABSF(RATIJ(I)-1.0) RLRT 27
53 IF (X-DELTA) 7,8,8 RLRT 28
54 7 IPIV(I)=1 RLRT 29
55 GO TO 9 RLRT 30
56 8 IPIV(I)=0 RLRT 31
57 9 CONTINUE RLRT 32
58 NCUR1=NCUR+1 RLRT 33
59 II=0 RLRT 34
60 MULT=0 RLRT 35
61 I=1 RLRT 36

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14=1
10 I1=I1+1
12=I1+1
MULT=MULT+1
IF (IPIV(I2)) 10,10,11
11 ROMOD(I4)=A(I2,3)/A(I1,3)
IROMOD=IA(I2,3)-IA(I1,3)
CALL SCALE (ROMOD(I4),IROMOD)
IF (IROMOD(I4)) 12,13,13
12 ROMOD(I4)=-ROMOD(I4)
13 CALL DOURLOG (ROMOD(I4),IROMOD,XN,IXN)
T=2*XM*II
XN=XN/T
CALL SCALE (XN,IXN)
CALL OLEXP (XN,IXN,ROMOD(I4),IROMOD)
IF (IROMOD-74) 14,14,15
14 IF (IROMOD+74) 15,15,16
15 ROMOD(I4)=0.0
IROMOD=0
GO TO 17
16 CALL UNSCALE (ROMOD(I4),IROMOD)
17 MROMOD(I4)=MULT
IF (NCUR+I-12) 19,19,18
18 I=I2
I4=I4+1
MULT=0
I1=0
GO TO 10
19 Q=0.0
NCO=0
DO 28 I=L,I4
KL=I4+1-I
W=-ROMOD(KL)
IS=MROMOD(KL)
DO 26 J=1,IS
J=J
20 CALL TEST (A,IA,W,Q,NCUR,ROMOD(KL),EPSILON,K)
IF (K) 23,23,21
21 ROOTS(1,NCUR)=-W
ROOTS(2,NCUR)=0.0
ARED(1)=A(I1,1)
IARED(1)=IA(I1,1)
DO 22 L=2,NCUR
Y=ARED(L-1)*W
IY=IARED(L-1)
CALL SCALE (Y,IY)
CALL SBTKT (A(I,L,1),IA(I,L,1),Y,IY,ARED(L),IARED(L))
A(L,1)=ARED(L)
IA(L,1)=IARED(L)
22 CONTINUE
GO TO 25
23 IF (W) 24,27,27
24 W=-W
GO TO 20
25 NCUR=NCUR-1
26 CONTINUE
GO TO 28
27 NCO=NCO+1
NONRT(NCO)=KL
NONRT(NCO)=IS+1-J
RLRT 37
RLRT 38
RLRT 39
RLRT 40
RLRT 41
RLRT 42
RLRT 43
RLRT 44
RLRT 45
RLRT 46
RLRT 47
RLRT 48
RLRT 49
RLRT 50
RLRT 51
RLRT 52
RLRT 53
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RLRT 82
RLRT 83
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RLRT 85
RLRT 86
RLRT 87
RLRT 88
RLRT 89
RLRT 90
RLRT 91
RLRT 92
RLRT 93
RLRT 94
RLRT 95
RLRT 96

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28 CONTINUE          RLRT 97
      RETURN          RLRT 98
      END             RLRT 99-
      SUBROUTINE CCMPRT (A,IA,ROMOD,ROOTS,M,MNONRT,NOVRT,MRMODC,NCO,DELT***** 1
1A,EPISLON,NCURI)          ***** 2
      DIMENSION A(51,3), IA(51,3), ROMOD(50), ROOTS(2,50), SR(51,3), ISR 3
      1(51,3), SRMOD(50), SROOTS(2,50), MNOVRT(50), NOVRT(50), MSROMOD(5 4
      20), NSONRT(49), MSNORT(49), MRMOD(50), U(2), R(2), B(49)           5
      DO 28 I=1,NCO          6
      JA=MNONRT(I)          7
      II=MNONRT(I)          8
      II=II/2                9
      IF (II) 1,1,2          10
1   II=1                  11
2   IF (ROMOD(JA)) 3,28,3          12
3   Q=ROMOD(JA)          13
      DO 27 J=1,II          14
      CALL SUBRES (A,IA,NCUR,SR,ISR,Q)          15
      IF (NCUR-4) 5,4,6          16
4   NSCUR=2              17
      GO TO 7              18
5   NSCUR=1              19
      J2=1                  20
      GO TO 8              21
6   NSCUR=NCUR-3          22
7   J2=NSCUR          23
8   LL=NSCUR+1          24
      IF (NSCUR-1) 9,9,11          25
9   IF (SR(1,1)) 10,12,10          26
10  X=SR(1,1)          27
      IX=ISR(1,1)          28
      Y=SR(2,1)          29
      IY=ISR(2,1)          30
      CALL UNSCALE (X,IX)          31
      CALL UNSCALE (Y,IY)          32
      SROOTS(1,1)=-Y/X          33
      NSCUR=0              34
      GO TO 13              35
11  CALL ROOTSQ (SR,ISR,NSCUR,M)          36
      CALL REALRT (SR,ISR,M,NSCUR,DELTA,EPISLON,SROMOD,MSROMOD,NSONRT,MS
      1NORT,NSCO,SROOTS)          37
      IF (J2-NSCUR) 12,12,13          38
12  SROOTS(1,J2)=0.0          39
13  SROOTS(1,J2)=SROOTS(1,J2)*ROMOD(JA)          40
      T=ROMOD(JA)+RROMOD(JA)          41
      IF (SROOTS(1,J2)-T) 14,21,21          42
14  W=SROOTS(1,J2)          43
      WE=ROMOD(JA)*ROMOD(JA)          44
      CALL TEST (A,IA,W,WE,NCUR,ROMOD(JA),EPISLON,K)          45
      IF (K) 20,20,15          46
15  ROOTS(1,NCUR)=W/2.0          47
      T=4.0*WE          48
      U=W*W          49
      T=T-U          50
      IF (T) 16,16,17          51
16  T=-T          52
      U=SQRT(T)          53
      ROOTS(1,NCUR)=ROOTS(1,NCURI)-U/2.0          54
      ROOTS(1,NCURI-1)=-(W-U)/2.0          55
      ROOTS(2,NCURI)=0.0          56

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ROOTS(2,NCUR-1)=0.0      58
GO TO 18                  59
17 U=SQRT(T)              60
ROOTS(2,NCUR)=U/2.0       61
ROOTS(1,NCUR-1)=ROOTS(1,NCUR) 62
ROOTS(2,NCUR-1)=-ROOTS(2,NCUR) 63
18 D(1)=W                 64
D(2)=WE                  65
CALL QUADIV (NCUR,A,IA,R,D,B) 66
JX=NCUR-1                 67
DO 19 JY=1,JX              68
A(JY,1)=B(JY)              69
IA(JY,1)=0                  70
CALL SCALE (A(JY,1),IA(JY,1)) 71
19 CONTINUE                72
NCUR=NCUR-2                73
GO TO 27                  74
20 W=W
CALL TEST (A,IA,W,WE,NCUR,ROMOD(JA),EPSILON,K) 75
IF (K) 21,21,15             76
21 IF (J2-(NSCUR+1)) 28,22,24 77
22 IF (J2-1) 26,28,23       78
23 J2=J2-1                 79
SR ROOTS(1,J2)=0.0          80
GO TO 14                  81
24 IF (SR ROOTS(1,J2)-SR ROOTS(1,J2-1)) 25,26,25 82
25 J2=J2-1                 83
GO TO 13                  84
26 J2=J2-1                 85
GO TO 21                  86
27 CONTINUE                87
28 CCNTINUE                88
RETURN                     89
END                         90
91-
SUBROUTINE TEST (A,IA,W,Q,N,ROMOD,EPSILON,K)      **** 1
DIMENSION A(51,3), IA(51,3), B(3), T(2), E(2), C(51)
B(1)=0.0
IX=0
IW=0
IB(1)=0
B(2)=A(1,1)
IB(2)=IA(1,1)
DO 2 I=1,4
X=W*B(2)
IX=IB(2)
CALL SCALE (X,IX)
Y=Q*B(1)
IY=IN(1)
CALL SCALE (Y,IY)
CALL ADD (X,IX,Y,IY,Z,IZ)
CALL SBTRT (A(I+1,1),IA(I+1,1),Z,IZ,B(3),IB(3))
IF (IY-1) 2,2,1
1 B(1)=B(2)
IB(1)=IB(2)
B(2)=A(3)
IB(2)=IA(3)
2 CONTINUE
KOUT=1
CEPSIL=EPSILON
T(1)=0.0 T(2)=0.0
TEST 1
TEST 2
TEST 3
TEST 4
TEST 5
TEST 6
TEST 7
TEST 8
TEST 9
TEST 10
TEST 11
TEST 12
TEST 13
TEST 14
TEST 15
TEST 16
TEST 17
TEST 18
TEST 19
TEST 20
TEST 21
TEST 22
TEST 23
TEST 24
TEST 25
TEST 26

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N1=N+1          TEST 27
X=2.0*EPSILON   TEST 28
Y=X*ROMOD       TEST 29
E(1)=ROMOD*Y    TEST 30
E(2)=ROMOD*CEPSIL*ROMOD  TEST 31
DO 6 I=1,N1     TEST 32
IF (A(I,I)) 3,4,4  TEST 33
3 C(I)=A(I,I)*C=IA(I,I)  TEST 34
GO TO 5         TEST 35
4 C(I)=A(I,I)*C=IA(I,I)  TEST 36
5 CALL UNSCALE (C(I),IC)  TEST 37
T(1)=T(1)*E(1)*C(I)      TEST 38
T(2)=T(2)*E(2)*C(I)      TEST 39
6 CONTINUE       TEST 40
DIF=T(1)-T(2)      TEST 41
IF (0) 1B,7,1B      TEST 42
7 IF (R(3)) 8,9,9    TEST 43
8 B(3)=B(3)        TEST 44
9 IF ((B(3)-7)) 10,10,12  TEST 45
10 IF ((R(3)+7)) 12,11,11  TEST 46
11 CALL UNSCALE (B(3),R(3))  TEST 47
IF (DIF-B(3)) 12,12,17  TEST 48
12 K=0            TEST 49
IF (KOUNT-2) 13,14,16  TEST 50
13 IF (J) 15,14,15  TEST 51
14 IF (W) 15,16,16  TEST 52
15 SENSE EIGHT 2    TEST 53
KOUNT=KOUNT+1      TEST 54
16 RETURN         TEST 55
17 K=1            TEST 56
GO TO 16         TEST 57
18 IF ((B(2)-7)) 19,19,12  TEST 58
19 IF ((B(2)+7)) 12,20,20  TEST 59
20 CALL UNSCALE (B(2),B(2))  TEST 60
IF ((B(3)-7)) 21,21,12  TEST 61
21 IF ((R(3)+7)) 12,22,22  TEST 62
22 CALL UNSCALE (B(3),B(3))  TEST 63
X=Q*B(2)*B(2)    TEST 64
Y=W*B(2)*B(3)    TEST 65
Z=B(3)*B(3)      TEST 66
Y=X-Y+Z          TEST 67
IF (Y) 23,17,24  TEST 68
23 Y=-Y          TEST 69
24 DIF=DIF+DIF  TEST 70
IF (DIF-Y) 12,17,17  TEST 71
END             TEST 72-
SUBROUTINE SURRES (A,IA,N,SR,ISR,ROMOD)
DIMENSION A(51,3), IA(51,3), SR(51,3), ISR(51,3), C(51), R(50,3)
N1=N+1          1
T=1.0           2
DO 1 I=1,N      3
J=N1-I          4
T=ROMOD         5
C(J)=A(J,I)*T  6
IC=IA(J,I)      7
CALL UNSCALE (C(J),IC)  8
9
1 CONTINUE       10
C(N1)=A(N1,1)   11
IC=IA(N1,1)     12
CALL UNSCALE (C(N1),IC)  13
14

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```

1 IF (N-2) 17,17,2
2 N2=N-2
3 DO 3 I=1,N2
4 B(I,1)=0.0
5 B(I,2)=0.0
6 CONTINUE
7 I=2
8 B(I,2)=C(I)
9 B(I,3)=C(I)-B(I,1)
10 DO 5 J=2,N2
11 B(J,3)=-B(J-1,2)-R(J,1)
12 CONTINUE
13 IF (N-(3+I)) 8,6,6
14 I=I+1
15 DO 7 J=1,N2
16 B(J,1)=B(J,2)
17 B(J,2)=H(J,3)
18 CONTINUE
19 GO TO 4
20 IF (N-4) 19,9,13
21 IF (N-(2+I)) 12,10,10
22 10 I=I+1
23 DO 11 J=1,2
24 B(J,1)=B(J,2)
25 B(J,2)=B(J,3)
26 CONTINUE
27 GO TO 4
28 IF (N-4) 19,9,13
29 IF (N-(2+I)) 12,10,10
30 11 I=I+1
31 DO 12 J=1,2
32 B(J,1)=B(J,2)
33 B(J,2)=B(J,3)
34 CONTINUE
35 GO TO 4
36 IF (N-4) 19,9,13
37 IF (N-(2+I)) 12,10,10
38 12 I=I+1
39 DO 13 J=1,2
40 B(J,1)=B(J,2)
41 B(J,2)=B(J,3)
42 SR(3,1)=-C(5)+B(1,3)
43 ISR(3,1)=0
44 SR(2,1)=B(2,3)
45 ISR(2,1)=0
46 SR(1,1)=B(3,3)
47 ISR(1,1)=0
48 CALL SCALE (SR(1,1),ISR(1,1))
49 CALL SCALE (SR(2,1),ISR(2,1))
50 CALL SCALE (SR(3,1),ISR(3,1))
51 GO TO 16
52 IF (N2-2) 16,16,14
53 SR(N2,1)=C(4)-B(1,3)
54 ISR(N2,1)=0
55 SR(N2-1,1)=-C(N1)-B(2,3)
56 ISR(N2-1,1)=0
57 CALL SCALE (SR(N2,1),ISR(N2,1))
58 CALL SCALE (SR(N2-1,1),ISR(N2-1,1))
59 IF (N2-2) 16,16,14
60 DO 15 J=3,N2
61 K=N2+1-J
62 SR(K,1)=-B(J,3)
63 ISR(K,1)=0
64 CALL SCALE (SR(K,1),ISR(K,1))
65 CONTINUE
66 RETURN
67 SR(1,1)=C(1)
68 ISR(1,1)=0
69 SR(2,1)=-C(2)
70 ISR(2,1)=0
71 CALL SCALE (SR(1,1),ISR(1,1))
72 CALL SCALE (SR(2,1),ISR(2,1))
73 GO TO 16
74 SR(1,1)=-C(4)

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SR(2,1)=C(3)-C(1)          75
ISR(1,1)=0                  76
ISR(2,1)=0                  77
GO TO 18                     78
END                         79-
SUBROUTINE RECON (ROOTS,A,[A,D,N])      ***** 1
DIMENSION ROOTS(2,50), D(50)
X=A                           2
IX=[A                         3
CALL UNSCALE (X,IX)           4
DO I I=1,N                   5
D(I)=0.0                      6
1 CONTINUE                     7
D(N+1)=1.0                    8
I=1                          9
NL=N-1                       10
2 IF (ROOTS(2,1)) 3,7,3       11
3 T=ROOTS(1,1)*ROOTS(1,1)     12
U=ROOTS(2,1)*ROOTS(2,1)     13
T=T+U                         14
U=2.0*ROOTS(1,1)             15
DO S J=1,NL                  16
IF (I+J-N) 5,6,6              17
4 D(J)=D(J+2)+T*D(J)         18
D(J)=D(J)-U*D(J+1)           19
5 CONTINUE                     20
D(N)=T*D(N)                  21
D(N)=D(N)-U*D(N+1)           22
D(N+1)=T*D(N+1)              23
I=I+2                         24
6 IF (N-I) 10,2,2             25
7 DO 9 J=1,N                  26
IF (I+J-N) 9,9,8              27
8 D(J)=D(J+1)-D(J)*ROOTS(1,1) 28
9 CONTINUE                     29
D(N+1)=-D(N+1)*ROOTS(1,1)    30
I=I+1                         31
GO TO 6                        32
10 NS=N+1                      33
DO 11 I=1,NS                  34
D(I)=D(I)*X                   35
11 CONTINUE                     36
RETURN                        37
END                         38
39-
SUBROUTINE QUADIV (N,A,[A,R,D,B)
DIMENSION A(51,3), IA(51,3), R(2), B(49)
B(1)=A(1,1)                  1
IA(1,1)=1                      2
CALL UNSCALE (B(1),IA)          3
IF (N-2) 4,4,1                 4
1 AA=A(2,1)                    5
IAA=IA(2,1)                   6
CALL UNSCALE (AA,IAA)           7
B(2)=AA-R(1)*D(1)             8
IF (N-3) 4,4,2                 9
10 NT=N-1                      10
DO 13 I=3,NT                  11
RN=R(I-1)*D(I)                12
VN=B(I-2)*U(2)                13
AA=A(I,1)                      14
AA=A(I,1)                      15
AA=A(I,1)                      16

```

```

1 AA=IA(1,1) 17
2 CALL UNSCALE (AA,IAA) 18
3 B(1)=AA-(XN+YN) 19
4 CONTINUE 20
5 XN=B(N-1)*D(1) 21
6 YN=B(N-2)*D(2) 22
7 AA=A(N,1) 23
8 IAA=IA(N,1) 24
9 CALL UNSCALE (AA,IAA) 25
10 R(1)=AA-(XN+YN) 26
11 AA=A(N+1,1) 27
12 IAA=IA(N+1,1) 28
13 CALL UNSCALE (AA,IAA) 29
14 R(2)=AA-B(N-1)*D(2) 30
15 RETURN 31
16 END 32-
17 SUBROUTINE DOUBLG (X,IX,Y,IY) 32-
18 T=64.0 33
19 IF (IX) 1,2,3 34
20 1 PRINT 5 4P
21 2 Y=0.0 5
22 3 IY=0 6
23 GO TO 4 7
24 3 TO=IX 8
25 Y= ALOG(X)+T* ALOG(T) 9
26 IY=0 10
27 CALL SCALE (Y,IY) 11
28 4 RETURN 12
29
30 FORMAT (4H0!THE LOG OF A NON-POSITIVE NUMBER IS REQUESTED!) 13
31 END 14
32 SUBROUTINE DLEXP (X,IX,Z,IZ) 15-
33 Z=EXP(X) 1
34 IZ=0 2
35 IF (IX) 1,0,6 3
36 1 I=IX 4
37 I=6*I 5
38 DO 4 J=1,11 6
39 K=XHDF(IZ,2) 7
40 IF (K) 2,3,2 8
41 2 IZ=IZ-1 9
42 Z=64.0*Z 10
43 IZ=IZ/2 11
44 Z=SQRT(Z) 12
45 CALL SCALE (Z,IZ) 13
46 CONTINUE 14
47 RETURN 15
48 I=6*I X 16
49 DO 7 J=1,1 17
50 Z=Z*Z 18
51 IZ=IZ+IZ 19
52 CALL SCALE (Z,IZ) 20
53 7 CONTINUE 21
54 GO TO 5 22
55 CALL SCALE (Z,IZ) 23
56 GO TO 5 24
57 END 25
58 SUBROUTINE ADD (X,IX,Y,IY,Z,IZ) 26-
59 IF (IX) 3,1,3 6
60 Z=Y 7

```

```

1 Z=Y
2 RETURN
3 IF (Y) 3,4,5
4 Z=X
5 Z=IX
6 GO TO 2
7 IDIFF=(X-Y)
8 IF (IDIFF) 6,7,7
9 IA=IY
A=Y
B=X
IDIFF=-IDIFF
GO TO 8
10 IA=IX
A=X
B=Y
11 IF (16-IDIFF) 9,9,10
12 Z=A
13 Z=IA
14 GO TO 2
15 IF (IDIFF) 11,13,11
16 DO 12 I=1,1DIFF
17 B=B/64.0
18 CONTINUE
19 CONTINUE
20 Z=A+B
21 Z=IA
22 CALL SCALE (Z,IZ)
23 GO TO 2
24 END
25 SUBROUTINE SBTRT (X,IX,Y,IY,Z,IZ)
26 W=-Y
27 CALL ADD (X,IX,W,IY,Z,IZ)
28 RETURN
29 ENDO
30 SUBROUTINE SCALE (X,IX)
31 REC64=1.0/64.0
32 IF (X) 1,11,2
33 Y=X
34 GO TO 3
35 Y=X
36 IF (64.0-Y) 4,5,5
37 Y=Y/64.0
38 IX=IX+1
39 GO TO 3
40 IF (Y-REC64) 6,7,7
41 Y=Y*64.0
42 IX=IX-1
43 GO TO 5
44 IF (X) 8,9,9
45 X=-Y
46 GO TO 10
47 X=Y
48 RETURN
49 IX=0
50 GO TO 10
51 ENDO
52 SUBROUTINE UNSCALE (X,IX)
53 IF (IX+R4) 1,2,2
54 X=0.0

```

```

1 IX=0
2 GO TO 6
3 IF (IX=94) 4,4,3
4 X=1.0E+153
5 IX=0
6 RETURN

7 FORMAT (25HOEXP. OVERFLOW IN UNSCALE)
8 END
9
10 SUBROUTINE CHECK1 (V,AC,PP1,PP2,PP3)
11 DIMENSION V(6), AC(3)
12 V(1)=V(1)*EXP(PP1)
13 V(2)=V(2)*EXP(-PP1)
14 AV3=EXP(PP2)*(V(3)*COS(PP3)+V(4)*SIN(PP3))
15 AV4=EXP(PP2)*(V(4)*COS(PP3)-V(3)*SIN(PP3))
16 AV5=EXP(-PP2)*(V(5)*COS(PP3)+V(6)*SIN(PP3))
17 AV6=EXP(-PP2)*(V(6)*COS(PP3)-V(5)*SIN(PP3))
18 V(3)=AV3
19 V(4)=AV4
20 V(5)=AV5
21 V(6)=AV6
22
23 RETURN
24 END
25
26 SUBROUTINE CHECK2 (V,AC,PP1,PP2,PP3)
27 DIMENSION V(6), AC(3)
28 AV1=V(1)*COS(PP1)+V(2)*SIN(PP1)
29 AV2=-V(1)*SIN(PP1)+V(2)*COS(PP1)
30 V(1)=AV1
31 V(2)=AV2
32 AV3=EXP(PP2)*(V(3)*COS(PP3)+V(4)*SIN(PP3))
33 AV4=EXP(PP2)*(V(4)*COS(PP3)-V(3)*SIN(PP3))
34 AV5=EXP(-PP2)*(V(5)*COS(PP3)+V(6)*SIN(PP3))
35 AV6=EXP(-PP2)*(V(6)*COS(PP3)-V(5)*SIN(PP3))
36 V(3)=AV3
37 V(4)=AV4
38 V(5)=AV5
39 V(6)=AV6
40
41 RETURN
42 END
43
44 SUBROUTINE CHECK4 (V,AC,PP1,PP2,PP3)
45 DIMENSION V(6), AC(3)
46 V(1)=V(1)*EXP(PP1)
47 V(2)=V(2)*EXP(-PP1)
48 AV3=V(3)*COS(PP2)+V(4)*SIN(PP2)
49 AV4=-V(3)*SIN(PP2)+V(4)*COS(PP2)
50 AV5=V(5)*COS(PP3)+V(6)*SIN(PP3)
51 AV6=-V(5)*SIN(PP3)+V(6)*COS(PP3)
52 V(3)=AV3
53 V(4)=AV4
54 V(5)=AV5
55 V(6)=AV6
56
57 RETURN
58 END
59
60 SUBROUTINE CHECK5 (V,AC,PP1,PP2,PP3)
61
62 ***** 1
63 CHK4 2
64 CHK4 3
65 CHK4 4
66 CHK4 5
67 CHK4 6
68 CHK4 7
69 CHK4 8
70 CHK4 9
71 CHK4 10
72 CHK4 11
73 CHK4 12
74 CHK4 13
75 CHK4 14-
76 ***** 1

```

```

DIMENSION V(6), AC(3)
V(1)=V(1)*EXP(PP1)
V(2)=V(2)*EXP(-PP1)
V(3)=V(3)*EXP(PP2)
V(4)=V(4)*EXP(-PP2)
AV5=V(5)*COS(PP3)+V(6)*SIN(PP3)
AV6=-V(5)*SIN(PP3)+V(6)*COS(PP3)
V(5)=AV5
V(6)=AV6
RETURN
END
SUBROUTINE RICF (ROOT,AC,KLIP)
DIMENSION ICJ(6), IMI(6), IRE(6)
DIMENSION ROOT(2,6), AC(3)
COMMON /KLP/ IC1,IC2,IC3,IC4,IC5,IC6,IM1,IM2,IM3,IM4,IM5,IM6,IR1,IRICF
IR2,IR3,IR4,IR5,IR6
EQUIVALENCE (IC1,IC0(1)), (IM1,IM(1)), (IR1,IRE(1))
WRITE (9,20)
WRITE (9,21) ((ROOT(I,J),I=1,2),J=1,6)
DO 1 J=1,6
IC0(J)=0
IMI(J)=0
IRE(J)=0
1 CONTINUE
IR=3
IC=0
II=0
DO 5 J=1,6
IF (ABS(ROOT(1,J)).LT.1.E-12) ROOT(1,J)=0.0
IF (ABS(ROOT(2,J)).LT.1.E-12) ROOT(2,J)=0.0
IF (ROOT(2,J).EQ.0.0) GO TO 4
IF (ROOT(1,J).LT.0.0) GO TO 2
2 IC=IC+1
IC0(IC)=J
GO TO 5
3 II=II+1
IMI(II)=J
GO TO 5
4 IR=IR+1
IRE(IR)=J
5 CONTINUE
WRITE (9,22) IR,II,IC
IF (IR.EQ.2.AND.IC.EQ.4) GO TO 6
IF (IR.EQ.2.AND.IM.EQ.4) GO TO 7
IF (IM.EQ.6) GO TO 9
IF (IP.EQ.2.AND.IC.EQ.4) GO TO 16
IF (IR.EQ.4.AND.IM.EQ.2) GO TO 17
IF (IR.EQ.6) GO TO 19
GO TO 19

6 KLIP=1
AC(1)=ABS(ROOT(1,IR))
AC(2)=ABS(ROOT(1,IC))
AC(3)=ABS(ROOT(2,IC))
RETURN
7 KLIP=2
AC(1)=ABS(ROOT(1,IR))

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AC(2)=ABS(ROOT(2,IM1))          RICF 50
IF (AC(2).NE.ABS(ROOT(2,IM2))) GO TO 8  RICF 51
AC(3)=ABS(ROOT(2,IM3))          RICF 52
RETURN                           RICF 53
8 AC(3)=ABS(ROOT(2,IM2))          RICF 54
RETURN                           RICF 55
RICF 56
RICF 57
RICF 58
RICF 59
RICF 60
RICF 61
RICF 62
RICF 63
RICF 64
RICF 65
RICF 66
RICF 67
RICF 68
RICF 69
RICF 70
RICF 71
RICF 72
RICF 73
RICF 74
RICF 75
RICF 76
RICF 77
RICF 78
RICF 79
RICF 80
RICF 81
RICF 82
RICF 83
RICF 84
RICF 85
RICF 86
RICF 87
RICF 88
RICF 89
RICF 90
RICF 91
RICF 92
RICF 93
RICF 94
RICF 95
RICF 96
RICF 97
RICF 98
RICF 99
RICF100-
***** 1
2
3
4
5
6
7
8
9

* KLIP=3
AC(1)=ABS(ROOT(2,IM1))
IF (ABS(ROOT(2,IM1)).NE.ABS(ROOT(2,IM2))) GO TO 10
AC(2)=ABS(ROOT(2,IM3))
IF (ABS(ROOT(2,IM4)).NE.AC(1)).AND.ABS(ROOT(2,IM4)).NE.AC(2)) GO TO 12
1R12
GO TO 14
10 AC(2)=ABS(ROOT(2,IM2))
IF (ABS(ROOT(2,IM3)).EQ.AC(1)) GO TO 11
IF (ABS(ROOT(2,IM3)).EQ.AC(2)) GO TO 13
AC(3)=ABS(ROOT(2,IM5))
RETURN
11 IF (ABS(ROOT(2,IM4)).EQ.AC(2)) GO TO 14
12 AC(3)=ABS(ROOT(2,IM4))
RETURN
13 IF (ABS(ROOT(2,IM4)).NE.AC(1)) GO TO 15
14 AC(3)=ABS(ROOT(2,IM5))
RETURN
15 AC(3)=ABS(ROOT(2,IM4))
RETURN

* KLIP=4
AC(1)=ABS(ROOT(2,IM1))
AC(2)=ABS(ROOT(1,IC1))
AC(3)=ABS(ROOT(2,IC1))
RETURN

* KLIP=5
AC(1)=ABS(ROOT(1,IR1))
AC(3)=ABS(ROOT(2,IR2))
IF (ABS(ROOT(1,IR2)).EQ.AC(1)) GO TO 18
AC(2)=ABS(ROOT(1,IR2))
RETURN
18 AC(2)=ABS(ROOT(1,IR3))
19 KLIP=1776
RETURN

20 FORMAT (10X,10H ROOTS      )
21 FORMAT (1X,E12.5,12X,E12.5)
22 FORMAT (10X,3H|R=14,3X,3H|I=14,3X,3H|C=14/)

END
SUBROUTINE FINGLE (EN,VI)

DIMENSION FEN(3,3), VP(3)           3
DIMENSION DEN(3,3), EN(6,7), VI(6), DUDU(6), C(3,1)   4
DO 1 I=1,3                          5
DO 1 J=1,3                          6
DEN(I,J)=0.0                         7
1 CONTINUE                           8
DO 2 I=1,3                          9

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```

DEN(1,1)=EN(1+3,1)          10
DEN(1,2)=EN(1+3,3)          11
DEN(1,3)=EN(1+3,4)          12
C(1,1)=EN(1+3,7)           13
2 CONTINUE                   14
WRITE (9,9) ((DEN(I,J),J=1,3),I=1,3)    15W
CALL LEQ (DEN,C,3,1,3,3,DET)             16
V(1)=C(1,1)                         17
V(3)=C(2,1)                         18
V(4)=C(3,1)                         19
WRITE (9,10) V(1),V(3),V(4)           20W
DO 3 I=1,3                         21
DO 3 J=1,3                         22
FEN(I,J)=0.0                        23
3 CONTINUE                         24
DO 4 I=1,3                         25
FEN(1,1)=EN(1,2)                   26
FEN(1,2)=EN(1,5)                   27
FEN(1,3)=EN(1,6)                   28
4 CONTINUE                         29
C(1,1)=EN(1,7)-EN(1,1)*V(1)-EN(1,3)*V(3)-EN(1,4)*V(4) 30
C(2,1)=EN(2,7)-EN(2,1)*V(1)-EN(2,3)*V(3)-EN(2,4)*V(4) 31
C(3,1)=EN(3,7)-EN(3,1)*V(1)-EN(3,3)*V(3)-EN(3,4)*V(4) 32
WRITE (9,9) ((FEN(I,J),J=1,3),I=1,3)    33H
WRITE (9,6) C(1,1),C(2,1),C(3,1)        34W
CALL LEQ (FEN,C,3,1,3,3,DET)           35
V(2)=C(1,1)                         36
V(5)=C(2,1)                         37
V(6)=C(3,1)                         38
WRITE (9,11) V(2),V(5),V(6)           39W
DO 5 I=1,6                         40
DUDU(I)=EN(1,1)*V(1)+EN(1,2)*V(2)+EN(1,3)*V(3)+EN(1,4)*V(4)+EN(1,5)
     *V(5)+EN(1,6)*V(6)-EN(1,7)           41
5 CONTINUE                         42
WRITE (9,7) (DUDU(I),I=1,6)          43
RETURN                               44W
6 FORMAT (1X,25H FOR THE FEN MATRIX      /10X,7HC(1,1)=E12.5,5X,7HC1
12,1)=E12.5,5X,7HC(3,1)=E12.5/)   45
7 FORMAT (1X,36H SOLUTION CHECK BY SUBSTITUTION IS /16E12.5/) 46
8 FORMAT (1X,25H FEN(I,J) BY ROWS IS/(3E15.7)) 47
9 FORMAT (1X,25H DEN(I,J) BY ROWS IS/(3E15.7)) 48
10 FORMAT (1X,5HV(1)=E12.5,2X,5HV(3)=E12.5,2X,5HV(4)=E12.5) 49
11 FORMAT (1X,5HV(2)=E12.5,2X,5HV(5)=E12.5,2X,5HV(6)=E12.5) 50
END                                     51
SUBROUTINE LEQ (A,B,NEQS,NSOLNS,IA,IB,DET)          ***** 1
LINEAR EQUATIONS SOLUTIONS FORTRAN II VERSION       2
SOLVE A SYSTEM OF LINEAR EQUATIONS OF THE FORM AX=B BY A MODIFIED 3
GAUSS ELIMINATION SCHEME                         4
                                         5
NEQS = NUMBER OF EQUATIONS AND UNKNOWNs            6
NSOLNS = NUMBER OF VECTOR SOLUTIONS DESIRED        7
IA = NUMBER OF ROWS OF A AS DEFINED BY DIMENSION STATEMENT ENTRY 8
IB = NUMBER OF ROWS OF B AS DEFINED BY DIMENSION STATEMENT ENTRY 9
ADET = DETERMINANT OF A, AFTER EXIT FROM LEQ       10
                                         11
DIMENSION A(IA,IA), B(IB,IB)                      12
NSTZ=NEQS                                         13
NBSIZ=NSOLNS                                      14
NORMALIZE EACH ROW BY ITS LARGEST ELEMENT. FORM PARTIAL DETERNT 15

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```

DET=1.0          16
DO 6 I=1,NSIZ    17
BIG=A(I,I)       18
IF (NSIZ-I) LT.17,I 19
1 DO 3 J=2,NSIZ 20
  IF (ABSF(BIG)-ABSF(A(I,J))) 2,3,3 21
2 BIG=A(I,J)     22
3 CONTINUE        23
  BG=1.0/BIG      24
  DO 4 J=1,NSIZ    25
4 A(I,J)=A(I,J)*BG 26
  DO 5 J=1,NRSIZ   27
5 B(I,J)=B(I,J)*BG 28
  DET=DET*BIG      29
6 CONTINUE        30
  START SYSTEM REDUCTION 31
  NUMSYS=NSIZ-1       32
  DO 16 I=1,NUMSYS   33
    SCAN FIRST COLUMN OF CURRENT SYSTEM FOR LARGEST ELEMENT
    CALL THE ROW CONTAINING THIS ELEMENT, ROW NBGRW 34
    MN=I+1           35
    BIG=A(I,I)       36
    NBGRW=I          37
    DO 8 J=MN,NSIZ   38
      IF (ABSF(BIG)-ABSF(A(J,I))) 7,8,8 39
7 BIG=A(J,I)       40
    NBGRW=J          41
8 CONTINUE        42
  BG=1.0/BIG      43
  SWAP ROW I WITH ROW NBGRW UNLESS I=NBGRW 44
  IF (NBGRW-I) 4,12,9 45
  SWAP A-MATRIX ROWS 46
9 DO 10 J=I,NSIZ   47
  TEMP=A(NBGRW,J) 48
  A(NBGRW,J)=A(I,J) 49
10 A(I,J)=TEMP    50
  DET=-DET         51
  SWAP B-MATRIX ROWS 52
  DO 11 J=1,NBSIZ 53
  TEMP=B(I,NBGRW) 54
  B(NBGRW)=B(I,J) 55
11 A(I,J)=TEMP    56
  ELIMINATE UNKNOWN FROM FIRST COLUMN OF CURRENT SYSTEM 57
12 DO 15 K=NN,I,NSIZ 58
  COMPUTE PIVOTAL MULTIPLIER 59
  PMULT=-A(K,I)*BG 60
  APPLY PMULT TO ALL COLUMNS OF THE CURRENT A-MATRIX ROW 61
  DO 13 J=1,N,NSIZ 62
13 A(K,J)=PMULT*A(I,J)+A(K,J) 63
  APPLY PMULT TO ALL COLUMNS OF MATRIX B 64
  DO 14 L=1,NBSIZ 65
14 B(K,L)=PMULT*B(I,L)+B(K,L) 66
15 CONTINUE        67
16 CONTINUE        68
  DO BACK SUBSTITUTION 69
  WITH R-MATRIX COLUMN = NCOLB 70
17 DO 22 NCOLB=1,NBSIZ 71
  DO FOR ROW = NROW 72
  DO 21 I=1,NSIZ 73
  NROW=NSIZ+I-1 74
18 CONTINUE        75

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```

TEMP=0.0
NUMBER OF PREVIOUSLY COMPUTED UNKNOWN = NX5
NX5=NSIZ-NROW
ARE WE DOING THE BOTTOM ROW
IF (4XSI 1B,20,18
NO
18 DO 19 K=1,NXS
   KK=NSIZ+1-K
19 TEMP=TEMP+AIKK,NCOLAI*(NROW,KK)
20 B(NROW,NCOLB)=(B(NROW,NCOLB)-TEMP)/AINROW,NROW)
   HAVE WE FINISHED ALL ROWS FOR B-MATRIX COLUMN = NCOLB
21 CONTINUE
YES
HAVE WE JUST FINISHED WITH B-MATRIX COLUMN NCOLB=NSIZ
22 CONTINUE
YES
NOW FINISH COMPUTING THE DETERMINANT
DO 23 I=1,NSIZ
23 DET=DET*AI(I,I)
WE ARE ALL DONE NOW
WHEN...
RETURN
END
SUBROUTINE PRINT (X)
COMMON DLTEMP
COMMON TEMP,EA,EAC,YUA,YUAC,EB,EBC,NUB,NUBC,HA,HB,R,G3A,G3B,CA,CP,PRINT 3
IDA,DB,H,CACH,CAPCB,YUNU,YUMA,GAMB,K(40),A(55),G(7),RT(30),FTDIA,FNPINT 4
ZTOB,BDE(6001),LUD1,AM(50),LOAD2,EN(6,7),FNXA,FNXR,FMXA,FMXB,FNK,FMPINT 5
3X,FNDA,FNDB,AC(3),KLIP,ELL,DELPH(201),V(6),AL(18),WW,UW,DW,H,W,UNPRINT 6
4B,DDWB,DDDWB,WA,DWA,MU,DMU,DDDWU,TAU,DTAU,WPJ,WWA,DMWA,MU4,DMWPRINT 7
PRINT 8
5UA
COMMON /SHEAR/ QA,QB,FQ
REAL NUAA,YUAC,YUAB,YUAC,NUUU,K,LOAD1,LOAD2
WRITE (9,4) X,HA,HB,ELL,R,TEMP
WRITE (9,2) KLIP
WRITE (9,5) WW,WWA
WRITE (9,6) WU,WA
WRITE (9,7) WU,WUA
WRITE (9,8) TAU,WPJ
WRITE (9,9) FNXA,FMXB,FMXA,FMXB
WRITE (9,10) FNX,FME
WRITE (9,3) FNDA,FNDB
WRITE (9,1) QA,QR,FQ
RETURN

1 FORMAT (1X,3HGA=E12.5,2X,3HGB=E12.5,2X,5HQRAR=E12.5/)
2 FORMAT (1X,2H
   2H      KLIP=13)
3 FORMAT (2X,5HFNDA=E12.5,2X,5HFNDB=E12.5)
4 FORMAT (2X,2HX=F9.2,3HHAA=F9.2,2X,3HHB=F9.2,2X,4HELL=F7.1,2X,2HR=F7PRINT2d
   1,1,2X,5HTEMP=E5.1)
5 FORMAT (2X,5HW(1)=E12.5,2X,5HW(1)=E12.5/1)
6 FORMAT (2X,BHRETAIR1=E12.5,2X,BHBTET(1)=E12.5/1)
7 FORMAT (2X,SHU(B1)=E12.5,2X,SHU(A1)=E12.5/1)
8 FORMAT (2X,4HTAU=E12.5,2X,5HPL(J1)=E12.5/1)
9 FORMAT (1X,6HNX(A1)=E12.5,1X,6HNX(R1)=E12.5,1X,6HNX(A1)=E12.5,1X,6HNXPPRINT34
   1B1)=E12.5/1)
10 FORMAT (2X,5HNRAR=E12.5,2X,5HNRAR=E12.5)
END

```

```

SUBROUTINE RSLT (X)          ***** 1
COMMON DLTEMP
COMMON TEMP,EA,EAC,NUA,NUAC,EB,EBC,NUB,NURC,HA,HR,R,G3A,G3B,CA,CB,RSLT 2
1DA,DB,H,CACH,CAPCR,NUNU,GAMA,GAMR,K(40),A(55),G(7),BT(30),F1TOA,FNRSLT 3
2TOR,BDE(600),LOAD1,AM15J1,LOAD2,EN(6,7),FNXA,FNXR,FMXA,FMXU,FYX,FHRSLT 4
3X,FNGA,FNGD,AC(3),KLIP,ELL,JELPH(20),V(6),AL(18),WW,DWW,DDWW,WR,DWRSLT 5
4B,DDWB,DDWH,WA,DWA,WU,DWU,DDWU,TAU,DEAU,WPJ,WA,DWA,WA,LA,RSLT 6
5UA                                              RSLT 7
COMMON /SHEAR/ QA,QR,FO
REAL NUA,NUAC,NUB,NURC,NUNU,K,LOAD1,LOAD2
WW=WW+RT(9)
DWAA=DWW
WUA=WU+(HA/2.)*WA+(HB/2.)*WB
DWUA=DWU+(HA/2.)*DWA+(HB/2.)*DWB
FNXA=CA*DWA*(CA*WUA/R)+(WW)-BT(5)/(L.-NUA)+BT(21)
FNXD=CB*DWA*(CB*NUR/R)+(WW)-BT(6)/(L.-NUB)+BT(22)
FNX=FNXA+FNXR
FMXA=DA*DWA-BT(12)/(L.-NUA)+BT(1H)
FMXR=DR*DWB-BT(13)/(L.-NUB)+BT(19)
FMX=FMXA+FMXR+(HA/2.)*WB/2.)*FNXA
QA=(HA/12.)*TAU+GAMA*(WA*DWW)
QB=(HB/12.)*TAU+GAMR*(WB*DWW)
FQ=QA+QB
RETURN
END
SUBROUTINE BINGO (EN,V)
DIMENSION DUMMY(6)
DIMENSION EN(6,7), REN(6,6), C(6,1), V(6)
DO 1 I=1,6
DO 1 J=1,6
REN(I,J)=EN(I,J)
C(I,1)=EN(I,7)
1 CONTINUE
CALL LEQ (REN,C,6,1,6,6,DET)
DO 2 I=1,6
V(I)=C(I,1)
2 CONTINUE
DO 3 I=1,6
DUMMY(I)=EN(I,1)*C(I,1)+EN(I,2)*C(2,1)+EN(I,3)*C(3,1)+EN(I,4)*C(4,1)
I1)+EN(I,5)*C(5,1)+EN(I,6)*C(6,1)-EN(I,7)
3 CONTINUE
WRITE (9,4)
WRITE (9,5) (DUMMY(I),I=1,6)
RETURN
4 FORMAT (40H SOLUTION CHECK BY SUBSTITUTION      )
5 FORMAT (1X,9HSHARI(0)=E12.5,1X,9HMHAR(0)=E12.5,1X,9HQHAR(0)=E12.5/,9HGO 21
1IX,9HSHARI(4)=E12.5,1X,9HMHAR(4)=E12.5,1X,9HQHAR(4)=E12.5//) 22
END

```

APPENDIX B
Data

TABLE B.1
AVERAGE MATERIAL PROPERTIES FOR PG AND ATJ GRAPHITE

	<u>PG</u>	<u>ATJ</u>
E	3.1×10^6 psi	2.26×10^6 psi
v	-0.21	+ 0.30
v_c	+ 0.90	+ 0.25
R	varies with case	30 in.
L	40 in.	40 in.
α	1.43×10^{-6} <u>in</u> <u>in</u> - ${}^{\circ}\text{F}$	4.25×10^{-6} <u>in</u> <u>in</u> - ${}^{\circ}\text{F}$
α_c	13.1×10^{-6} <u>in</u> <u>in</u> - ${}^{\circ}\text{F}$	4.25×10^{-6} <u>in</u> <u>in</u> - ${}^{\circ}\text{F}$

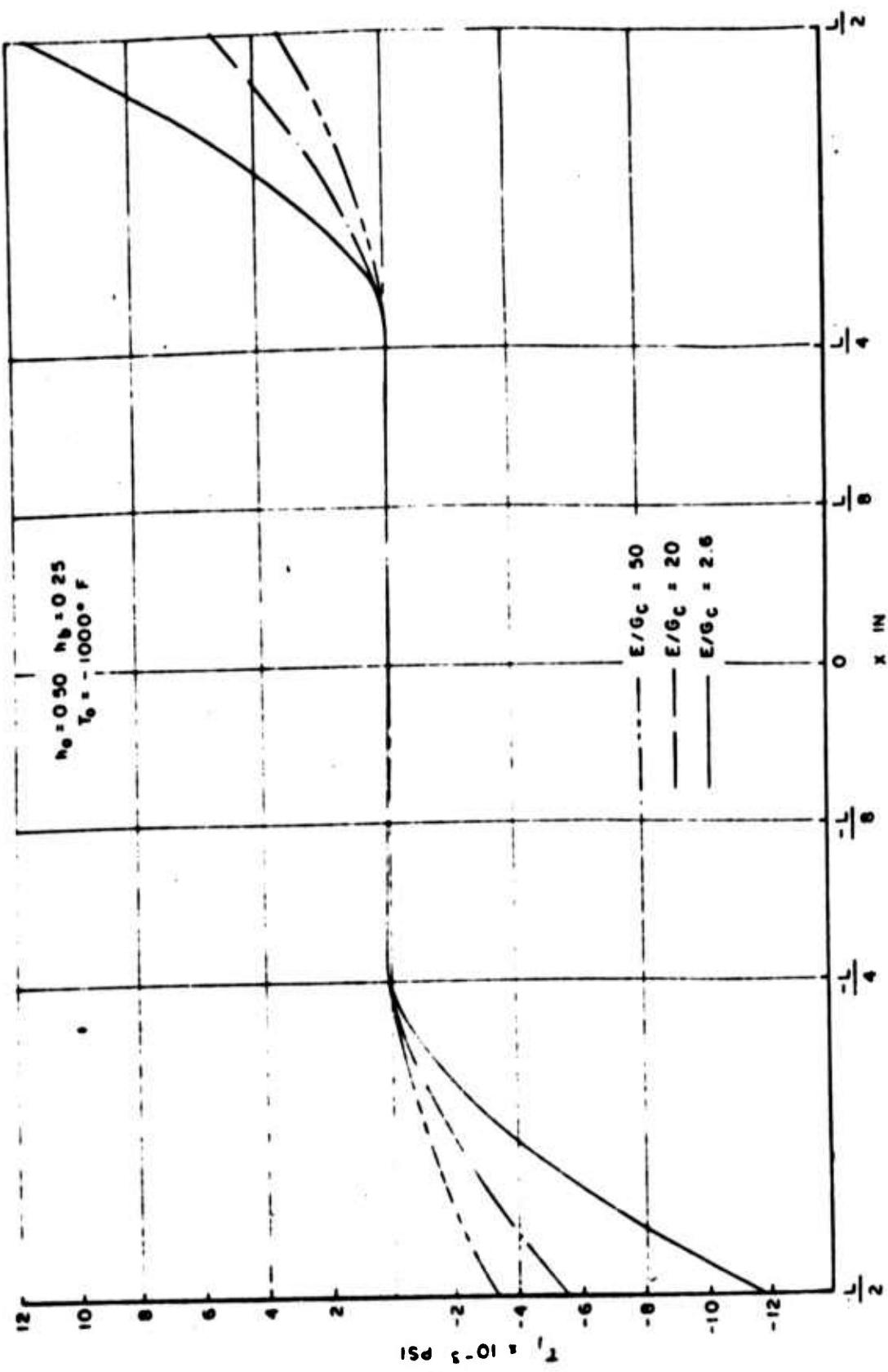


Figure B.1 Effect of E/G_c on Joint Shear Stress.

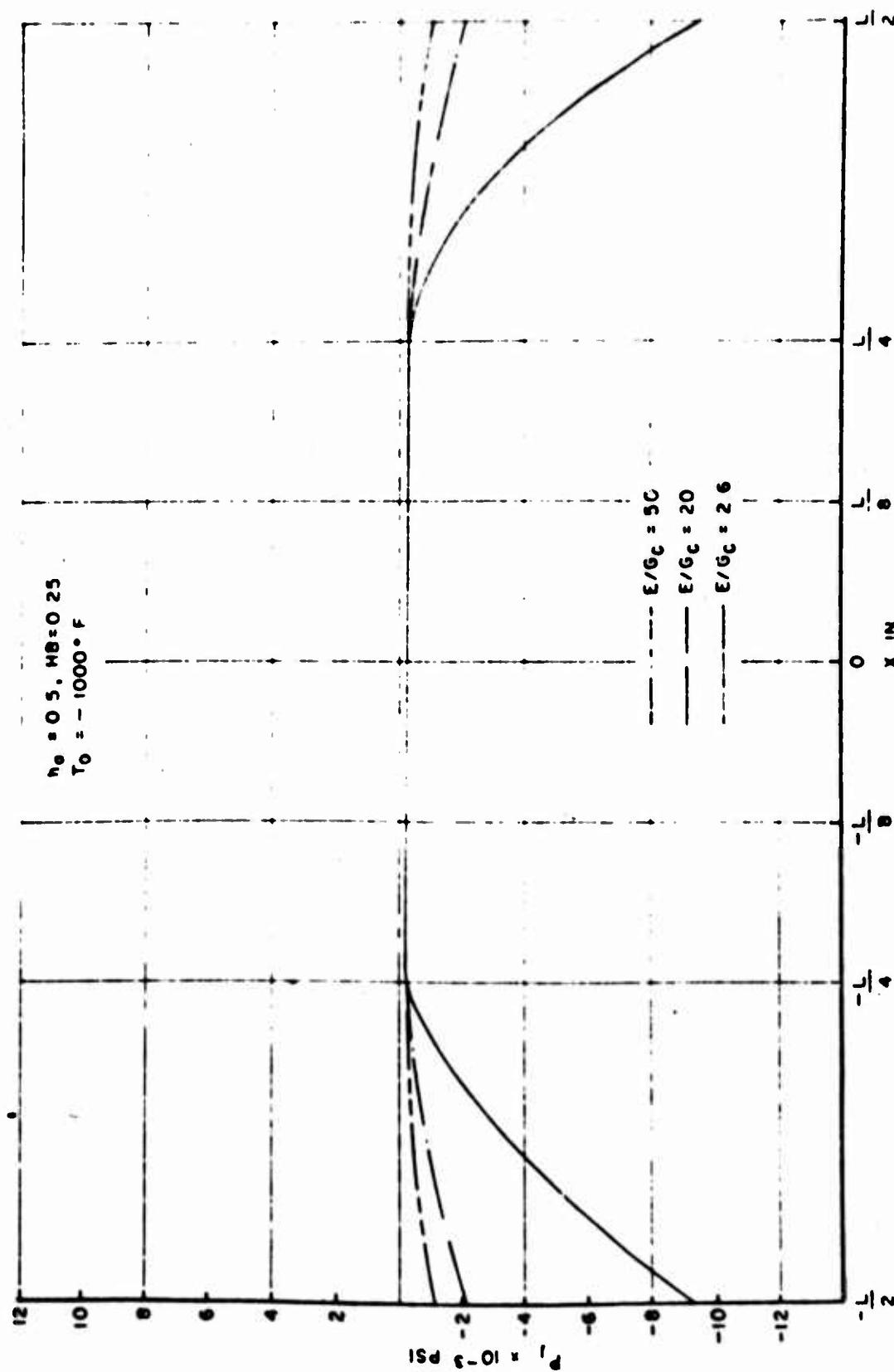


Figure B-2 Effect of E/G_c on Joint Normal Stress

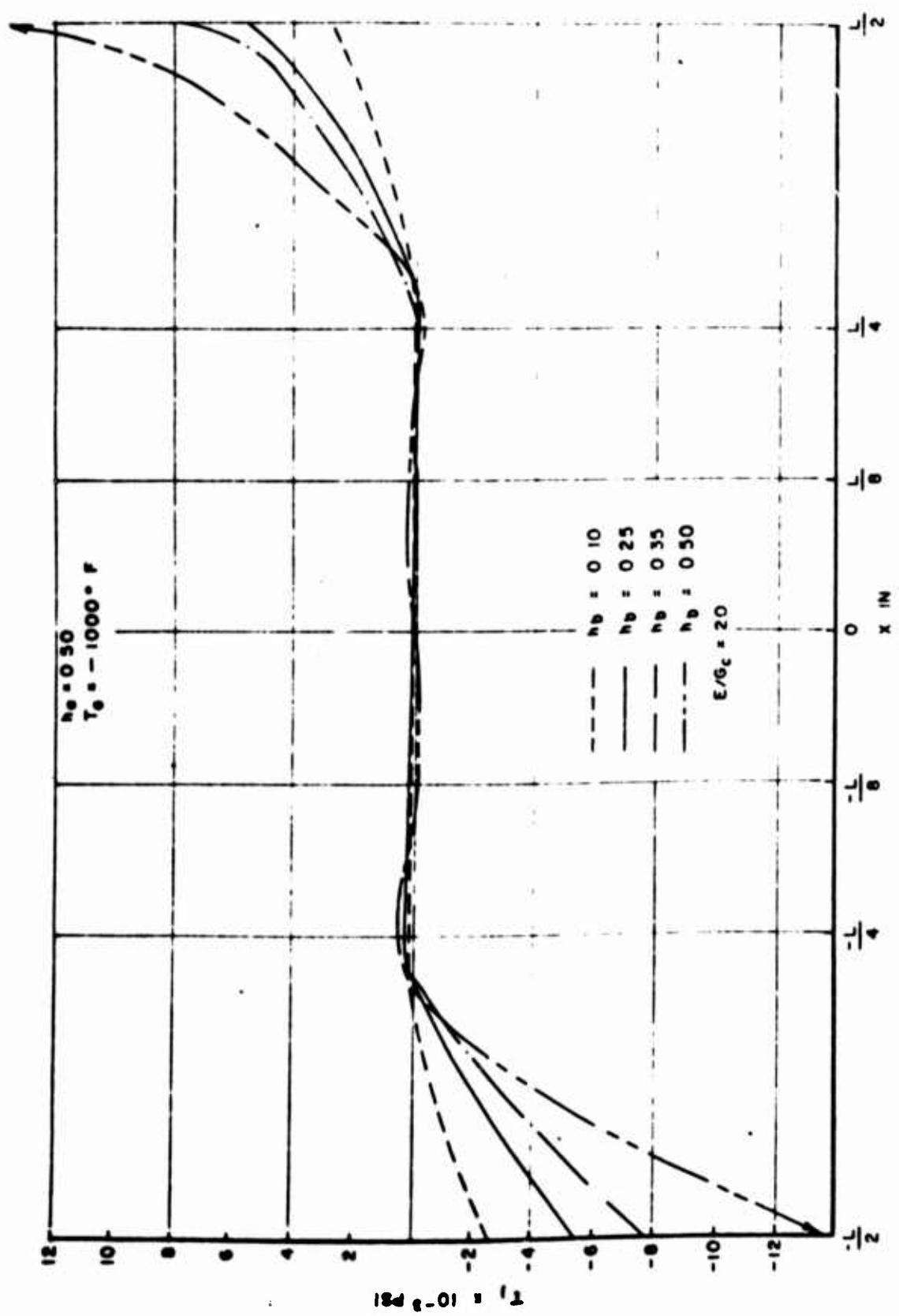


Figure B.3 Behavior of Joint Shear Stress With Varying Mandrel Thickness

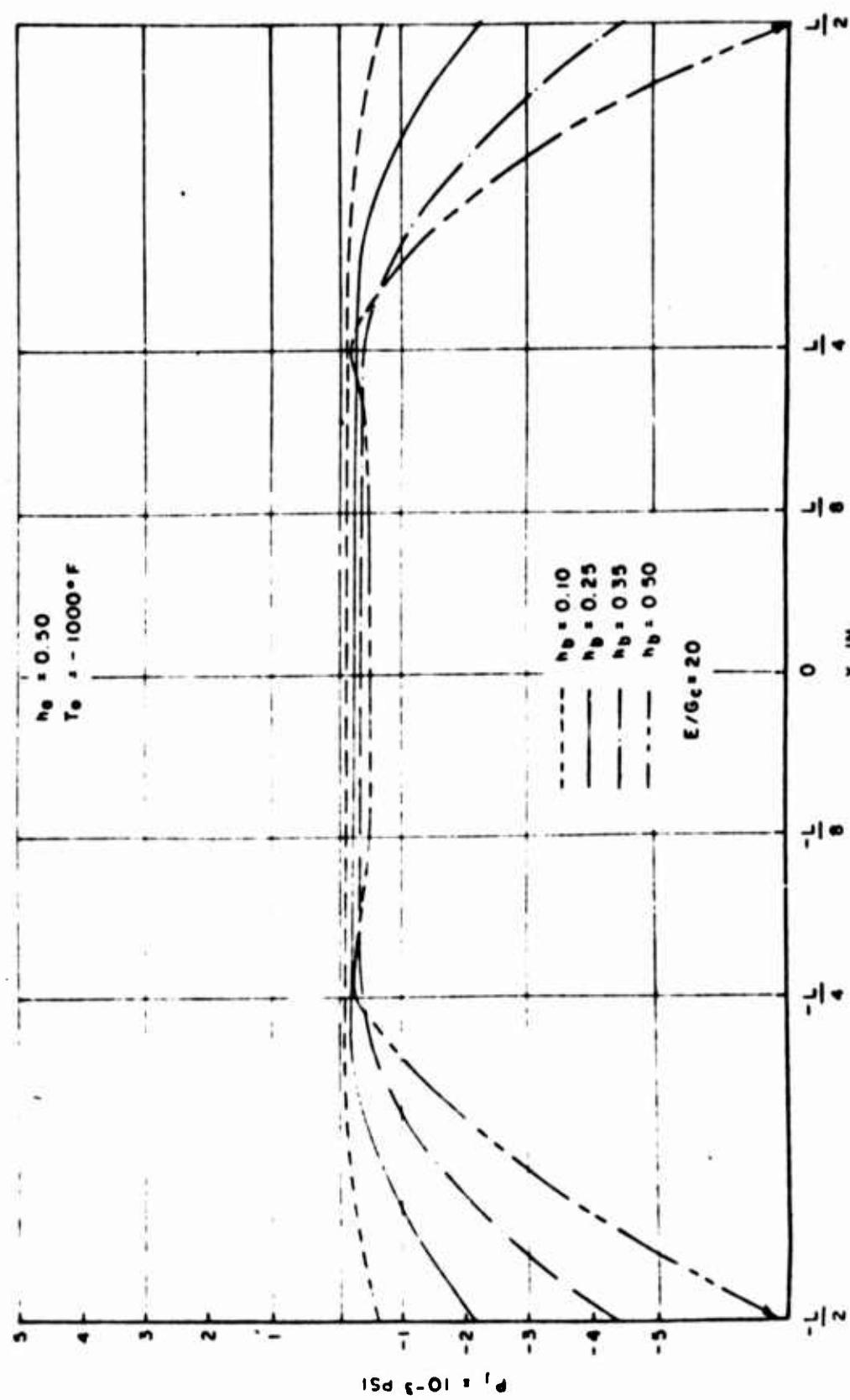


Figure B 4 Behavior of Joint Normal Stresses With Varying Mandrel Thickness

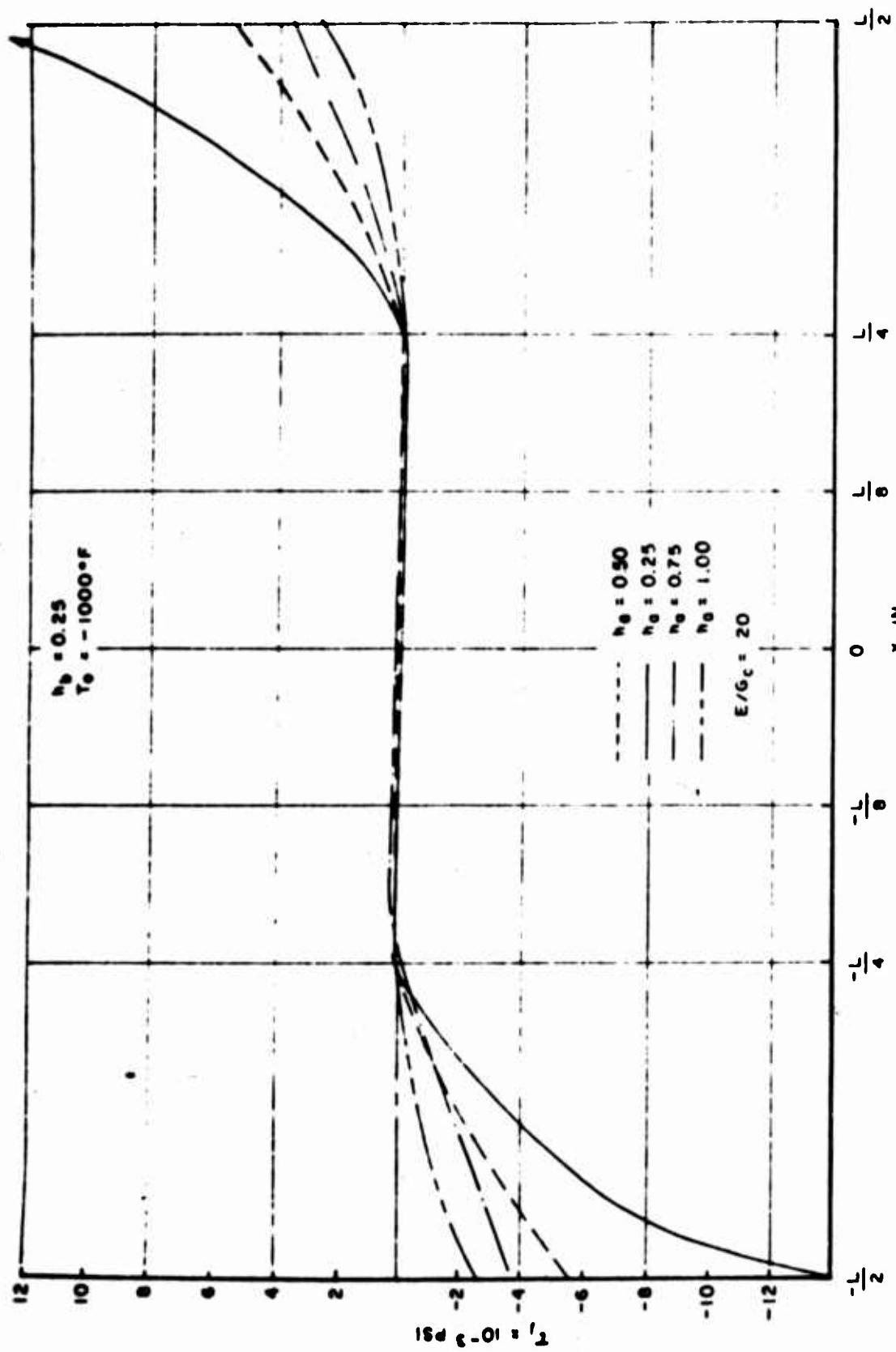


Figure 8.5 Variation of Joint Shear Stresses With Increasing PG Deposition Thickness

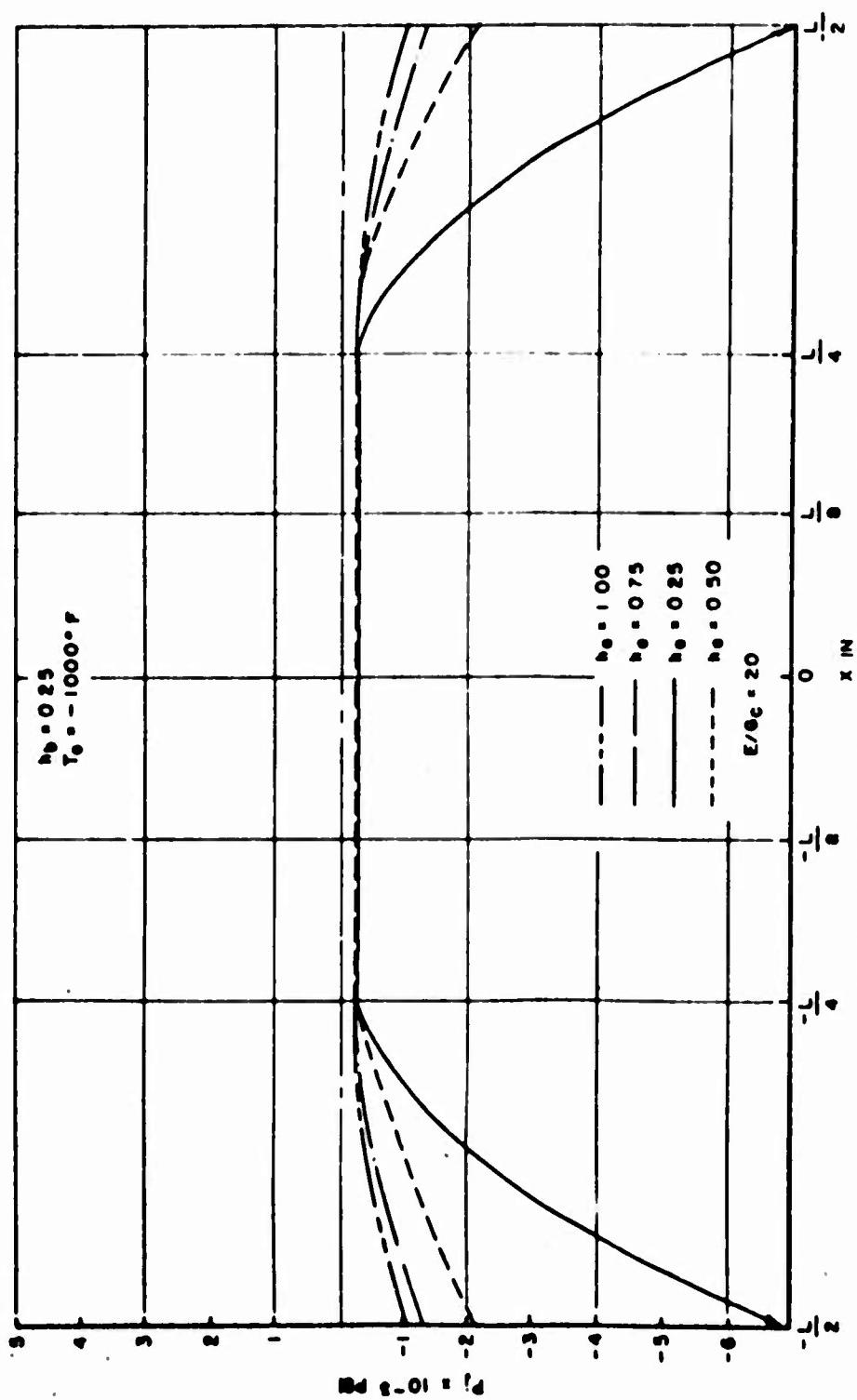


Figure B 6 Variation of Joint Normal Stresses With Increasing PG Deposition Thickness

APPENDIX C
DEFINITION OF TERMS

$$z_1 = (c_a + c_b) D \{(\alpha_{17} + \alpha_{27} + \pi_1 + \pi_2) - \frac{h}{2} D (\alpha_{47} + \pi_4)\}$$

$$+ \{ \frac{h}{2} c_b D^2 + (\frac{c_a v_a + c_b v_b}{R_a R_b}) \} (\alpha_{37} + \alpha_{47} + \pi_3 + \pi_4)$$

$$z_2 = (c_a + c_b) \{ \alpha_{67} + \pi_6 - \frac{5}{12} h_b (\alpha_{47} + \pi_4) \}$$

$$+ \frac{5}{12} h_b c_b (\alpha_{37} + \alpha_{47} + \pi_3 + \pi_4)$$

$$z_3 = (c_a + c_b) \{ \alpha_{57} + \pi_5 - \frac{5}{12} h_a (\alpha_{47} + \pi_4) \}$$

$$+ \frac{5}{12} h_a c_b (\alpha_{37} + \alpha_{47} + \pi_3 + \pi_4)$$

$$z_4 = a_{31} z_1 - (a_{11} D^2 + a_{15}) z_2$$

$$z_5 = a_{23} D^2 z_3 - (a_{31} D^2 + a_{32}) z_2 \quad (C.1)$$

$$a_{11} = D_a (c_a + c_b) + \frac{c_a c_b h_a h}{4}$$

$$a_{12} = - \frac{c_a c_b h_a}{2} \left(\frac{v_a}{R_a} - \frac{v_b}{R_b} \right)$$

$$a_{13} = D_b (c_a + c_b) + \frac{c_a c_b h_b h}{4}$$

$$a_{14} = \frac{h_b}{h_a} a_{12}$$

$$a_{15} = \frac{c_a c_b h}{2} \left(\frac{v_a}{R_a} - \frac{v_b}{R_b} \right)$$

$$a_{16} = \left(\frac{c_a v_a + c_b v_b}{R_a R_b} \right)^2 - \left(\frac{c_a + c_b}{R_a R_b} \right)^2$$

$$a_{21} = D_b (c_a + c_b) + \frac{5}{24} c_a c_b h_b^2$$

$$a_{22} = - (c_a + c_b) \frac{5}{6} G_c^b h_b$$

$$a_{23} = \frac{5}{24} c_a c_b h_a h_b$$

$$a_{24} = \frac{5}{12} c_a c_b h_b \left(\frac{v_a}{R_a} - \frac{v_b}{R_b} \right) + (c_a + c_b) \frac{5}{6} G_c^b h_b$$

$$a_{31} = D_a (c_a + c_b) + \frac{5}{24} c_a c_b h_a^2$$

$$a_{32} = - (c_a + c_b) \frac{5}{6} G_c^a h_a$$

$$a_{33} = a_{23}$$

$$a_{34} = \frac{5}{12} c_a c_b h_a \left(\frac{v_a}{R_a} - \frac{v_b}{R_b} \right) - \frac{5}{6} G_c^a h_a (c_a + c_b) \quad (c.2)$$

$$b_{11} = a_{13}a_{23} - a_{11}a_{21}$$

$$b_{12} = a_{14}a_{23} - a_{11}a_{22} - a_{21}a_{12}$$

$$b_{13} = - a_{12}a_{22}$$

$$b_{14} = a_{15}a_{23} - a_{11}a_{24}$$

$$b_{15} = a_{16}a_{23} - a_{12}a_{24}$$

$$b_{21} = a_{23}a_{23} - a_{31}a_{21}$$

$$b_{22} = - (a_{31}a_{22} + a_{32}a_{21})$$

$$b_{23} = - a_{32}a_{22}$$

$$b_{24} = a_{23}a_{34} - a_{31}a_{24}$$

$$b_{25} = - a_{32}a_{24}$$

(c.3)

$$g_1 = b_{21} b_{14} - b_{11} b_{24}$$

$$g_2 = b_{21} b_{15} + b_{22} b_{14} - b_{12} b_{24} - b_{11} b_{25}$$

$$g_3 = b_{22} b_{15} + b_{23} b_{14} - b_{12} b_{25} - b_{13} b_{24}$$

$$g_4 = b_{23} b_{16} - b_{13} b_{25} \quad (C.4)$$

$$L_I(x) = (b_{21} D^4 + b_{22} D^2 + b_{23}) z_4 - (b_{11} D^4 + b_{12} D^2 + b_{13}) z_5$$

$$L_{II}(x) = z_4$$

$$L_{III}(x) = z_2$$

$$L_{IV}(x) = 1/(c_a + c_b) (\alpha_{37} + \alpha_{47} + \pi_3 + \pi_4)$$

$$L_V(x) = \alpha_{47} + \pi_4$$

$$L_{VI}(x) = \alpha_{27} + \pi_2 \quad (C.5)$$

$$k_1 = c_a h_a / 2(c_a + c_b)$$

$$k_2 = C_a h_b / 2(C_a + C_b)$$

$$k_3 = 1/R (C_a v_a + C_b v_b) / (C_a + C_b)$$

$$k_4 = C_b$$

$$k_5 = k_8 = C_b v_b / R_b$$

$$k_6 = C_b h_b^2 / 12$$

$$k_7 = h_b / 2$$

$$k_9 = C_b / R_b^2 \quad (C.6)$$

$$m = 1/3 \left\{ 3 \left(\frac{g_3}{g_1} \right) - \left(\frac{g_2}{g_1} \right)^2 \right\}$$

$$n = 1/27 \left\{ 2 \left(\frac{g_2}{g_1} \right)^3 - 9 \left(\frac{g_2}{g_1} \right) \left(\frac{g_3}{g_1} \right) + 27 \left(\frac{g_4}{g_1} \right) \right\} \quad (C.7)$$

$$A = \left\{ -n/2 + n^2/4 + m^3/27 \right\}^{1/3}$$

$$B = \left\{ -n/2 - n^2/4 + m^3/27 \right\}^{1/3} \quad (C.8)$$

$$\lambda_1 = A + B - 1/3 (g_2/g_1)$$

$$\lambda_2 = +1/2 (A + B) + 1/3 (g_2/g_1) \quad (C.9)$$

$$\lambda_3 = \frac{\sqrt{3}}{2} (A - B)$$

$$r = (\lambda_2)^2 + (\lambda_3)^2$$

$$\tan \theta_1 = -\lambda_3 / \lambda_2$$

$$\tan \theta_2 = \lambda_3 / \lambda_2$$

(C.10)

$$c_1 = (\lambda_1)^{1/2}$$

$$c_2 = r^{1/4} \cos(\theta_1/2)$$

(C.11)

$$c_3 = r^{1/4} \sin(\theta_1/2)$$

$$q = \left| \frac{n^2}{4} + \frac{m^3}{27} \right|^{1/2} \quad (C.12)$$

$$c_4 = (-n)^{1/3}$$

$$c_5 = -1/2 \left\{ c_4 + 3(2q)^{1/3} \right\} \quad (C.13)$$

$$c_6 = -1/2 \left\{ c_4 - 3(2q)^{1/3} \right\}$$

APPENDIX D
Derivation of Integrated Shear Stress Resultant

The required shear stress-strain relation can be developed by weighted integration in order to obtain the factor 5/6 that is generally accepted for isotropic plates and shells. The method follows that in reference (15). For convenience, the procedure for cylindrical shells is reproduced here.

Expressions for normal stress distributions with z can be obtained by replacing the strains in the stress-strain relations (1) by the approximate forms (8) and neglecting (z/R) in comparison to unity in the same terms. It is to be understood that the following hold for each lamina.

$$\sigma_x = \frac{E}{1-v^2} \left(u'_0 + \frac{vw}{R} + z\beta' \right) - \frac{E\alpha T}{1-v} + \frac{vEw}{R(1-v^2)} \quad (D.1)$$

$$\sigma_\theta = \frac{E}{1-v^2} \left(vu'_0 + \frac{w}{R} + vzb' \right) - \frac{E\alpha T}{1-v} + \frac{Ew}{R(1-v^2)} \quad (D.2)$$

Expressions for the stress distributions in terms of stress resultants and couples can be obtained by using equations (11) in (D.1) and (D.2).

$$\begin{aligned} \sigma_x &= \frac{N_x}{h} + \frac{N_{Tx}}{(1-v)h} - \frac{vE}{Rh(1-v^2)} \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{w} dz + \frac{12}{h^3} z \left[M_x + \frac{M_{Tx}}{1-v} - \frac{E}{R(1-v^2)} \int_{-\frac{h}{2}}^{\frac{h}{2}} \right. \\ &\quad \left. \bar{w} zdz \right] - \frac{E\alpha T}{1-v} + \frac{vEw}{R(1-v^2)} \end{aligned} \quad (D.3)$$

$$\sigma_0 = \frac{N_0}{h} + \frac{N_{T0}}{(1-v)h} - \frac{F}{Rh(1-v^2)} \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{w} dz + \frac{12}{h^3} z \left[M_0 + \frac{M_{T0}}{1-v} \right] - \frac{E}{Rh(1-v^2)} \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{w} z dz - \frac{E\alpha T}{1-v} + \frac{E}{R(1-v^2)} \bar{w} \quad (D.4)$$

The shear stress is related to the normal stress by the equilibrium equations (2). If the first of these is integrated with respect to z , the following is obtained.

$$\sigma_{xz} - \tau_{2i} = \frac{1}{R} \int_{-\frac{h}{2}}^z \frac{\partial}{\partial x} (R\sigma_x) dz \quad (D.5)$$

Using (D.3) in (D.5)

$$\sigma_{xz} - \tau_{2i} = - \frac{1}{Rh} \int_{-\frac{h}{2}}^z \left\{ RN'_x + \frac{12}{h^2} z RM'_x \right\} dz + \frac{1}{R} \int_{-\frac{h}{2}}^z \Omega dz \quad (D.6)$$

where

$$\Omega = \frac{R}{1-v} \frac{\partial}{\partial x} \left\{ \frac{N_{Tx}}{h} + \frac{12z}{h^3} M_{Tx} - E\alpha T - \frac{vE}{(1+v)R} \left[\frac{1}{h} \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{w} dz + \frac{12z}{h^3} \int_{-\frac{h}{2}}^{\frac{h}{2}} \bar{w} dz - \bar{w} \right] \right\}$$

Referring to the integrated equilibrium equations (10) and replacing the normal stress resultants and couples in (D.5) by their

equivalents in terms of the shear stress resultant.

$$\sigma_{xz} - \tau_{2i} = \frac{1}{h} \int_{-\frac{h}{2}}^z \left[-\frac{12z}{h^2} Q + (1 + \frac{6z}{h}) \tau_{1i} - (1 - \frac{6z}{h}) \tau_{2i} \right] dz + \frac{1}{R} \int_{-\frac{h}{2}}^{\frac{z}{2}} Q dz \quad (D.8)$$

Performing the indicated integration

$$\begin{aligned} \sigma_{xz} - \tau_{2i} = & \frac{3}{2} \left[1 - \left(\frac{2z}{h} \right)^2 \right] \frac{Q}{h} - \frac{\tau_{1i}}{4} \left[1 - \frac{4z}{h} - 3 \left(\frac{2z}{h} \right)^2 \right] - \frac{\tau_{2i}}{4} \left[1 + \frac{4z}{h} \right. \\ & \left. - 3 \left(\frac{2z}{h} \right)^2 \right] + \frac{1}{R} \int_{-\frac{h}{2}}^z Q dz \end{aligned} \quad (D.9)$$

From (D.6) and equations (9), it can readily be shown that

$$\int_{-\frac{h}{2}}^{\frac{h}{2}} Q dz = 0 \quad (D.10)$$

$$\int_{-\frac{h}{2}}^{\frac{h}{2}} Q z dz = 0$$

The shear stress distribution (D.9) satisfies shear stress boundary conditions and the definition of the shear stress resultant. To prove the latter it is necessary to make use of (D.10) and

$$\int_{-\frac{h}{2}}^{\frac{h}{2}} \int_{-\frac{h}{2}}^z f(y) dy dz = \int_{-\frac{h}{2}}^{\frac{h}{2}} \left(\frac{h}{2} - z\right) f(z) dz \quad (D.11)$$

McDonough (15) points out that the effect of the last term in (D.9) is to modify the classical quadratic shear stress distribution but not the magnitude of the shear stress resultant. The modification is due to the nonlinearity of the normal stress distribution, primarily its distribution with x .

Proceeding with the weighted integration as in (15), the shear stress-strain relation of the set (1) is multiplied by the weighting function $\left[1 - \left(\frac{2z}{h}\right)^2\right]$ and then integrated through the thickness of the shell to yield:

$$\int_{-\frac{h}{2}}^{\frac{h}{2}} \left[1 - \left(\frac{2z}{h}\right)^2\right] \sigma_{xz} dz = 2G_c \int_{-\frac{h}{2}}^{\frac{h}{2}} \left[1 - \left(\frac{2z}{h}\right)^2\right] G_{xz} dz \quad (D.12)$$

The integral on the left hand side is evaluated using (D.9) with the aid of (D.10) and (D.11). The integral on the right hand side is evaluated by using the shear strain given in equation (8). After integration, rearrangement and simplification, equation (11A) results:

$$Q = \frac{\bar{m}}{6} + \frac{5}{6} G_c h (\beta_i + w_i') + \frac{5}{4} \int_{-\frac{h}{2}}^{\frac{h}{2}} \left[1 - \left(\frac{2z}{h} \right)^2 \right] G_c \bar{w}' dz + \frac{5}{4R} \int_{-\frac{h}{2}}^{\frac{h}{2}}$$

$$\left(\frac{2z}{h} \right)^2 \int_{-\frac{h}{2}}^z \Omega dy dz \quad (D.13)$$

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13. ABSTRACT

A theory for the analysis of stresses in laminated circular cylindrical shells subjected to arbitrary axisymmetric mechanical and thermal loadings has been developed. This theory, specifically for use with pyrolytic graphite type materials, differs from the classical thin shell theory in that it includes the effects of transverse shear deformation and transverse isotropy, as well as thermal expansion through the shell thickness.

Solutions in several forms are developed for the governing equations. The form taken by the solution function is governed by geometric considerations. A range in which the various solution forms occur was determined numerically.

As a sample problem, the slow cooling of pyrolytic graphite deposited onto a commercial graphite mandrel was considered. Investigation of normal and shear stress behavior at the pyrolytic graphite - mandrel interface showed that these stresses decrease in magnitude with increasing E/G ratio and increasing deposit to mandrel thickness (h_a/h_b) ratio. This implies that a thin mandrel and a material weak in shear are desirable to minimize the possibilities of flaking and delamination of the pyrolytic graphite.

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